



Marshall Space Flight Center  
GFSSP Training Course



# GFSSP Training Course Lectures

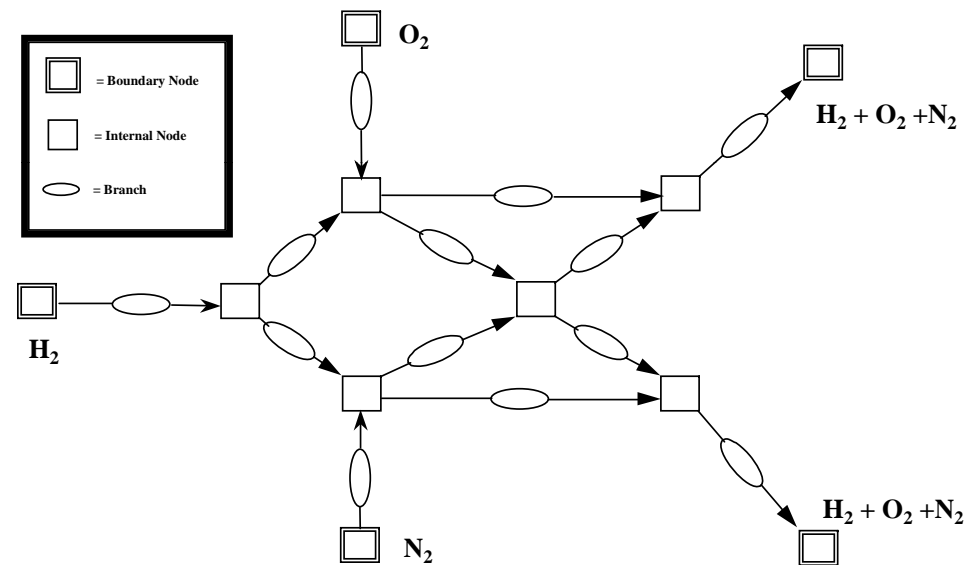
**Alok Majumdar**  
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**Thermal & Fluids Analysis Workshop**  
NASA/Ames Research Center & San Jose University  
August 18-22, 2008



# INTRODUCTION & OVERVIEW





# CONTENT

- Introduction
  - Background
  - Course Outline
- Overview
  - Network Flow or Navier Stokes Analysis
  - Network Definition
  - Data Structure
  - Mathematical Formulation
  - Program Structure
  - Graphical User Interface
  - Resistance & Fluid Options
  - Advanced Options
  - Applications



## BACKGROUND -1

- GFSSP stands for Generalized Fluid System Simulation Program
- It is a general-purpose computer program to compute pressure, temperature and flow distribution in flow network
- It was primarily developed to analyze
  - Internal Flow Analysis of Turbopump
  - Transient Flow Analysis of Propulsion System
- GFSSP development started in 1994 with an objective to provide a generalized and easy to use flow analysis tool



## BACKGROUND -2

### DEVELOPMENT HISTORY

- Version 1.4 (Steady State) was released in 1996
- Version 2.01 (Thermodynamic Transient) was released in 1998
- Version 3.0 (User Subroutine) was released in 1999
- Graphical User Interface, VTASC was developed in 2000
- Selected for NASA Software of the Year Award in 2001
- Version 4.0 (Fluid Transient and post-processing capability) is released in 2003

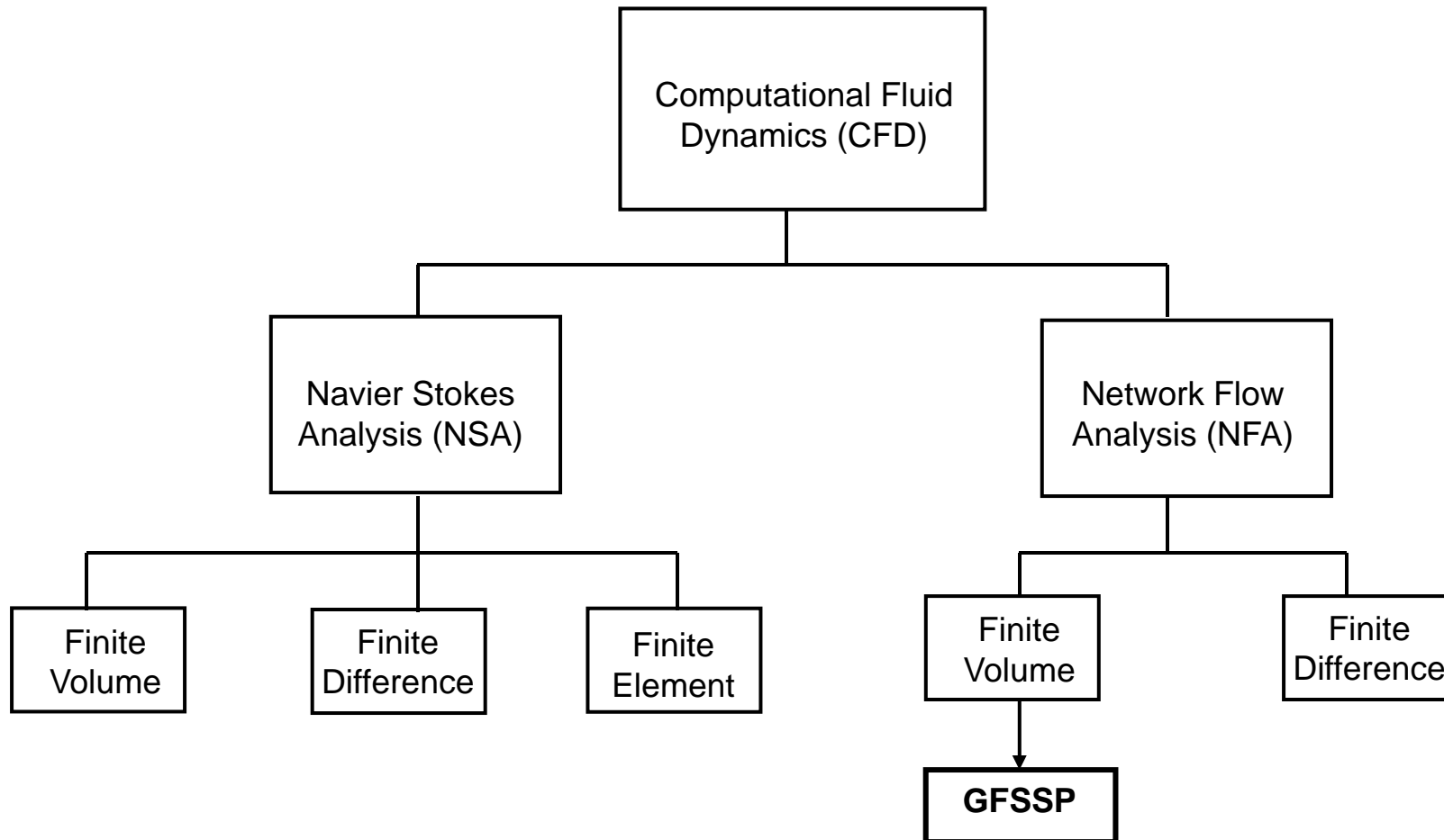


## COURSE OUTLINE

1. Introduction & Overview
2. Graphical User Interface
3. Mathematical Formulation
4. User Subroutine
5. Pressurization, Waterhammer & Conjugate Heat Transfer
6. Tutorials (Afternoon)



# NETWORK FLOW OR NAVIER STOKES ANALYSIS - 1





## NETWORK FLOW OR NAVIER STOKES ANALYSIS - 2

### Navier Stokes Analysis

- Suitable for detailed flow analysis within a component
- Requires fine grid resolution to accurately model transport processes
- Used after after preliminary design

### Network Flow Analysis

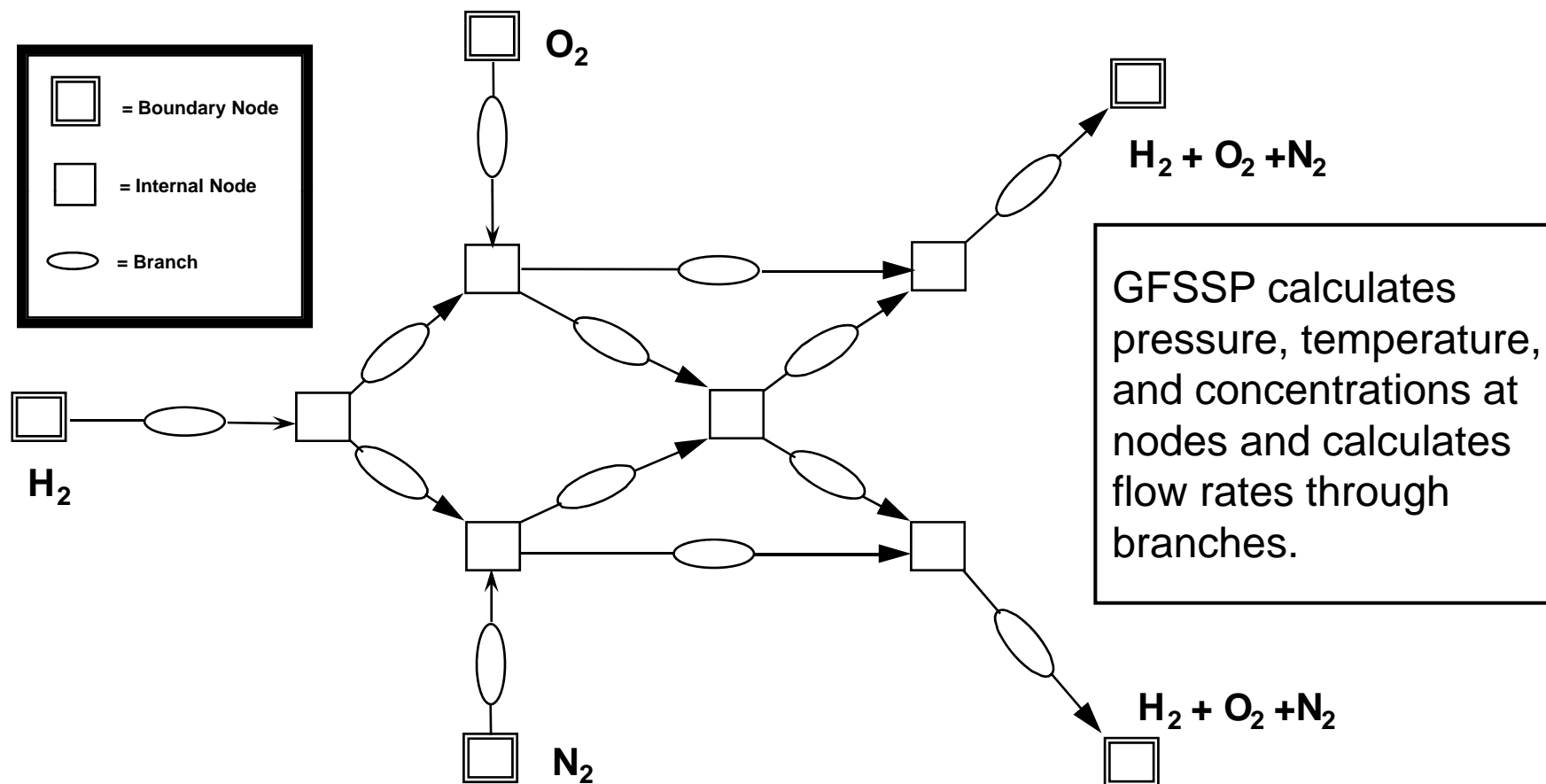
- Suitable for flow analysis of a system consisting of several components
- Uses empirical laws of transport process
- Used during preliminary design





# NETWORK DEFINITION – 1

## GFSSP FLOW CIRCUIT



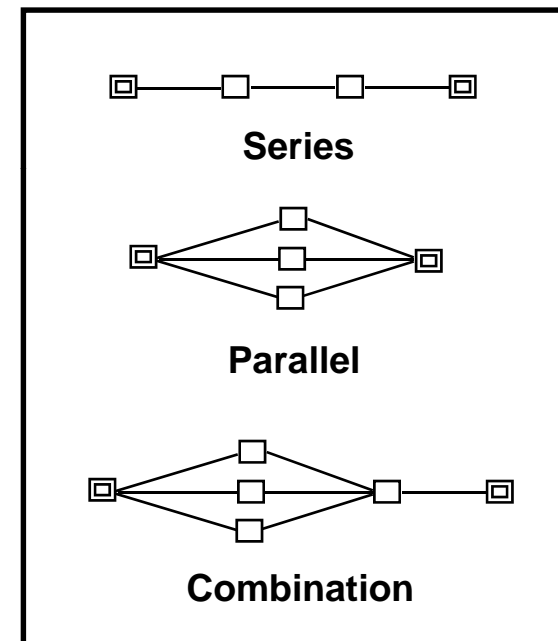


## NETWORK DEFINITIONS - 2

- **Network:**

- Boundary node
- Internal node
- Branch

- **At boundary nodes, all dependent variables must be specified**
- **At internal nodes, all dependent variables must be guessed for steady flow and specified for transient flow.**





## NETWORK DEFINITIONS - 3

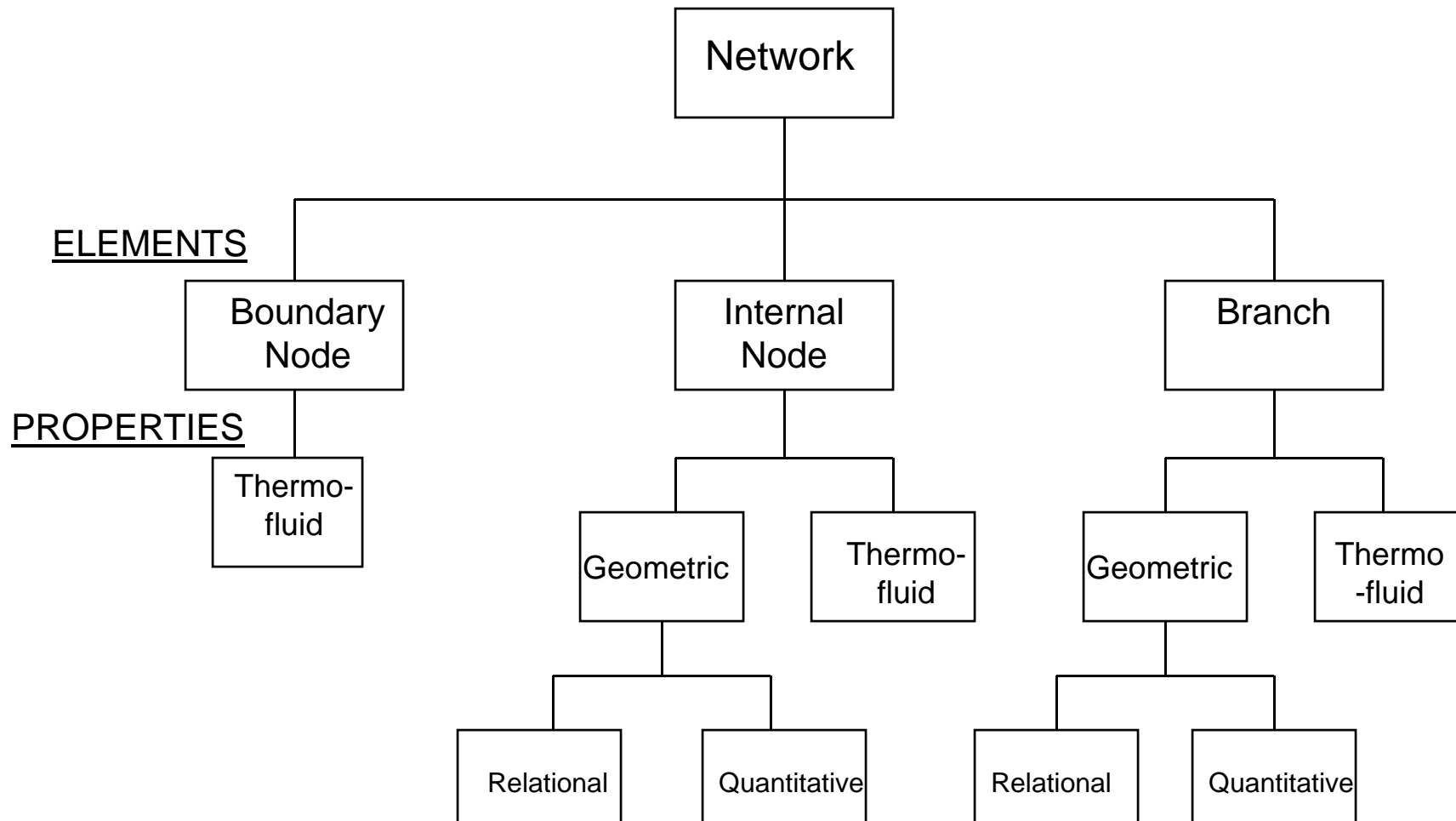
### UNITS AND SIGN CONVENTIONS

- **Units**

	<b>External</b> (input/output)	<b>Internal</b> (inside GFSSP)
– Length	- inches	- feet
– Area	- inches <sup>2</sup>	- feet <sup>2</sup>
– Pressure	- psia	- psf
– Temperature	- °F	- °R
– Mass injection	- lbm/sec	- lbm/sec
– Heat Source	- Btu/s OR Btu/lbm-	Btu/s OR Btu/lbm
- **Sign Convention**
  - Mass input to node = positive
  - Mass output from node = negative
  - Heat input to node = positive
  - Heat output from node = negative



# DATA STRUCTURE





# MATHEMATICAL FORMULATION - 1

## MATHEMATICAL CLOSURE - 1

### Principal Variables:

#### Unknown Variables

1. Pressure
2. Flowrate
3. Temperature
4. Specie Concentrations
5. Mass

#### Available Equations to Solve

1. Mass Conservation Equation
2. Momentum Conservation Equation
3. Energy Conservation Equation (First or Second Law of Thermodynamics)
4. Conservation Equations for Mass Fraction of Species
5. Thermodynamic Equation of State



# MATHEMATICAL FORMULATION - 2

## MATHEMATICAL CLOSURE -2

### **Auxiliary Variables:**

Thermodynamic Properties & Flow Resistance Factor

### Unknown Variables

Density

Specific Heats

Viscosity

Thermal Conductivity

Flow Resistance Factor

### Available Equations to Solve

Equilibrium Thermodynamic Relations

[GASP, WASP & GASPAK Property Programs]

Empirical Relations



## MATHEMATICAL FORMULATION - 3

### BOUNDARY CONDITIONS

- Governing equations can generate an infinite number of solutions
- A unique solution is obtained with a given set of boundary conditions
- User provides the boundary conditions

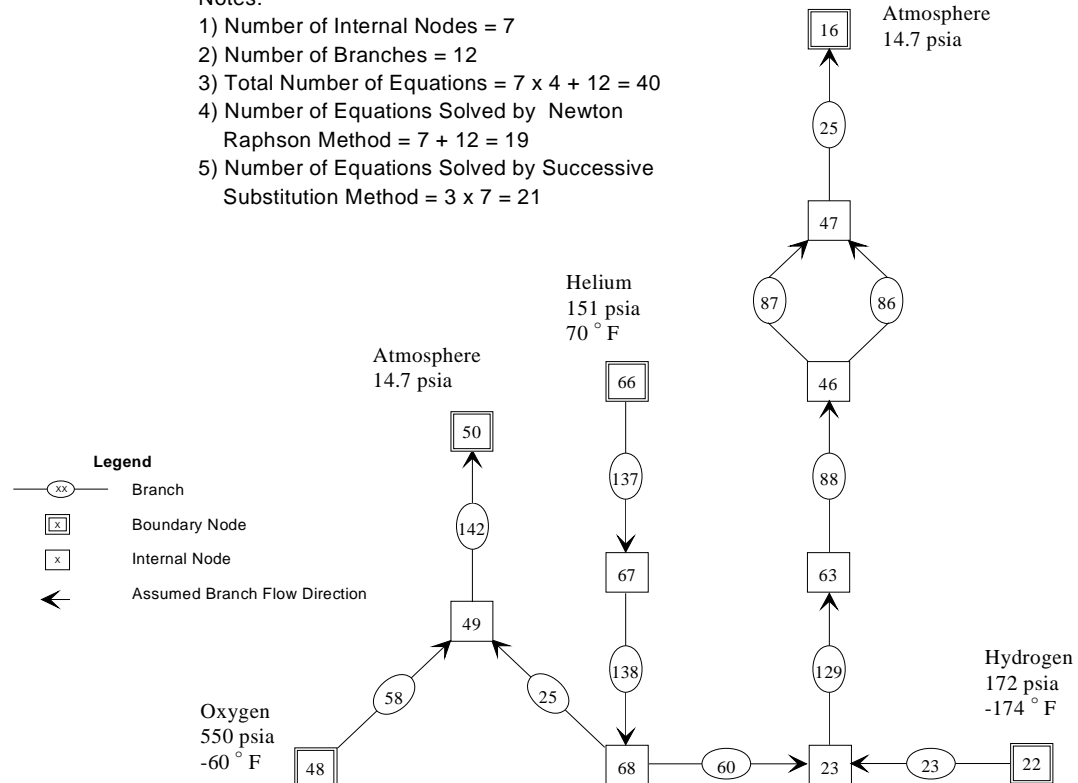


# MATHEMATICAL FORMULATION - 3

## A TYPICAL FLOW CIRCUIT

Notes:

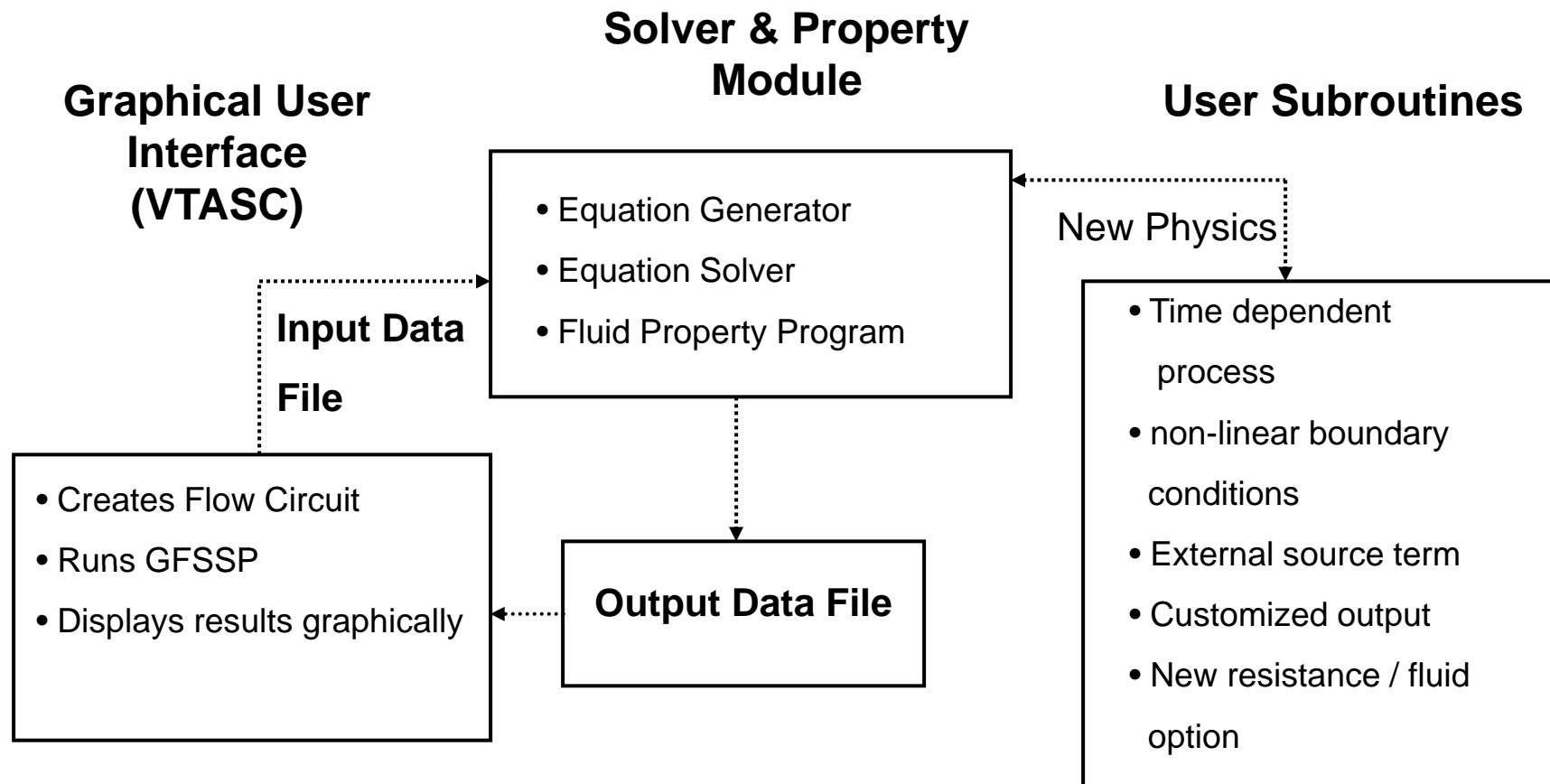
- 1) Number of Internal Nodes = 7
- 2) Number of Branches = 12
- 3) Total Number of Equations =  $7 \times 4 + 12 = 40$
- 4) Number of Equations Solved by Newton Raphson Method =  $7 + 12 = 19$
- 5) Number of Equations Solved by Successive Substitution Method =  $3 \times 7 = 21$

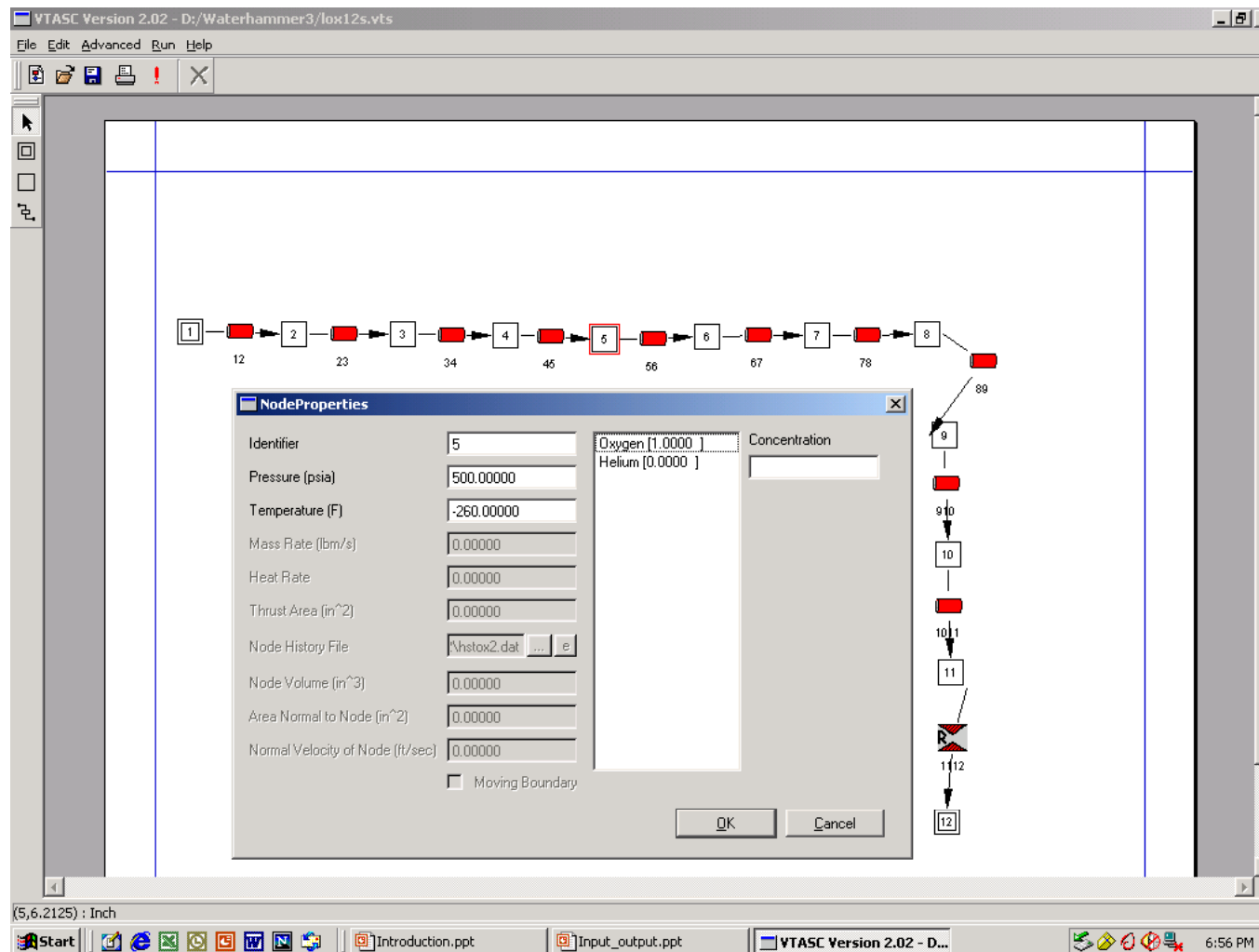
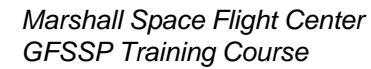






# PROGRAM STRUCTURE

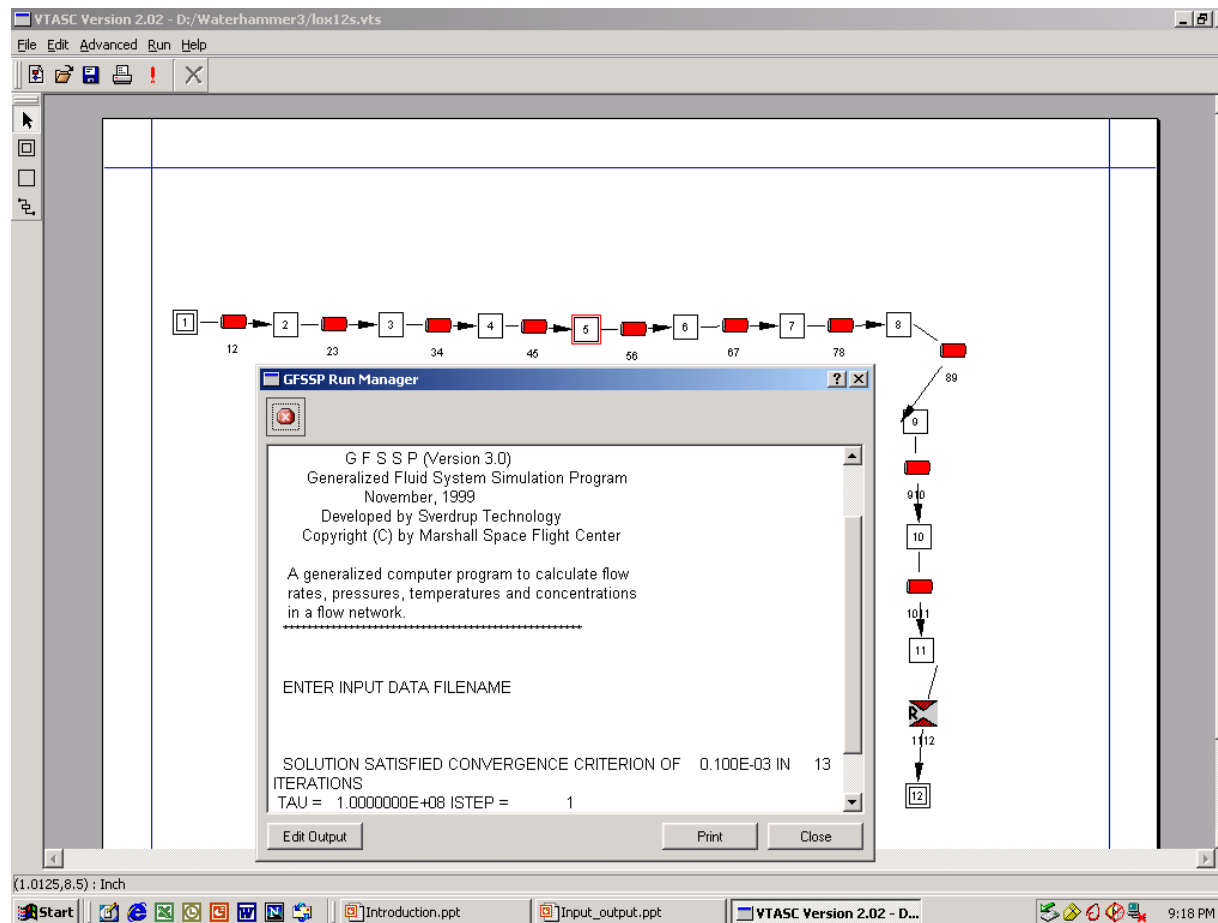






# GRAPHICAL USER INTERFACE - 2

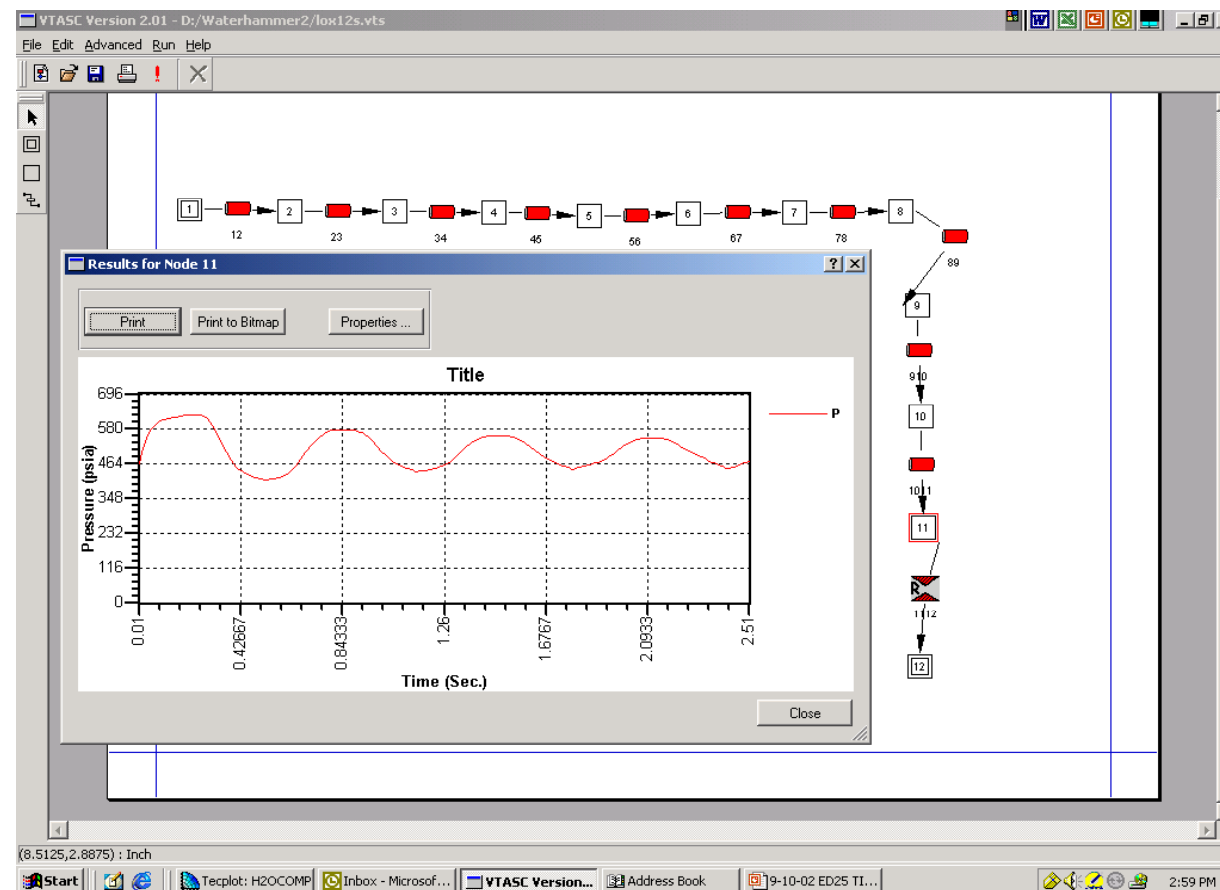
## MODEL RUNNING





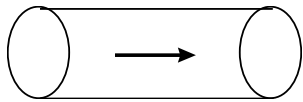
# GRAPHICAL USER INTERFACE - 3

## MODEL RESULTS

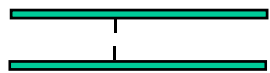




# RESISTANCE & FLUID OPTIONS - 1



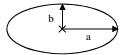
1. Pipe Flow



2. Flow Through a Restriction



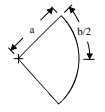
(a) - Rectangle



(b) - Ellipse

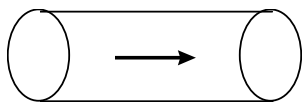


(c) - Concentric Annulus

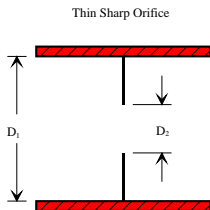


(d) - Circular Sector

3. Non-Circular Duct

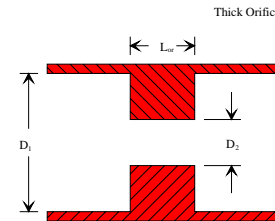


4. Pipe Flow with Entrance & Exit Losses



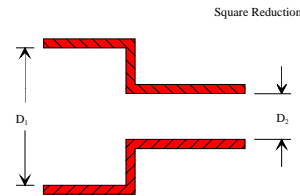
Where:  
 $D_1$  = Pipe Diameter  
 $D_2$  = Orifice Throat Diameter

5. Thin, Sharp Orifice



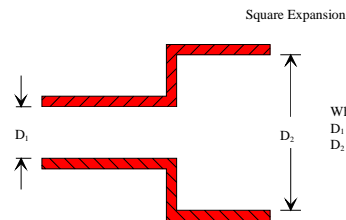
Where:  
 $D_1$  = Pipe Diameter  
 $D_2$  = Orifice Throat Diameter  
 $L_w$  = Orifice Length

6. Thick Orifice



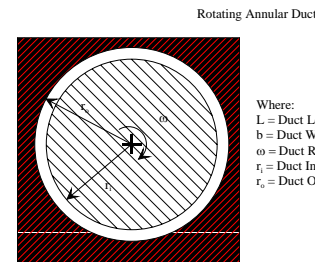
Where:  
 $D_1$  = Upstream Pipe Diameter  
 $D_2$  = Downstream Pipe Diameter

7. Square Reduction



Where:  
 $D_1$  = Upstream Pipe Diameter  
 $D_2$  = Downstream Pipe Diameter

8. Square Expansion

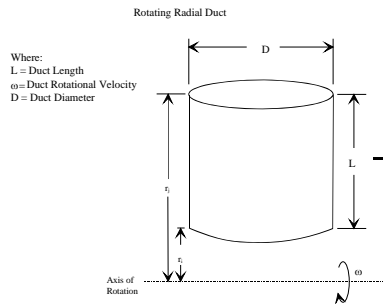


Where:  
 $L$  = Duct Length (Perpendicular to Page)  
 $b$  = Duct Wall Thickness ( $b = r_o - r_i$ )  
 $\omega$  = Duct Rotational Velocity  
 $r_i$  = Duct Inner Radius  
 $r_o$  = Duct Outer Radius

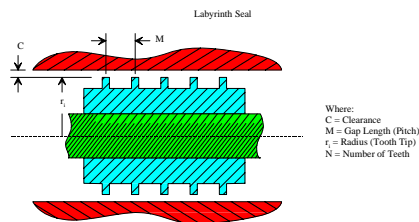
9. Rotating Annular Duct



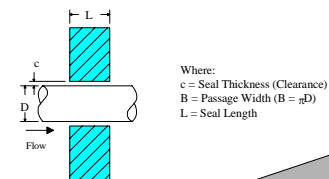
# RESISTANCE & FLUID OPTIONS -2



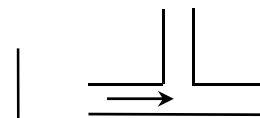
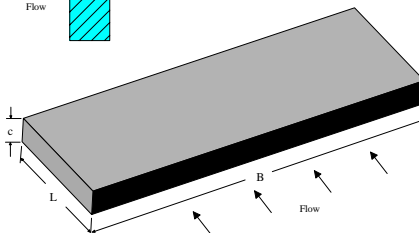
10. Rotating Radial Duct



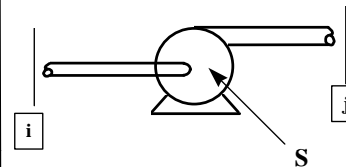
11. Labyrinth Seal



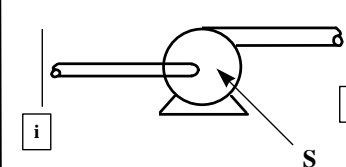
12. Face Seal



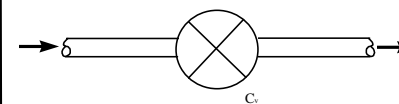
13. Common Fittings & Valves



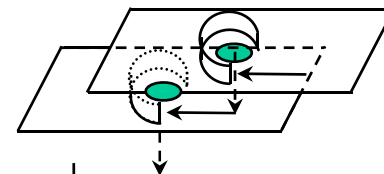
14. Pump Characteristics



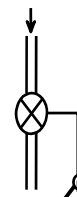
15. Pump Power



16. Valve with Given  $C_v$



17. Viscojet



18. Control Valve



## RESISTANCE & FLUID OPTIONS - 3

### GASP & WASP

Index	Fluid	Index	Fluid
1	HELIUM	7	ARGON
2	METHANE	8	CARBON DIOXIDE
3	NEON	9	FLUORINE
4	NITROGEN	10	HYDROGEN
5	CARBON MONOXIDE	11	WATER
6	OXYGEN	12	RP-1



## RESISTANCE & FLUID OPTIONS - 4

### GASPAK

Index	Fluid	Index	Fluid
1	HELIUM	18	HYDROGEN SULFIDE
2	METHANE	19	KRYPTON
3	NEON	20	PROPANE
4	NITROGEN	21	XENON
5	CO	22	R-11
6	OXYGEN	23	R12
7	ARGON	24	R22
8	CO <sub>2</sub>	25	R32
9	PARAHYDROGEN	26	R123
10	HYDROGEN	27	R124
11	WATER	28	R125
12	RP-1	29	R134A
13	ISOBUTANE	30	R152A
14	BUTANE	31	NITROGEN TRIFLUORIDE
15	DEUTERIUM	32	AMMONIA
16	ETHANE	33	IDEAL GAS
17	ETHYLENE	34	AIR
		35	HYDROGEN PEROXIDE





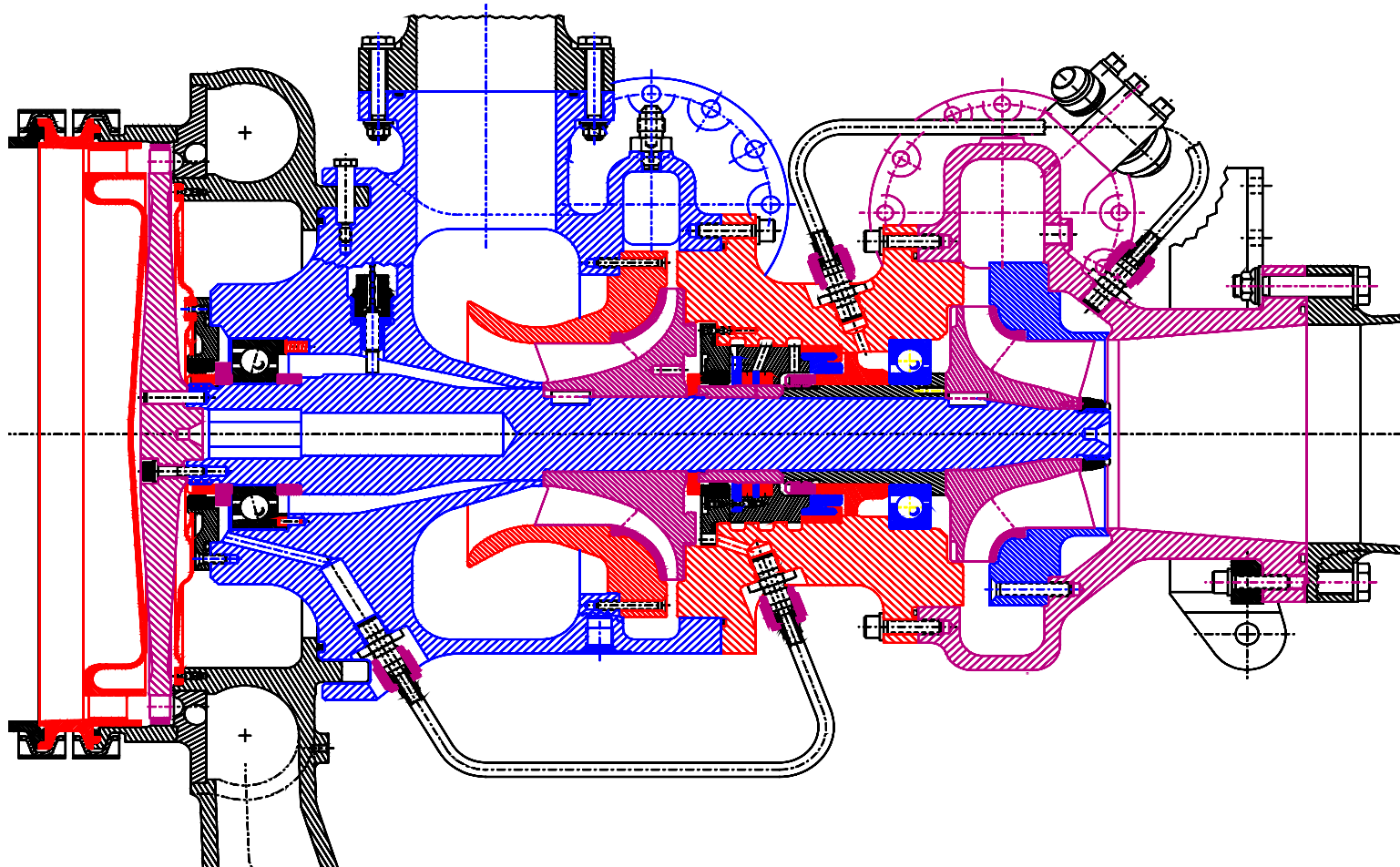
## ADDITIONAL OPTIONS

- Variable Geometry Option
- Variable Rotation Option
- Variable Heat Addition Option
- Turbopump Option
- Heat Exchanger
- Tank Pressurization
- Control Valve



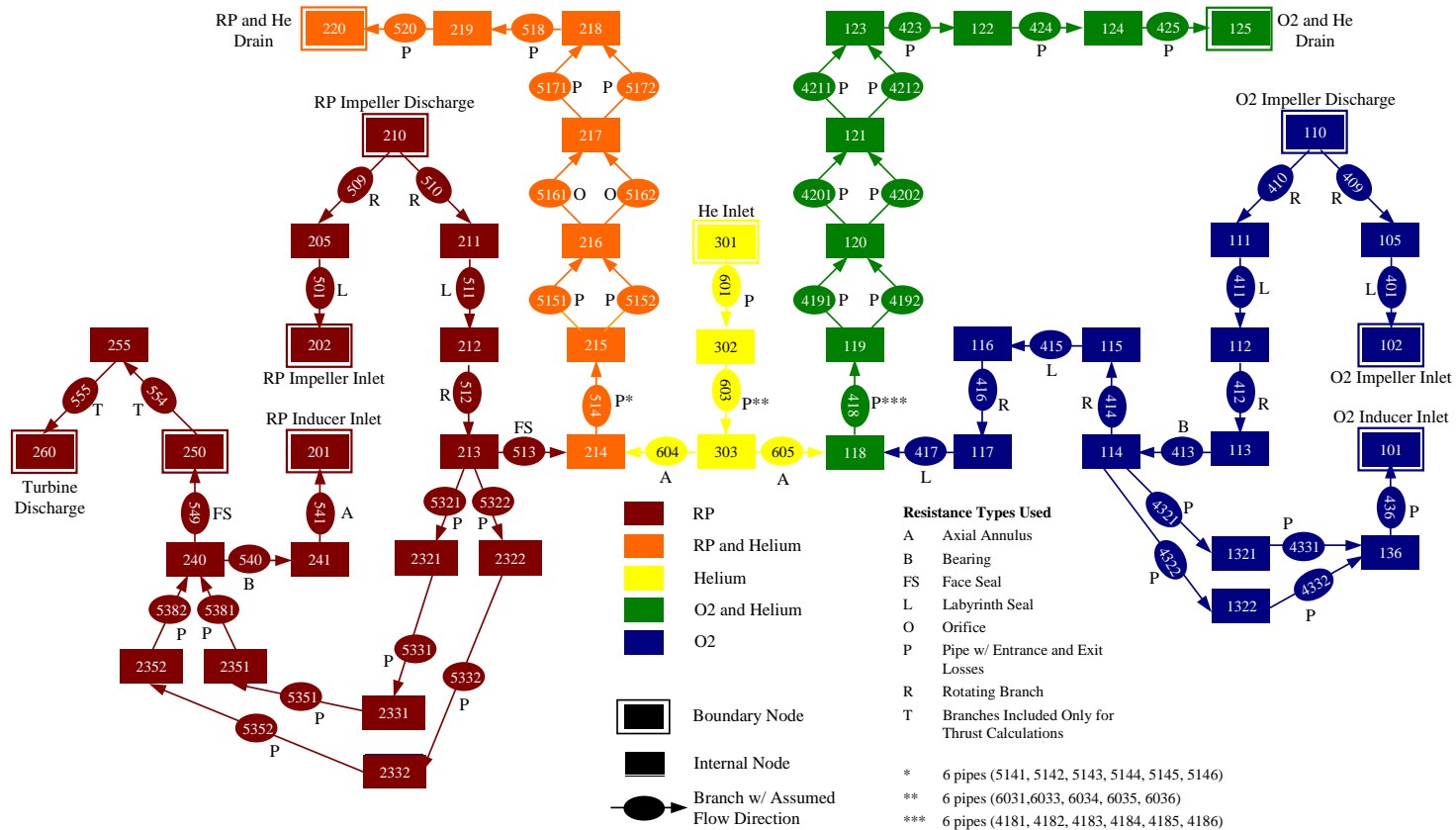
# APPLICATIONS - 1

## FASTRAC TURBOPUMP



## APPLICATIONS - 2

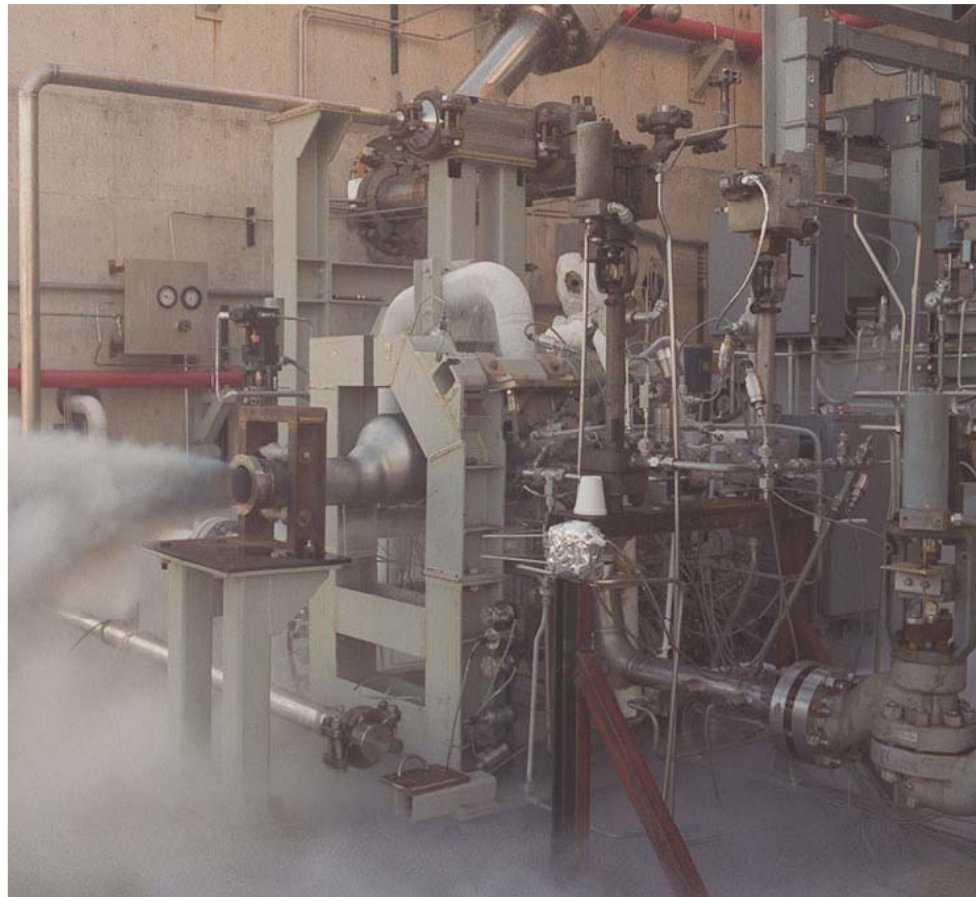
## **GFSSP Model of the Fastrac Turbopump**





## APPLICATIONS - 3

### Turbopump Test to 20000 RPM with Gas Generator

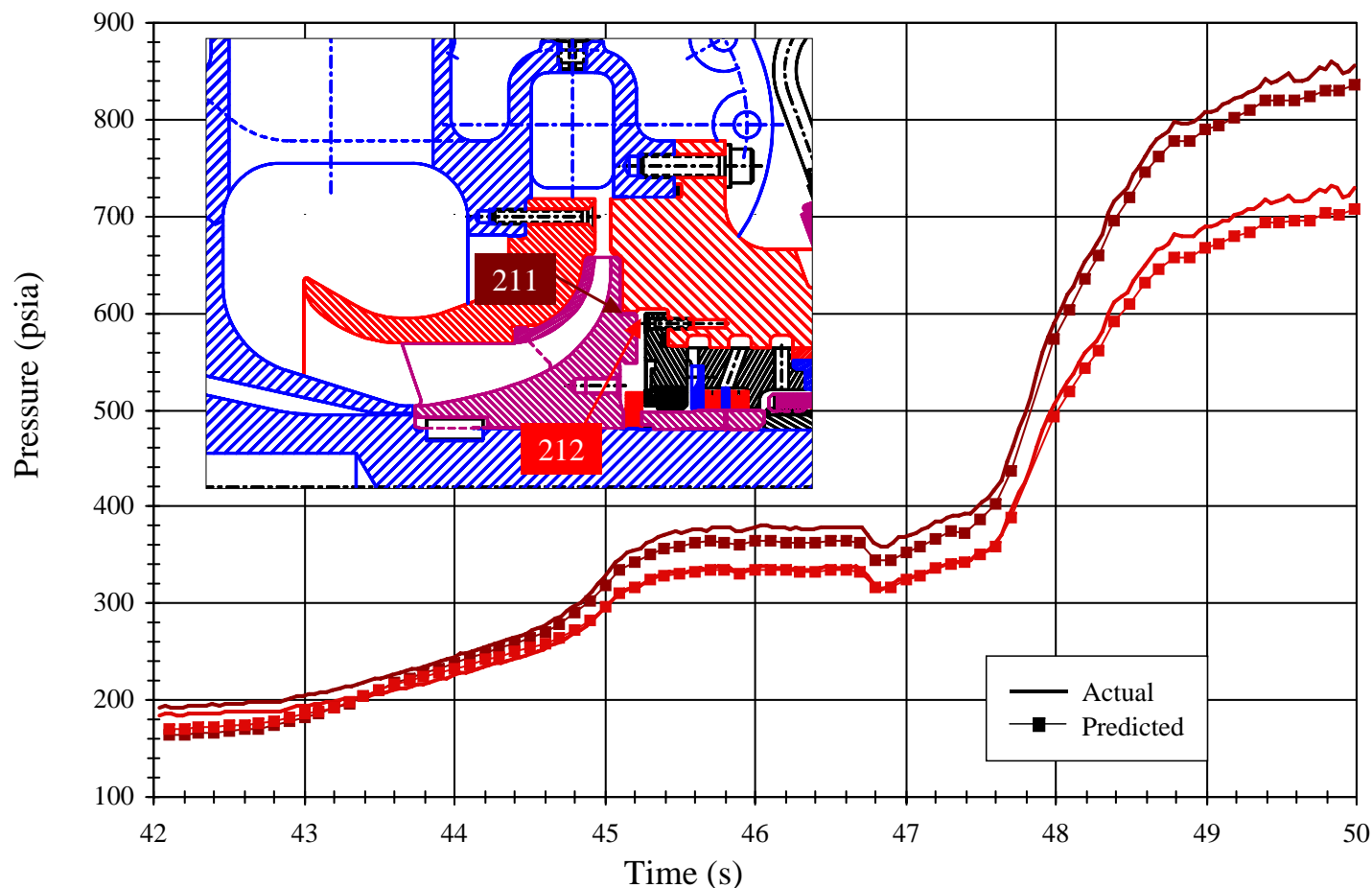




## APPLICATIONS - 4

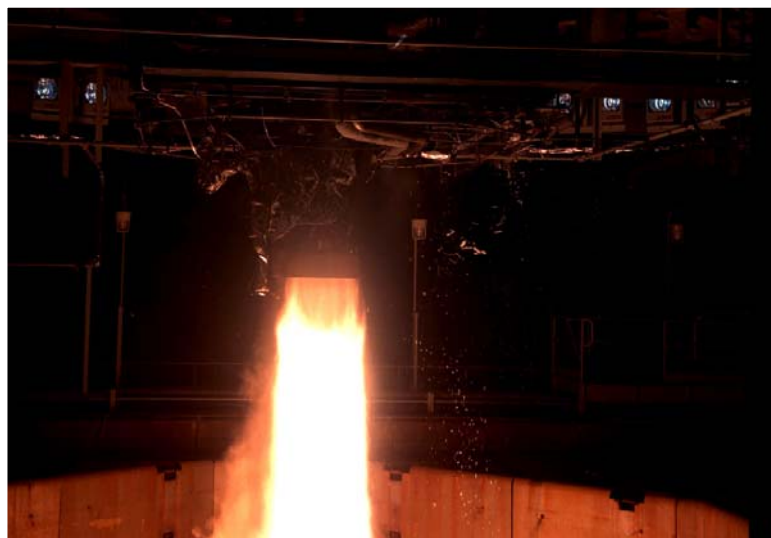
### Fastrac Turbopump Model Results

Pressure history comparison at RP-1 Impeller back face [Labyrinth seal inlet (211) and outlet (212)]





## APPLICATIONS - 5



LOX Tank

RP-1 Tank

Engine  
Interface



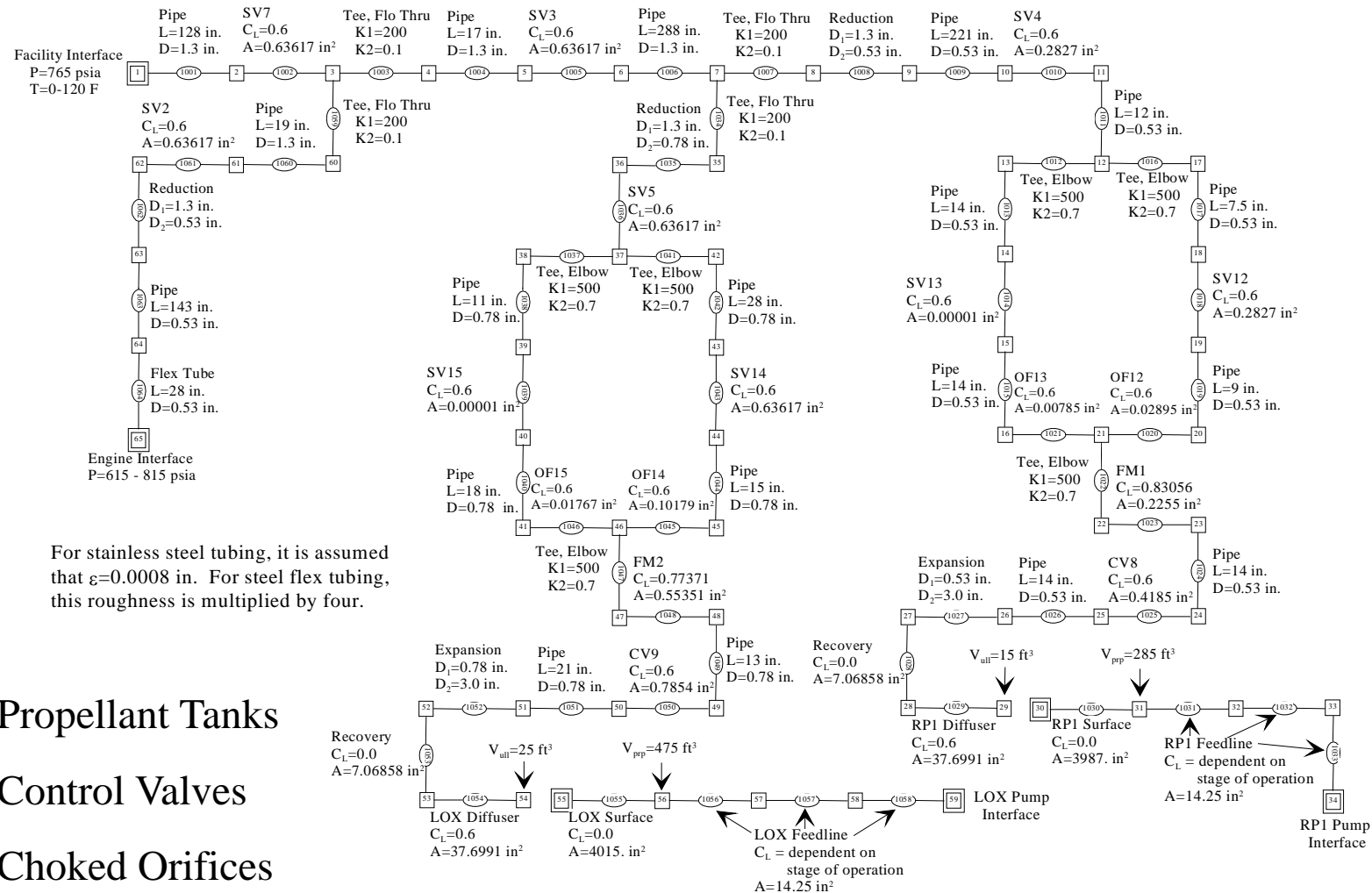


# APPLICATIONS - 6

Marshall Space Flight Center  
GFSSP Training Course



## GFSSP Model of PTA Helium Pressurization System

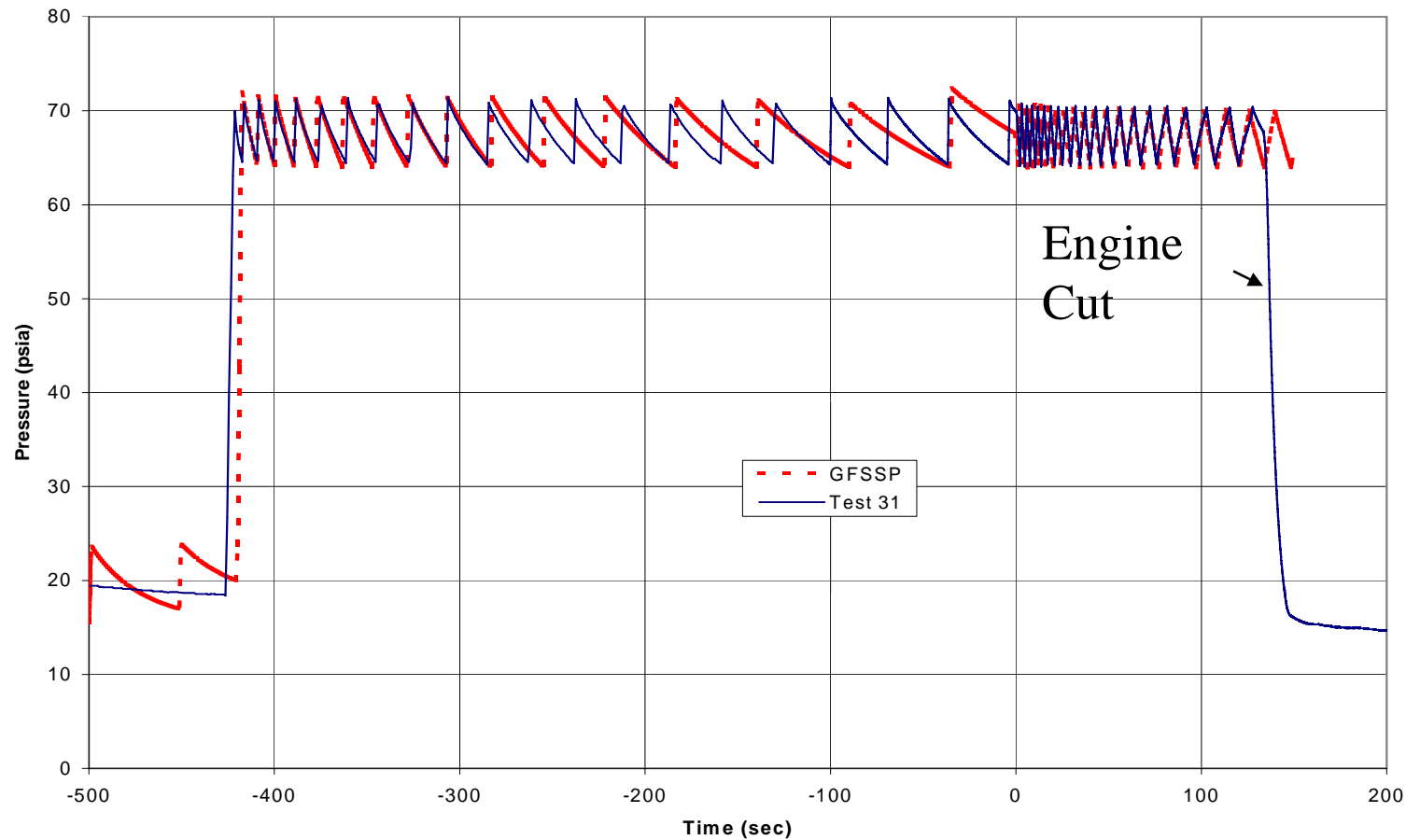


- Propellant Tanks
- Control Valves
- Choked Orifices
- Various fittings



## APPLICATIONS - 7

### Comparison of LOX Ullage Pressure with Test Data

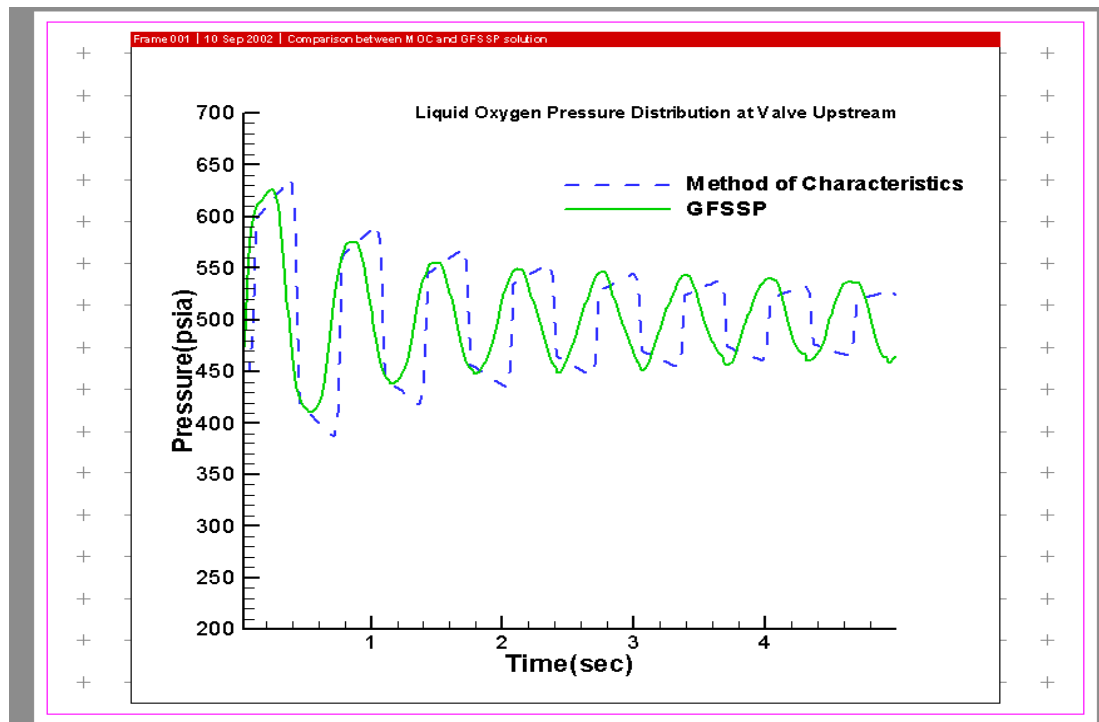
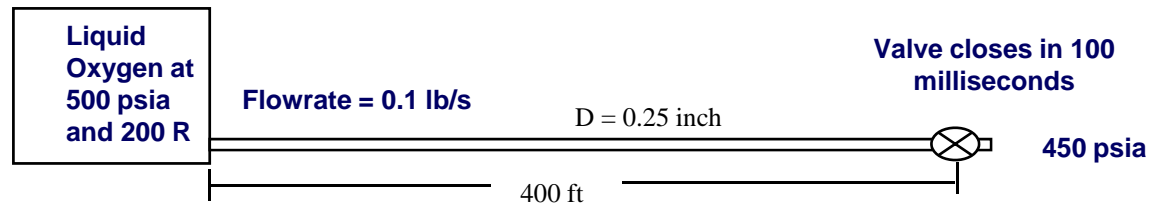






## APPLICATIONS - 8

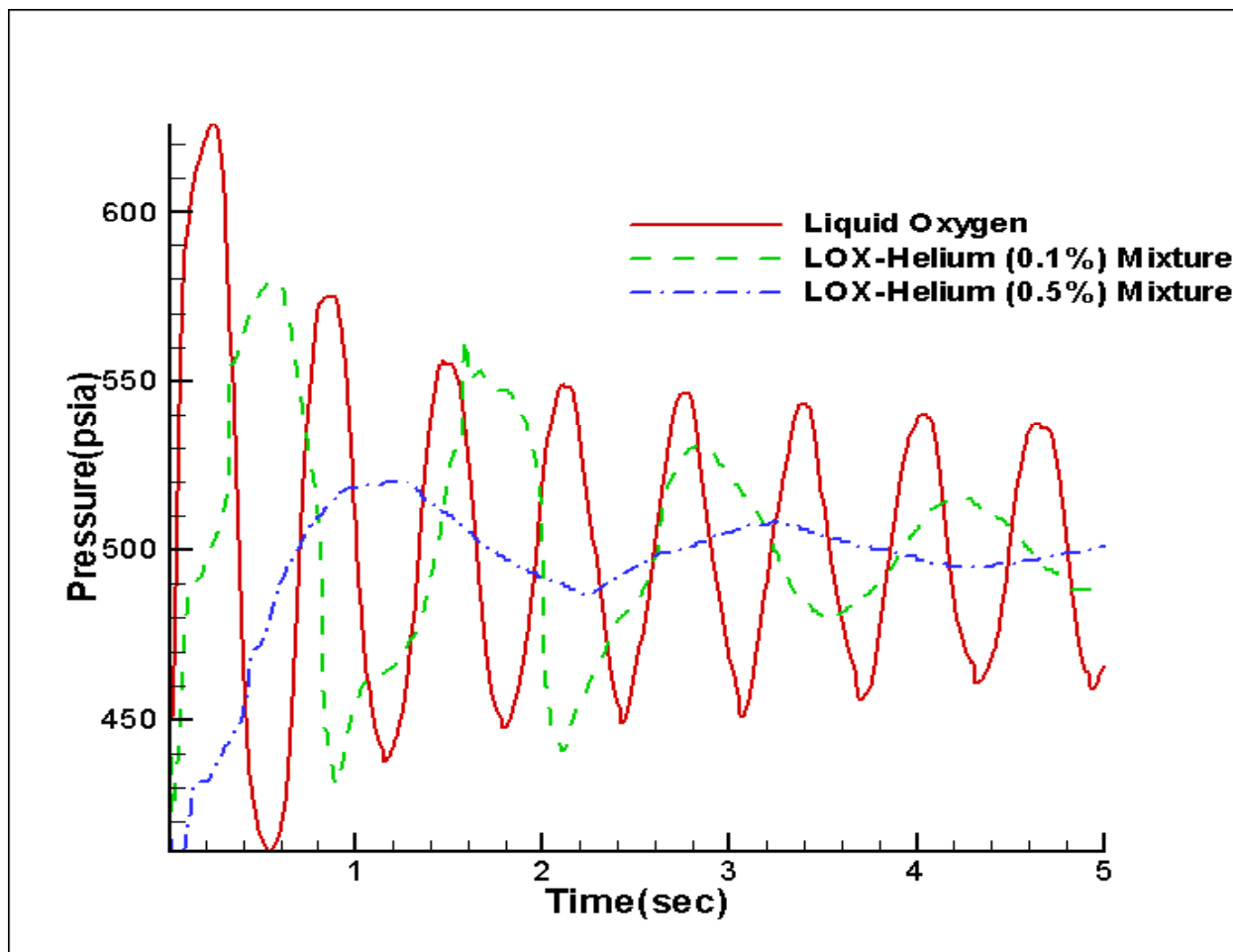
### Verification of Fluid Transient Computation





## APPLICATIONS - 9

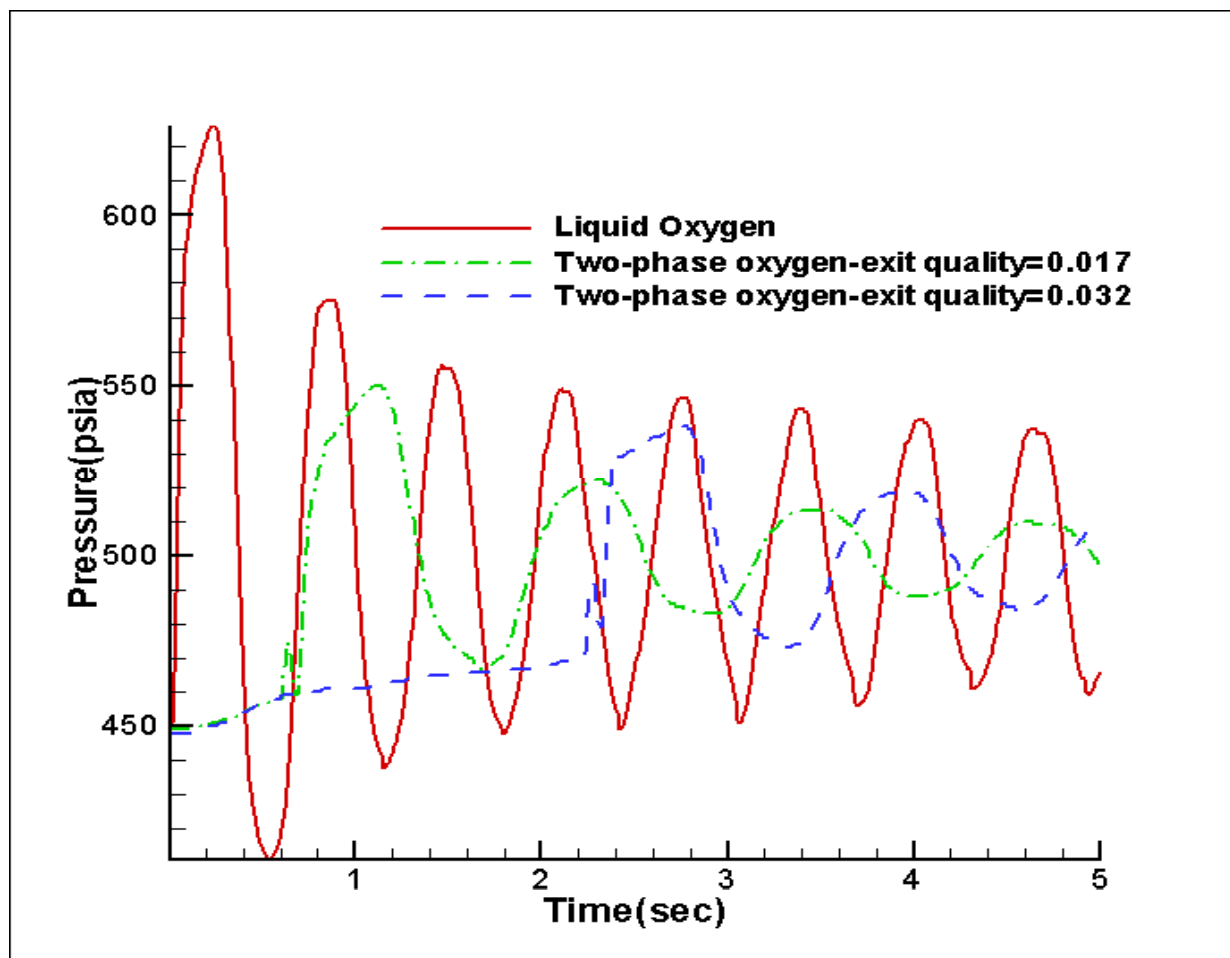
### Fluid Transient in Two phase flow





## APPLICATIONS - 10

### Predicted Fluid Transient Due to Condensation





## SUMMARY - 1

- GFSSP is a finite volume based Network Flow Analyzer
- Flow circuit is resolved into a network consisting of nodes and branches
- Mass, energy and specie conservation are solved at internal nodes. Momentum conservation is solved at branch
- Generalized data structure allows generation of all types of flow network
- Modular code structure allows to add new capabilities with ease



## SUMMARY – 2

- Unique mathematical formulation allows effective coupling of thermodynamics and fluid mechanics
- Numerical scheme is robust; adjustment of numerical control parameters is seldom necessary
- Intuitive Graphical User Interface makes it easy to build, run and evaluate numerical models
- GFSSP has been successfully applied in various applications that included
  - Incompressible & Compressible flows
  - Phase change (Boiling & Condensation)
  - Fluid Mixture
  - Thermodynamic transient (Pressurization & Blowdown)
  - Fluid Transient (Waterhammer)
  - Conjugate Heat Transfer



## SUMMARY – 3

- GFSSP is available from NASA/MSFC's Technology Transfer Office for US Government agencies and contractors
- An Audio-Video Training Course is also available
- More information about the code and its methodology is available at <http://mi.msfc.nasa.gov/GFSSP/index.shtml>





## BACKGROUND -1

Visual Thermo-fluid dynamic Analyzer for Systems and Components (VTASC) is a program designed to efficiently build flow network models for use in the GFSSP program.

- Visually Interactive
  - Eliminates pre-design of models
  - Immediate feedback on model
- Self-Documenting
  - Hard copy of flow network
  - Bitmap image of flow network for inclusion into papers and presentations





## BACKGROUND -2

- Eliminates errors during model building process
  - Automatic node and branch numbering
  - Save and restore models at any point in the model building process
  - Robust
- Pushbutton generation of GFSSP input file
  - Steady and Transient cases
  - Advanced features such as Turbopump, Tank Pressurization and Heat Exchangers
- Run GFSSP directly from VTASC window
  - GFSSP Run Manager acts as VTASC/GFSSP interface

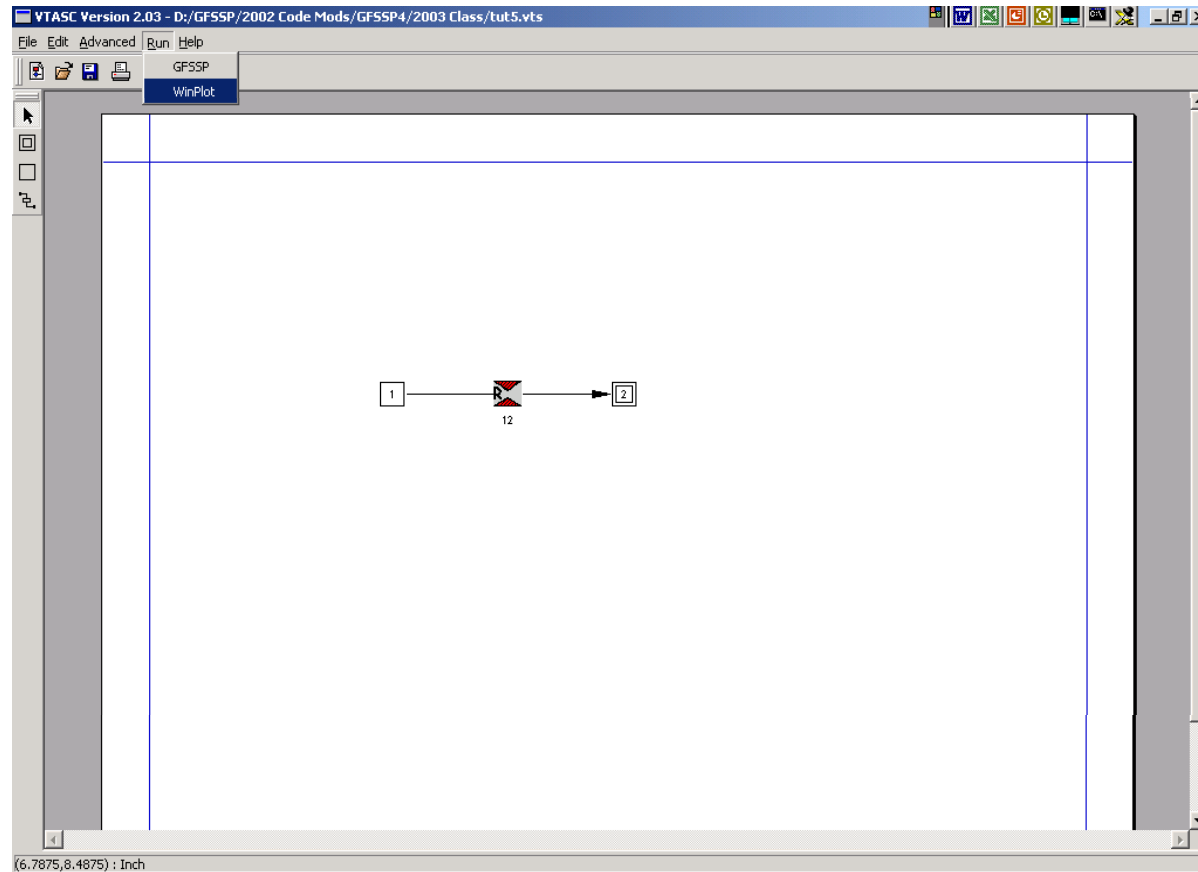


## BACKGROUND -3

- Post-processing capability allows quick study of results
  - Pushbutton access to GFSSP output file
  - Point and click access to output at each node and branch
  - Built-in plotting capability for transient cases
  - Capable of plotting through Winplot
- Cross platform operation
  - Program written in C++
  - Uses cross platform C++ GUI toolkit



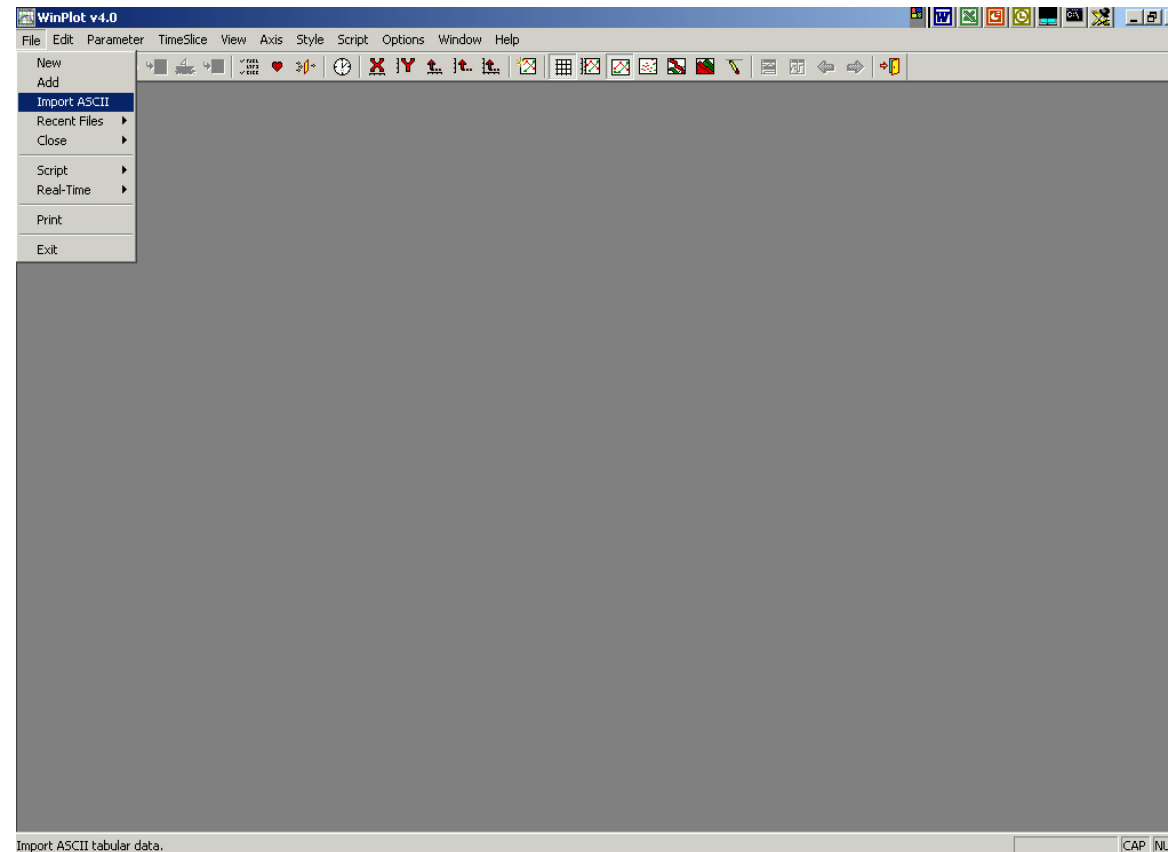
# CREATING A CHART IN WINPLOT -1



- After completing a model run, select Winplot from the VTASC Run Menu



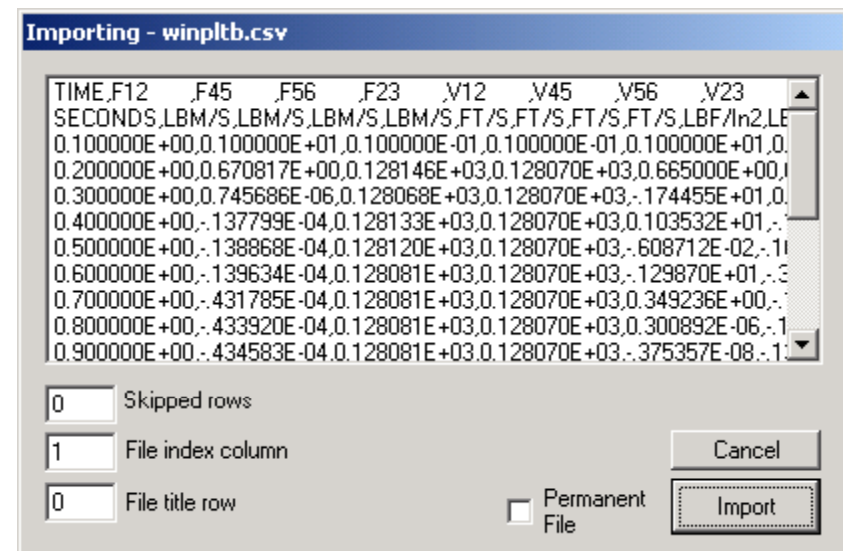
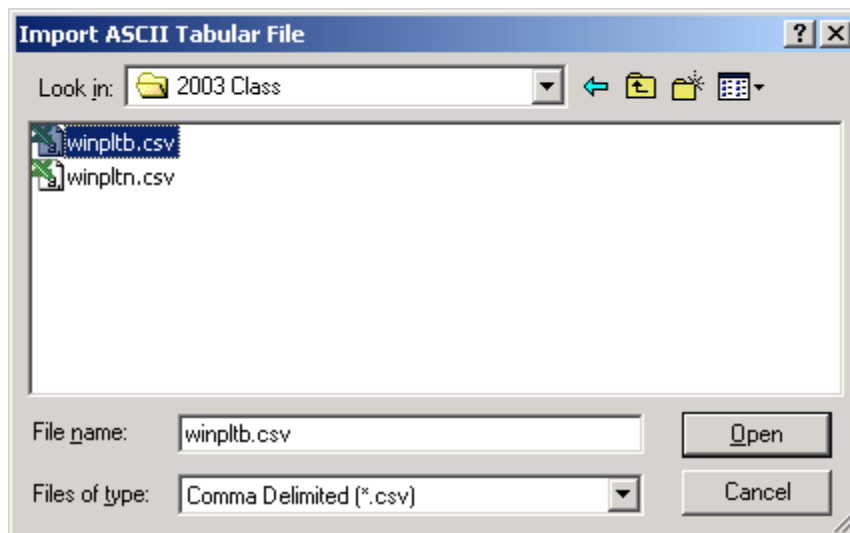
## CREATING A CHART IN WINPLOT -2



- From Winplot's File Menu, select "Import ASCII"



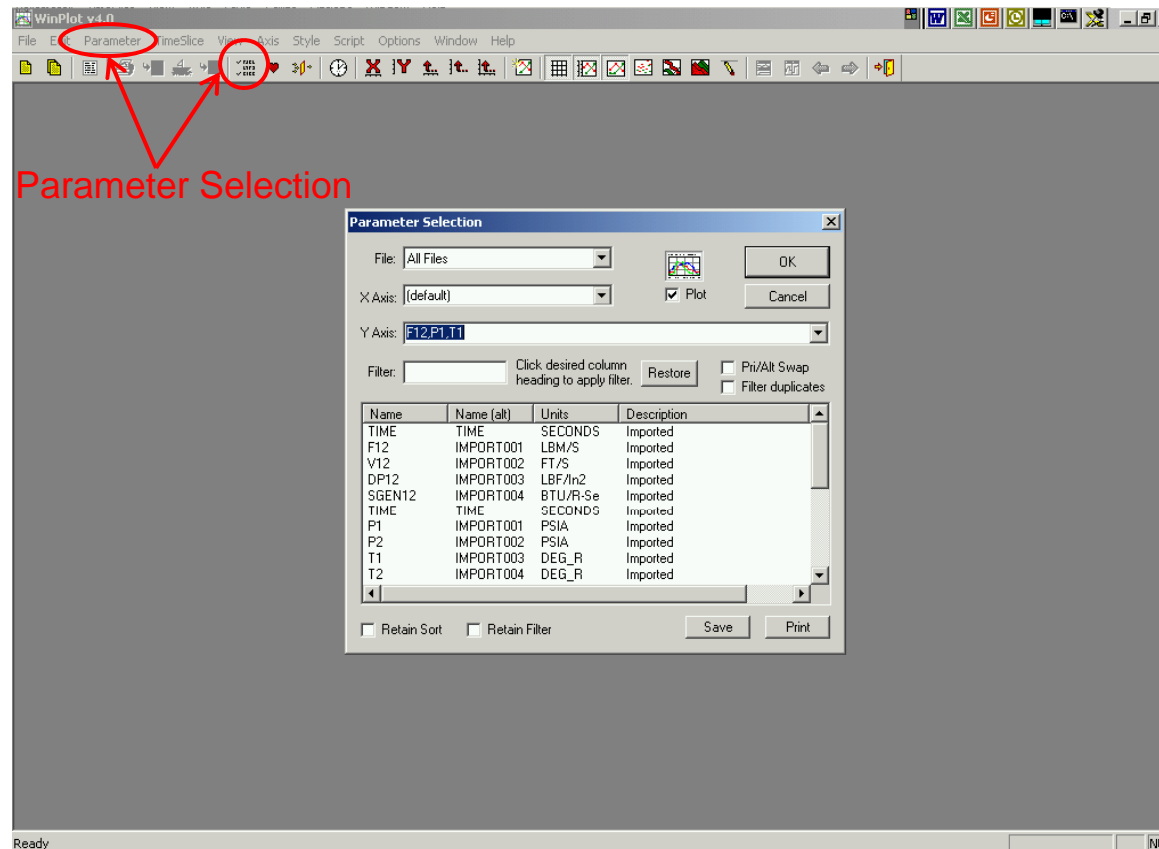
## CREATING A CHART IN WINPLOT -3



- Use the Browse window to select the files you wish to import
- The default GFSSP Winplot files are “winpltb.csv” & “winpltn.csv”
- Selecting a file opens the Importing window. Click Import.



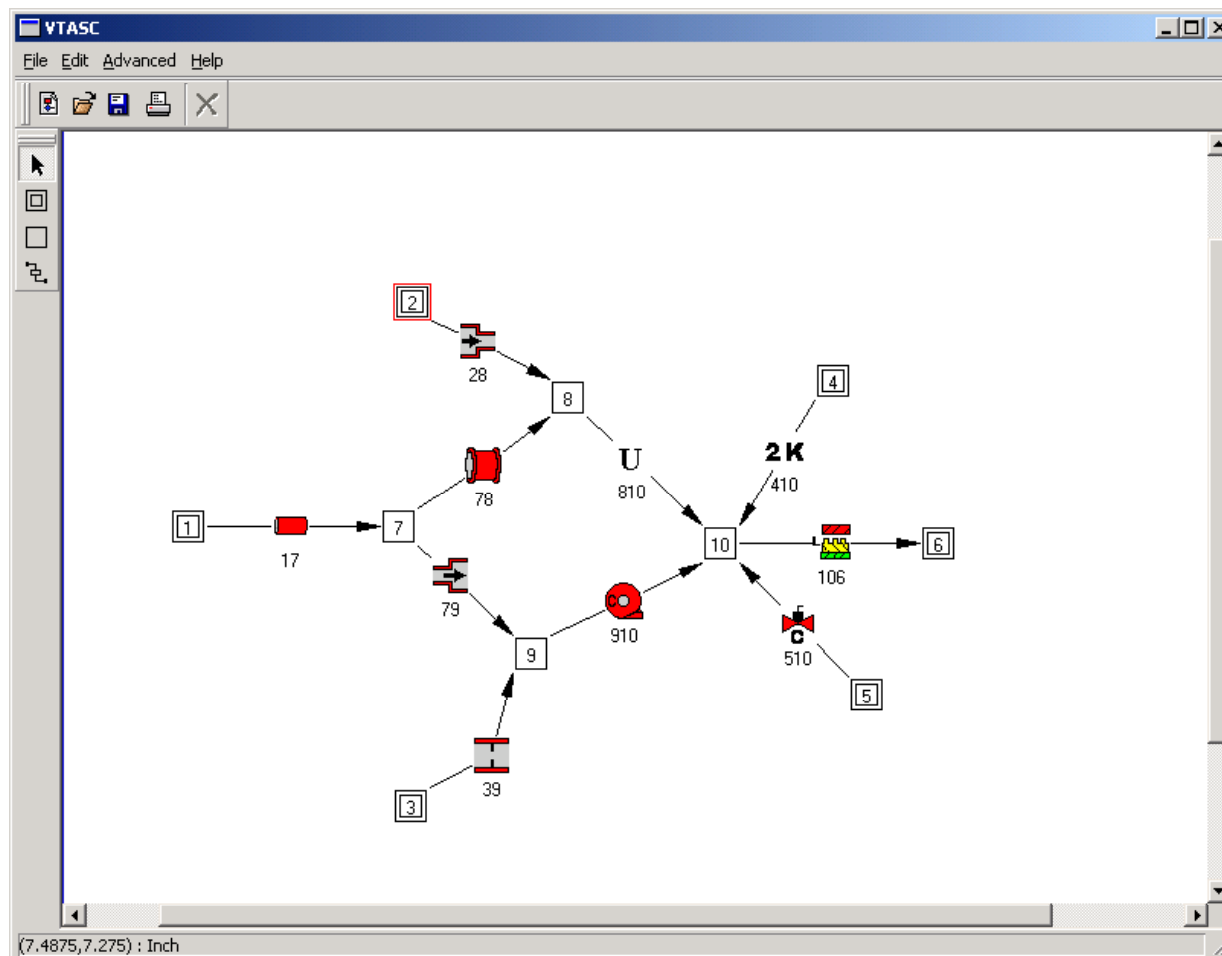
## CREATING A CHART IN WINPLOT -4



- From the Parameter Selection window, select the data you wish to plot

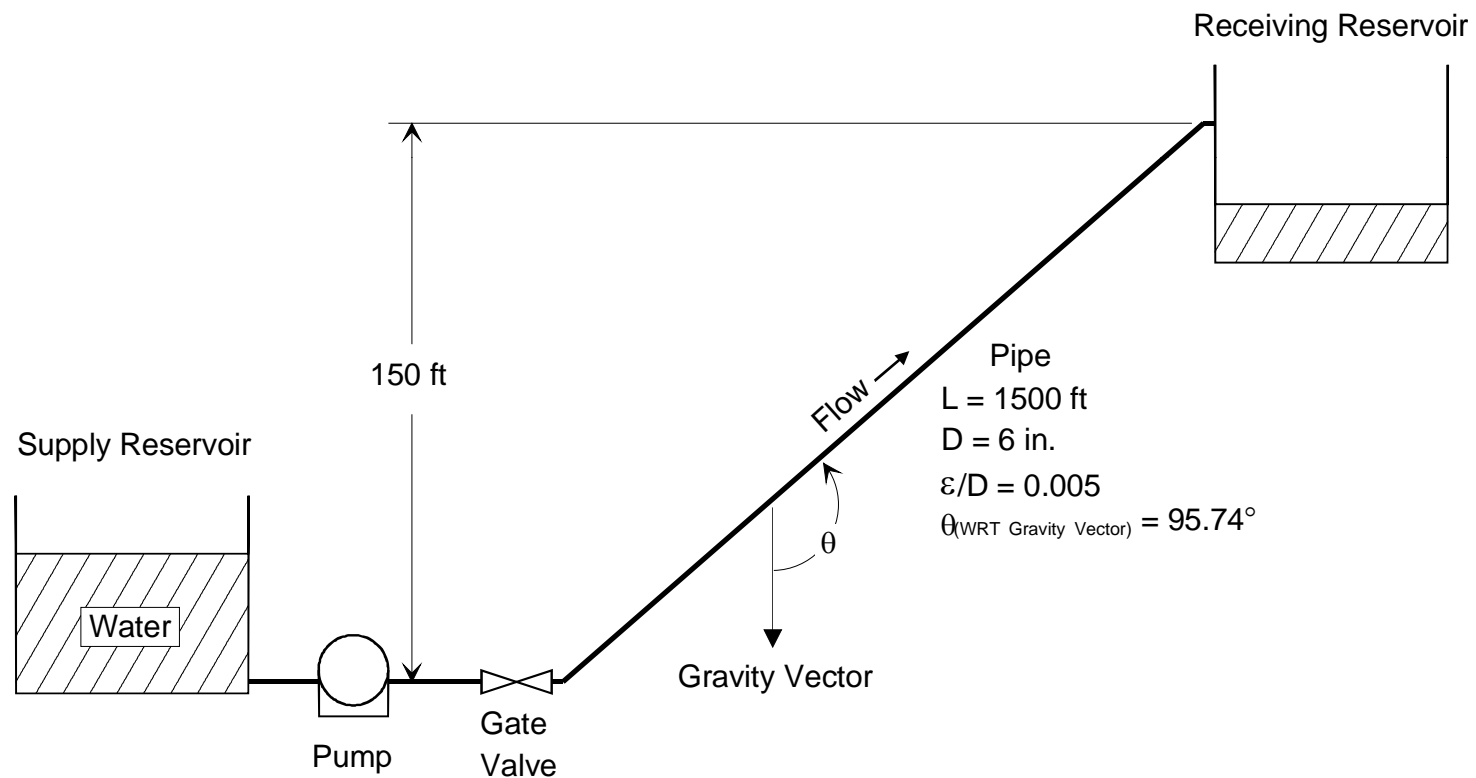


# DEMONSTRATION





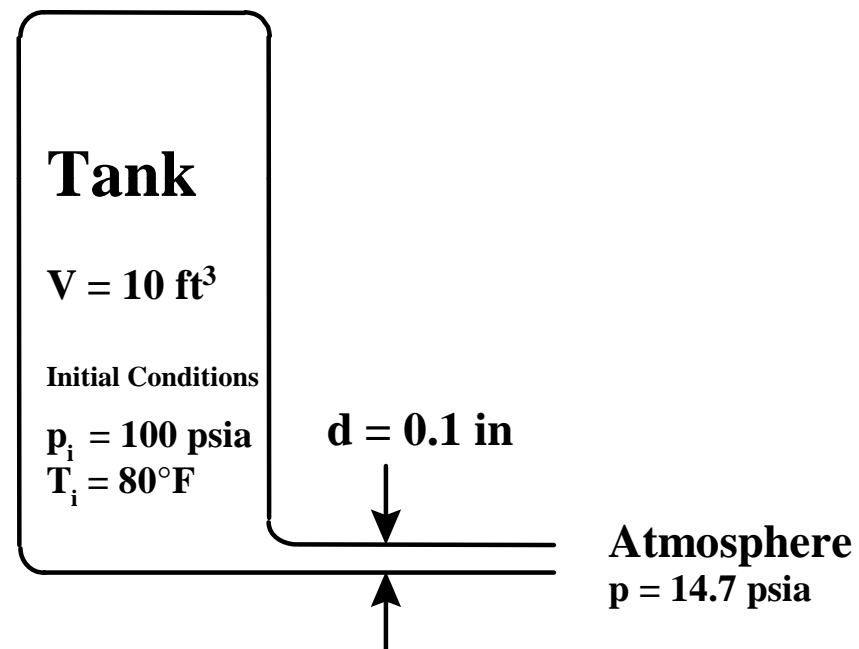
# VTASC DEMONSTRATION PROBLEMS -1





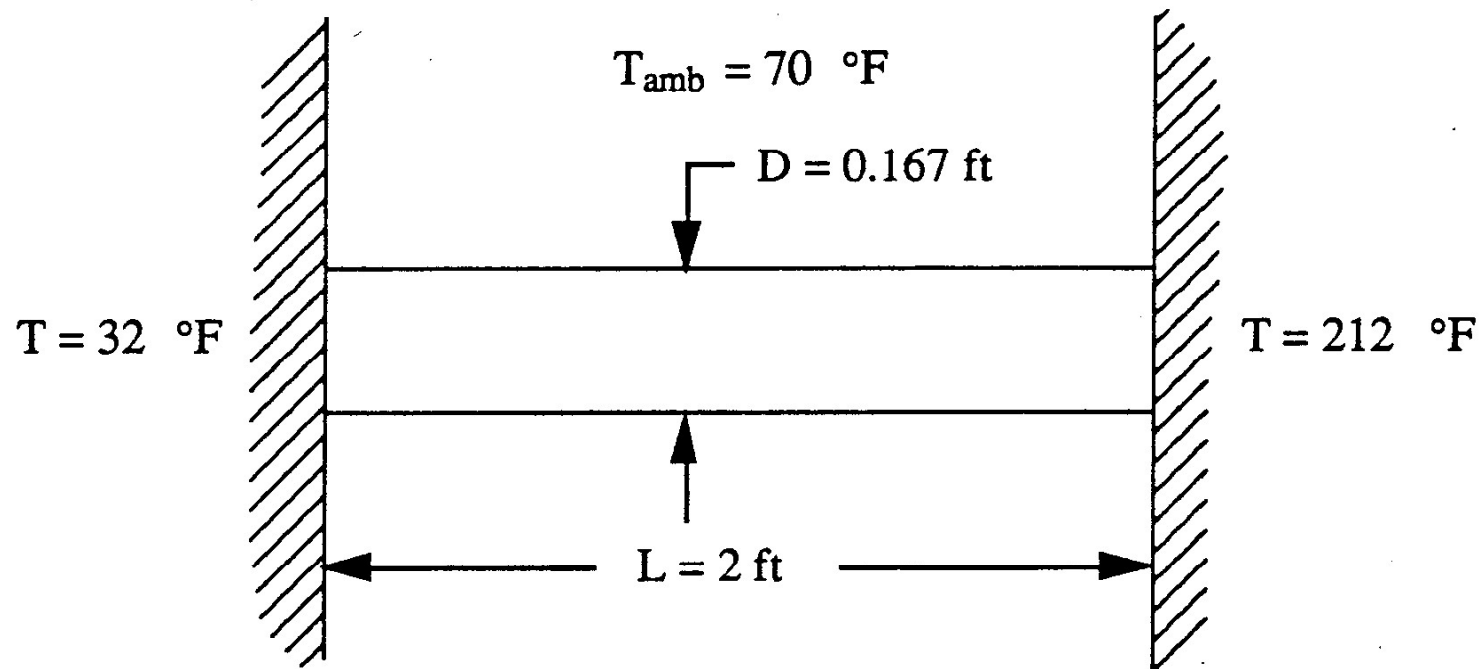


## VTASC DEMONSTRATION PROBLEMS -2





## VTASC DEMONSTRATION PROBLEMS -3



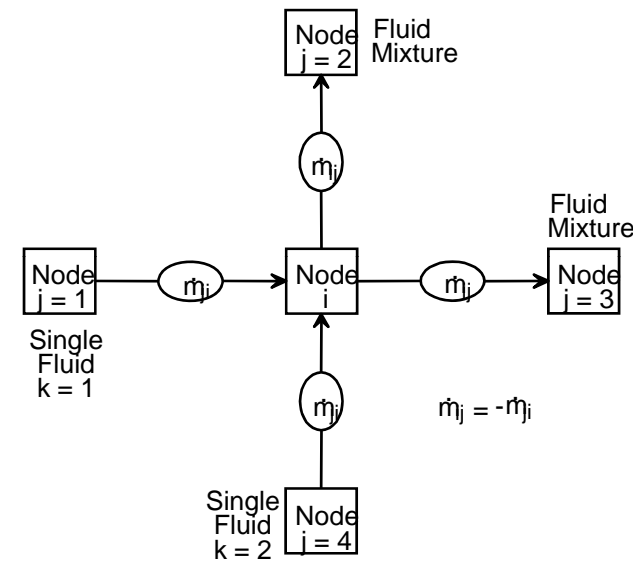


## SUMMARY

- VTASC is a flow network model builder for use with GFSSP
- Flow networks can be designed and modified interactively using a “Point and Click” paradigm
- Generates GFSSP Version 4.0 compatible input files



# MATHEMATICAL FORMULATION



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# Content

- Mathematical Closure
- Governing Equations
- Solution Procedure



# MATHEMATICAL CLOSURE

## Problem of a Steady State Flow Network

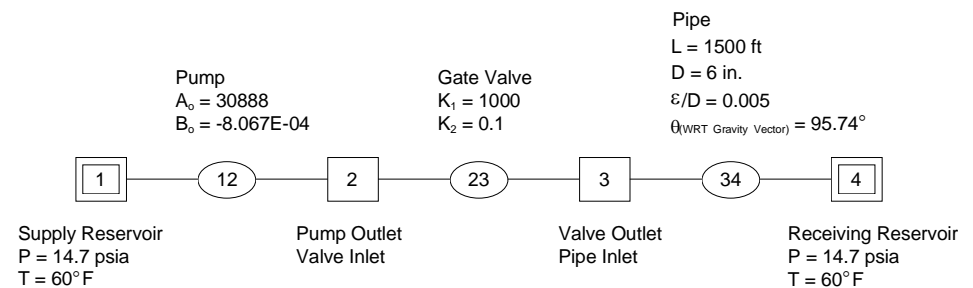
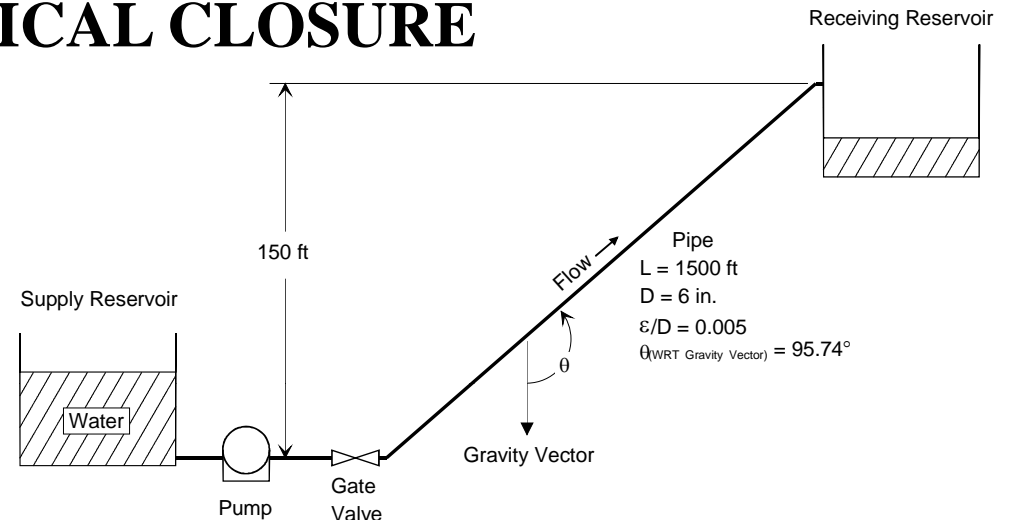
- **Given** : Pressures and Temperatures at Boundary Nodes
- **Find** : Pressures and Temperatures at Internal Nodes and Flowrates in Branches

### *Primary Variables*

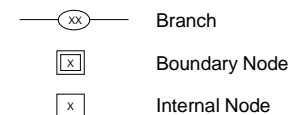
$$p_2, p_3, T_2, T_3, m_{12}, m_{23}, m_{34}$$

### *Secondary Variables*

$$\rho_2, \rho_3, \mu_2, \mu_3$$



#### Legend





## MATHEMATICAL CLOSURE

### Problem of an Unsteady Flow

#### Network

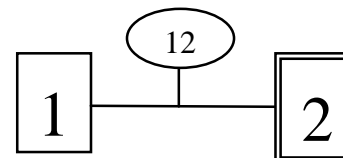
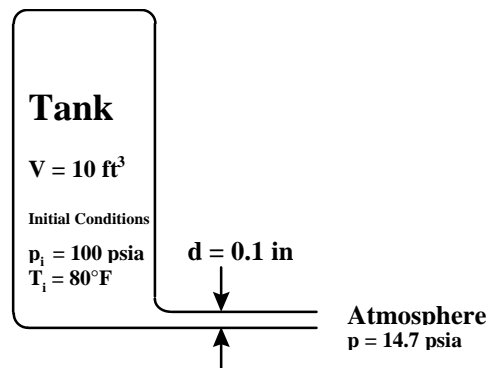
- **Given** : Pressures and Temperatures at Boundary Nodes and Initial Values at Internal Nodes
- **Find** : Pressures and Temperatures at Internal Nodes and Flowrates in Branches with Time.

#### *Primary Variables*

$$p_1(\tau), T_1(\tau), m_1(\tau), \dot{m}(\tau)$$

#### *Secondary Variables*

$$\rho_1(\tau), \mu_1(\tau)$$





# MATHEMATICAL CLOSURE

## Principal Variables:

### Unknown Variable

### Available Equations to Solve

- |                                       |  |
|---------------------------------------|--|
| 1. Pressure                           | 1. Mass Conservation Equation                          |
| 2. Flowrate                           | 2. Momentum Conservation Equation                      |
| 3. Temperature                        | 3. Energy Conservation Equation                        |
| 4. Specie Concentrations<br>(Mixture) | 4. Conservation Equations for Mass Fraction of Species |
| 5. Mass (Unsteady)                    | 5. Thermodynamic Equation of State                     |





# MATHEMATICAL CLOSURE

## Secondary Variables:

### Thermodynamic & Thermophysical Properties

#### Unknown Variable

#### Available Equations to Solve

Density

Specific Heats

Viscosity

Thermal Conductivity

Equilibrium Thermodynamic Relations

[GASP, WASP & GASPAK Property Programs]

### Flow Resistance

#### Unknown Variable

#### Available Equations to Solve

1. Friction Factor

2. Loss Coefficient

1. Empirical Relations

2. User Specified



# GOVERNING EQUATIONS

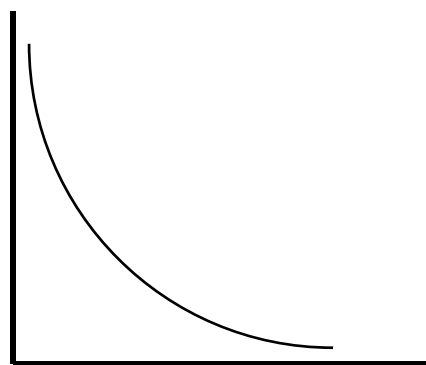
- Mass Conservation
- Momentum Conservation
- Energy Conservation
- Fluid Species Conservation
- Equation of State
- Mixture Property



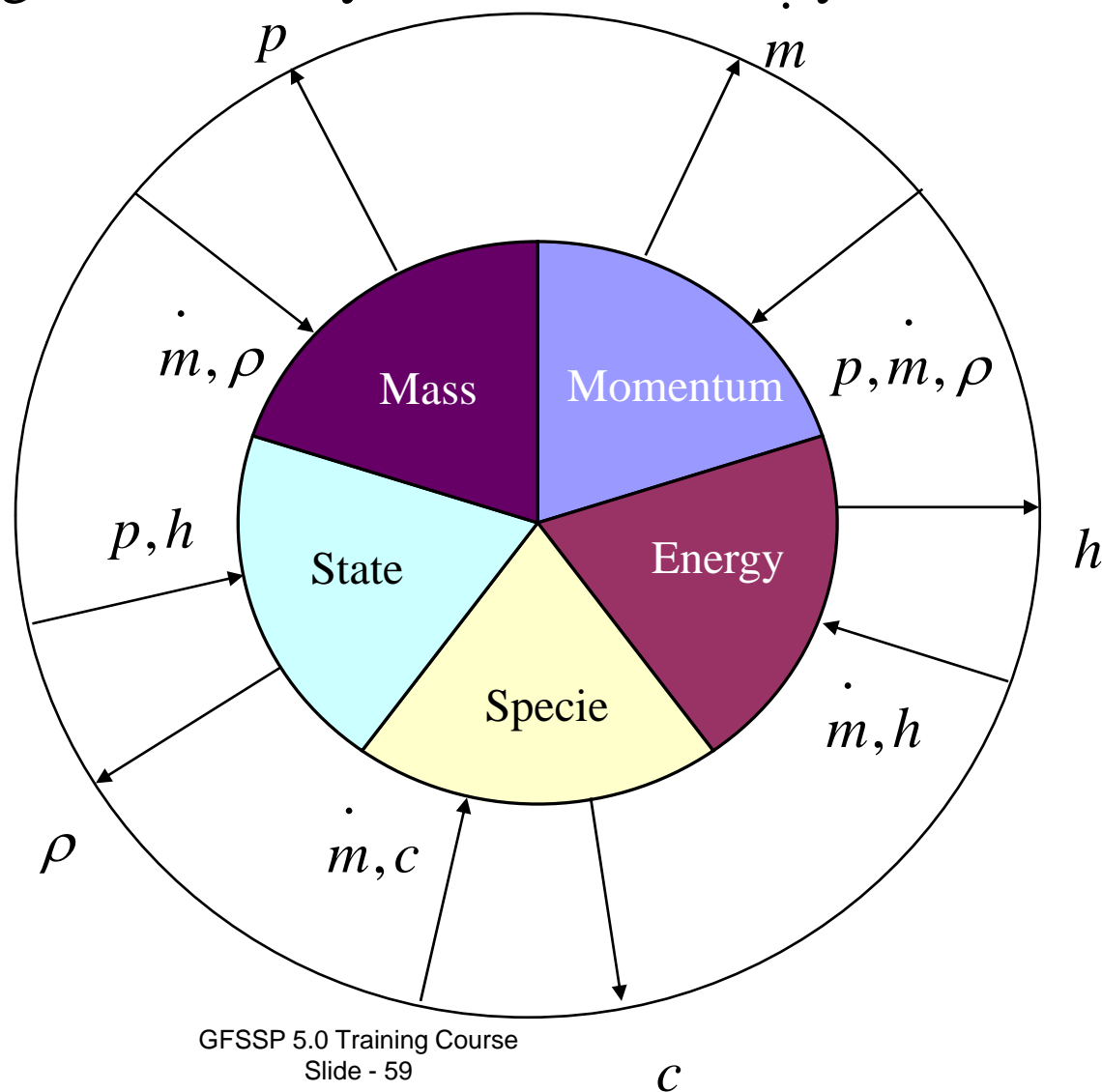
## Coupling of Thermodynamics & Fluid Dynamics

$p$  – Pressure  
 $\dot{m}$  - Flowrate  
 $h$  - Enthalpy  
 $c$  - Concentration  
 $\rho$  - Density

Error



Iteration Cycle



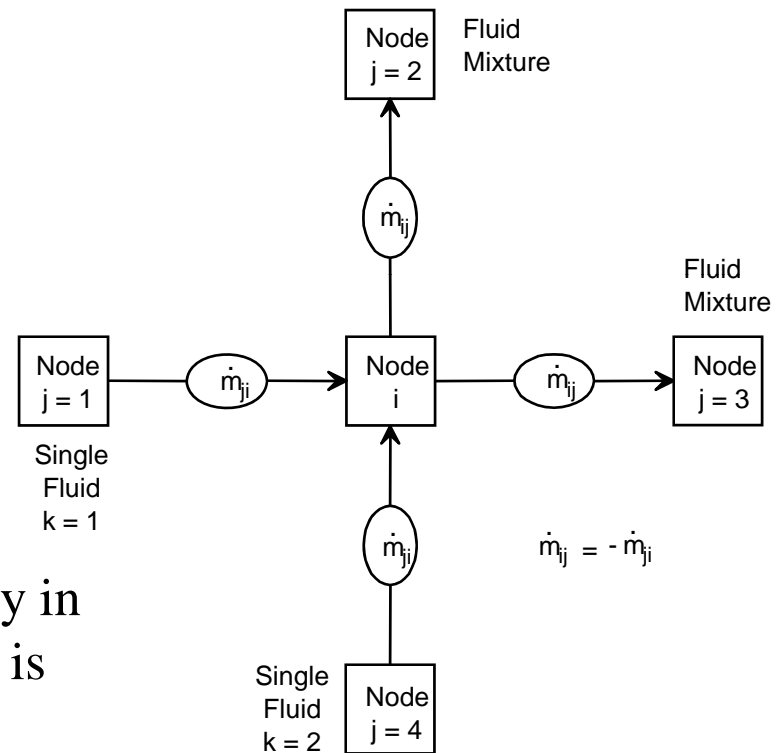


# GOVERNING EQUATIONS

## MASS CONSERVATION EQUATION

$$\frac{m_{\tau+\Delta\tau} - m_{\tau}}{\Delta\tau} = \sum_{j=1}^{j=n} \dot{m}_{ij}$$

Note : Pressure does not appear explicitly in Mass Conservation Equation although it is earmarked for calculating pressures





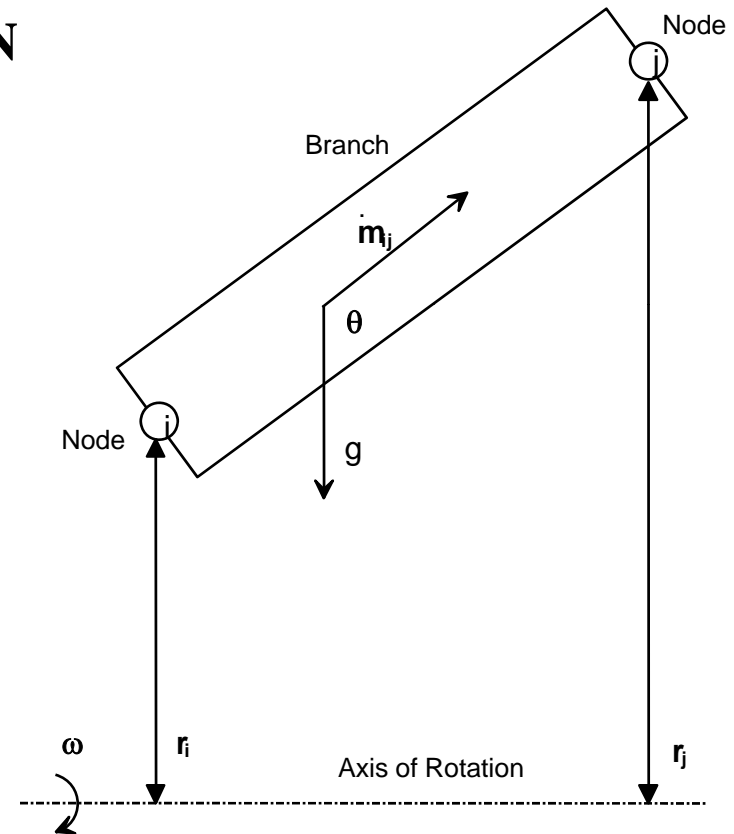
# GOVERNING EQUATIONS

## MOMENTUM CONSERVATION EQUATION

- Represents Newton's Second Law of Motion

$$\text{Mass} \times \text{Acceleration} = \text{Forces}$$

- Unsteady
- Longitudinal Inertia
- Transverse Inertia
- Pressure
- Gravity
- Friction
- Centrifugal
- Shear Stress
- Moving Boundary
- Normal Stress
- External Force





## MOMENTUM CONSERVATION EQUATION

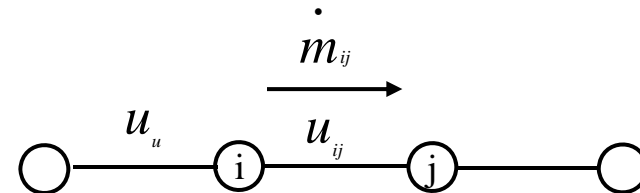
*Mass x Acceleration Terms in GFSSP*

Unsteady

$$\frac{(mu_{ij})_{\tau+\Delta\tau} - (mu_{ij})_{\tau}}{g_c \Delta\tau}$$

Longitudinal Inertia

$$MAX|\dot{m}_{ij}, 0|(u_{ij} - u_u) - MAX|-\dot{m}_{ij}, 0|(u_{ij} - u_u)$$





## MOMENTUM CONSERVATION EQUATION

### *Force Terms in GFSSP*

#### Pressure

$$(p_i - p_j)A_{ij}$$

#### Gravity

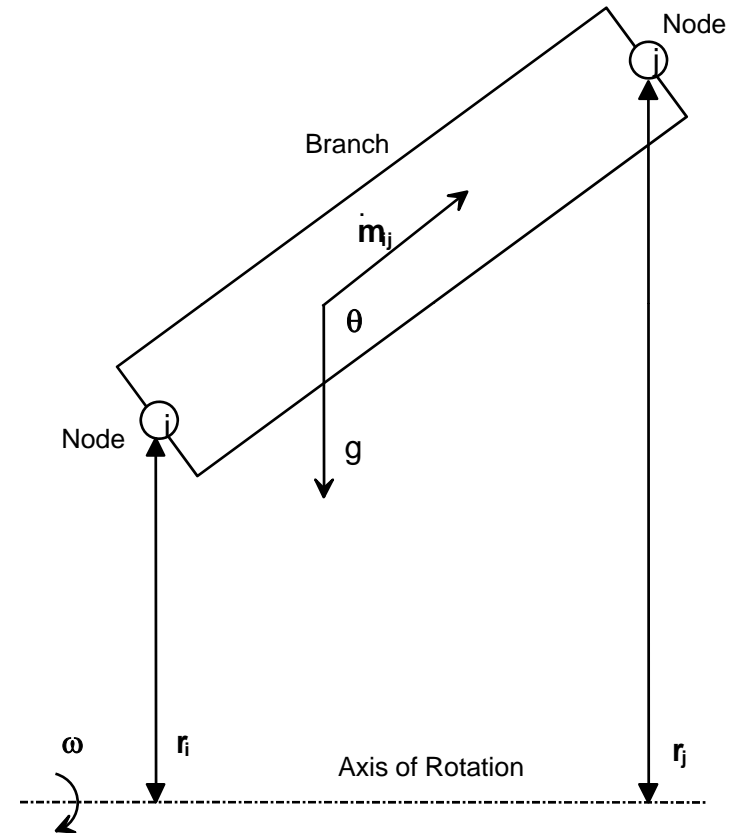
$$\frac{\rho g V \cos \theta}{g_c}$$

#### Friction

$$-K_f \dot{m}_{ij} \left| \dot{m}_{ij} \right| A_{ij}$$

#### Centrifugal

$$\frac{\rho K_{rot} \omega^2 A}{g_c}$$





# GOVERNING EQUATIONS

## ENERGY CONSERVATION EQUATION

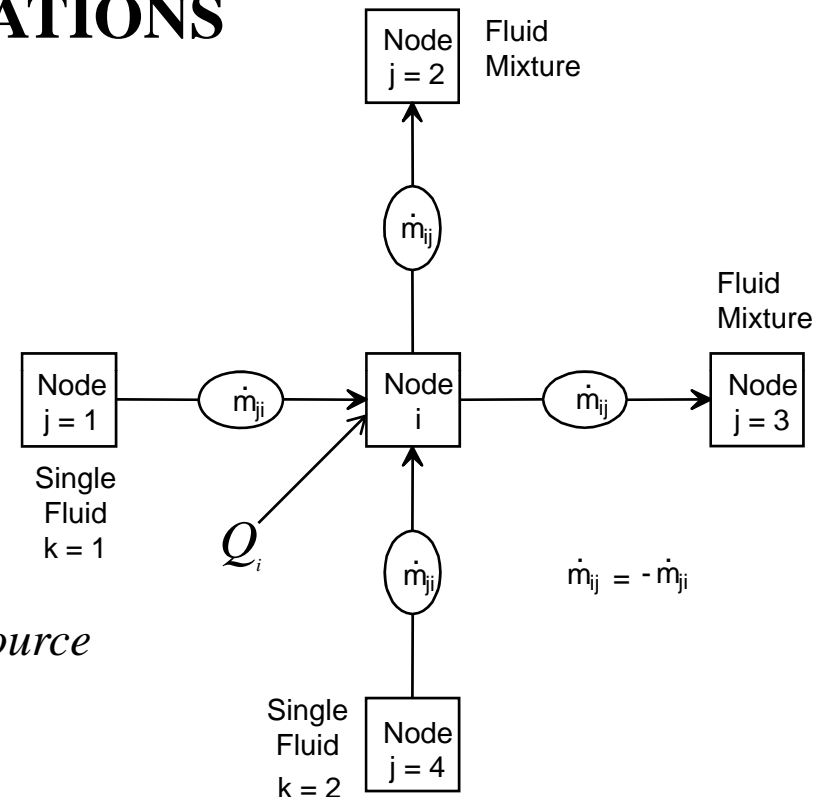
- Energy Conservation Equation can be written in Enthalpy or Entropy
- Based on Upwind Scheme

### Enthalpy Equation

*Rate of Increase of Internal Energy =*

*Enthalpy Inflow - Enthalpy Outflow + Heat Source*

$$\frac{m \left( h - \frac{p}{\rho J} \right)_{\tau + \Delta \tau} - m \left( h - \frac{p}{\rho J} \right)_{\tau}}{\Delta \tau} = \sum_{j=1}^{j=n} \left\{ \text{MAX} \left[ -\dot{m}_{ij}, 0 \right] h_j - \text{MAX} \left[ \dot{m}_{ij}, 0 \right] h_i \right\} + Q_i$$





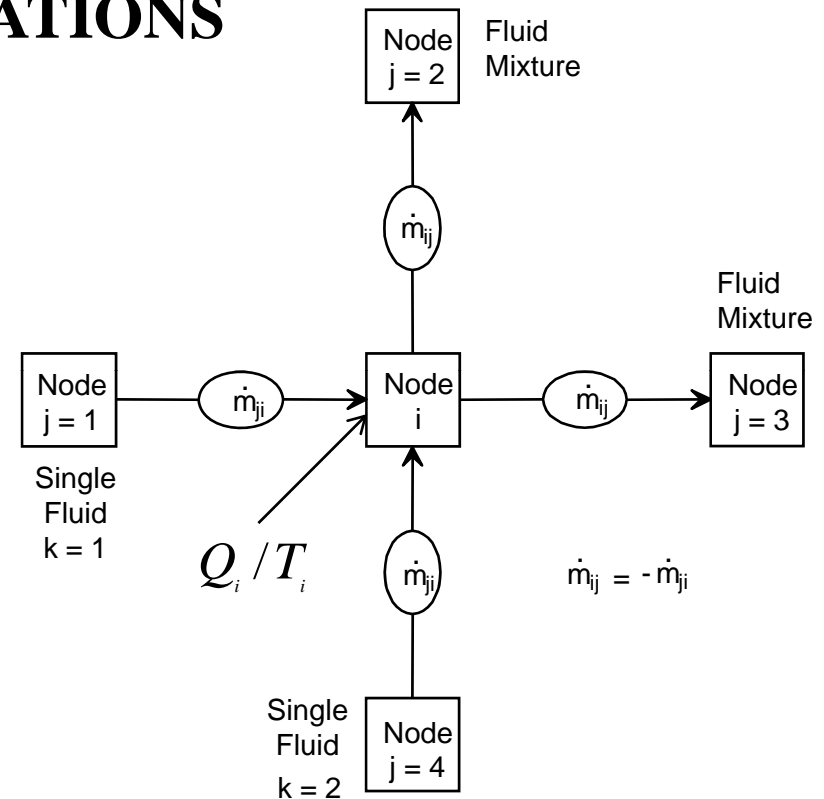


# GOVERNING EQUATIONS

## ENERGY CONSERVATION EQUATION

### Entropy Equation

*Rate of Increase of Entropy =*  
*Entropy Inflow - Entropy Outflow +*  
*Entropy Generation + Entropy Source*



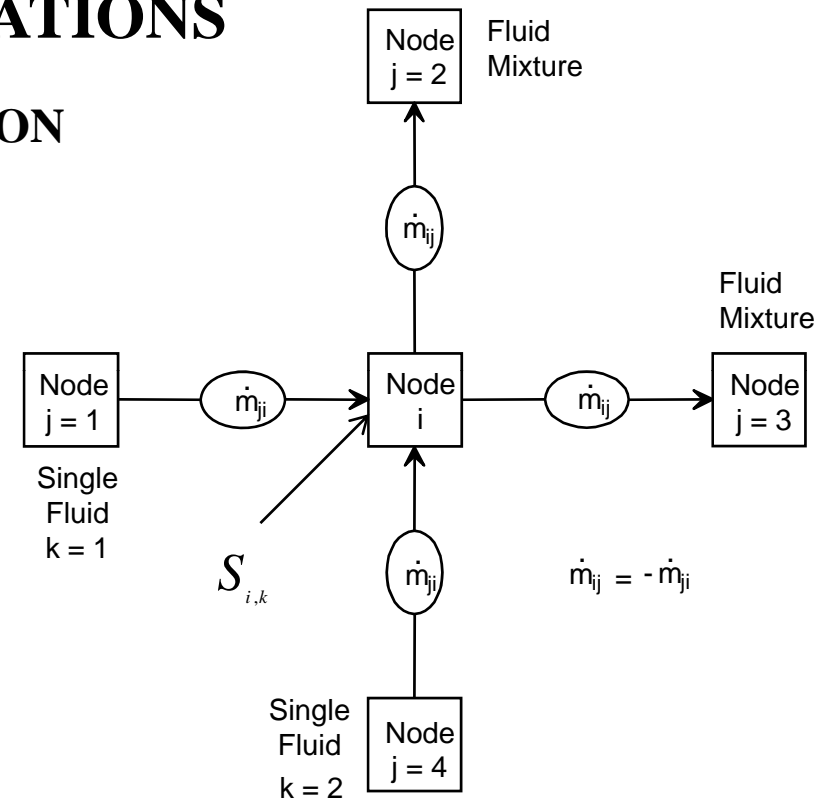
$$\frac{(ms)_{\tau + \Delta\tau} - (ms)_{\tau}}{\Delta\tau} = \sum_{j=1}^{j=n} \left\{ \text{MAX}[-\dot{m}_{ij}, 0] s_j - \text{MAX}[\dot{m}_{ij}, 0] s_i \right\} + \sum_{j=1}^{j=n} \left\{ \frac{\text{MAX}[-\dot{m}_{ij}, 0]}{|\dot{m}_{ij}|} \right\} \dot{S}_{ij, gen} + \frac{Q_i}{T_i}$$



# GOVERNING EQUATIONS

## FLUID SPECIE CONSERVATION EQUATION

*Rate of Increase of Fluid Specie =  
Fluid Specie Inflow - Fluid Specie Outflow +  
Fluid Specie Source*



$$\frac{(m_i c_{i,k})_{\tau + \Delta \tau} - (m_i c_{i,k})_{\tau}}{\Delta \tau} = \sum_{j=1}^{j=n} \left\{ \text{MAX} \left[ -\dot{m}_{ij}, 0 \right] c_{j,k} - \text{MAX} \left[ \dot{m}_{ij}, 0 \right] c_{i,k} \right\} + \dot{S}_{i,k}$$



# GOVERNING EQUATIONS

## EQUATION OF STATE

For unsteady flow, resident mass in a control volume is calculated from the equation of state for a real fluid

$$m = \frac{pV}{RT_z}$$

Z is the compressibility factor determined from higher order equation of state



# GOVERNING EQUATIONS

## EQUATION OF STATE

- GFSSP uses two separate Thermodynamic Property Packages  
GASP/WASP and GASPAK
- GASP/WASP uses modified Benedict, Webb & Rubin (BWR)  
Equation of State
- GASPAK uses “standard reference” equation from
  - National Institute of Standards and Technology (NIST)
  - International Union of Pure & Applied Chemistry (IUPAC)
  - National Standard Reference Data Service of the USSR



# GOVERNING EQUATIONS

## Mixture Property Relation

### Density

- Calculated from Equation of State of Mixture with Compressibility Factor

$$\rho_i = \frac{p_i}{z_i R_i T_i}$$

$$R_i = \sum_{k=1}^{k=n} x_k R_k$$

- Compressibility Factor of Mixture is Mole average of Individual Components

$$z_i = \sum_{k=1}^{k=n} x_k z_k$$

$$z_k = \frac{p_i}{\rho_k R_k T_k}$$



# GOVERNING EQUATIONS

## Mixture Property Relation

### Thermophysical Properties

- Viscosity, Specific Heat and Specific Heat Ratios are calculated

by taking Molar Average

$$\mu_i = \sum_{k=1}^{k=n} x_k \mu_k$$

$$\gamma_i = \sum_{k=1}^{k=n} x_k \gamma_k$$

$$C_{p,i} = \sum_{k=1}^{k=n} \frac{C_{p,k} x_k M_k}{x_k M_k}$$



# GOVERNING EQUATIONS

## Mixture Property Relation

### Temperature

- Mixture Temperature is calculated from Energy Conservation Equation

$$(T_i)_{\tau+\Delta\tau} = \frac{\sum_{j=1}^{j=n} \sum_{k=1}^{k=n_f} C_{p,k} x_k T_j \text{MAX}[-m_{ij}, 0] + (C_{p,i} m_i T_i)_{\tau} / \Delta\tau + Q_i}{\sum_{j=1}^{j=n} \sum_{k=1}^{k=n_f} C_{p,k} x_k \text{MAX}[m_{ij}, 0] + (C_{p,i} m)_{\tau} / \Delta\tau}$$

### Limitation

- Cannot handle phase change of mixture



# GOVERNING EQUATIONS

## Summary

- Familiarity with GFSSP's Governing Equations is not absolutely necessary to use the code
- However, working knowledge about Governing Equations is helpful to implement various options in a complex flow network
- A good understanding of Governing Equations is necessary to introduce new physics in the code





# SOLUTION PROCEDURE

- Successive Substitution
- Newton-Raphson
- Simultaneous Adjustment with Successive Substitution (SASS)
- Convergence



## SOLUTION PROCEDURE

- Non linear Algebraic Equations are solved by
  - Successive Substitution
  - Newton-Raphson
- GFSSP uses a Hybrid Method
  - SASS ( Simultaneous Adjustment with Successive Substitution)
  - This method is a combination of Successive Substitution and Newton-Raphson



# **SOLUTION PROCEDURE**

## **SUCCESSIVE SUBSTITUTION METHOD**

### *STEPS:*

- 1. Guess a solution for each variable in the system of equations**
- 2. Express each equation such that each variable is expressed in terms of other variables: e. g.  $X = f(Y, Z)$  and  $Y = f(X, Z)$  etc**
- 3. Solve for each variable**
- 4. Under-relax the variable, if necessary**
- 5. Repeat steps 1 through 4 until convergence**

### *ADVANTAGES:*

**Simple to program; takes less computer memory**

### *DISADVANTAGES:*

**It is difficult to make a decision in which order the equations must be solved to ensure convergence**



# **SOLUTION PROCEDURE**

## **NEWTON-RAPHSON METHOD**

### *STEPS:*

- 1. Guess a solution for each variable in the system of equations**
- 2. Calculate the residuals of each equation**
- 3. Develop a set of correction equations for all variables**
- 4. Solve for the correction equations by Gaussian Elimination method**
- 5. Apply correction to each variable**
- 6. Iterate until the corrections become very small**

### *ADVANTAGES:*

**No decision making process is involved to determine the order in which equations must be solved**

### *DISADVANTAGES:*

**Requires more computer memory; difficult to program.**



## **SOLUTION PROCEDURE**

### **SASS (Simultaneous Aadjustment with Successive Substitution) Scheme**

- **SASS is a combination of successive substitution and Newton-Raphson method**
- **Mass conservation and flowrate equations are solved by Newton-Raphson method**
- **Energy Conservation and concentration equations are solved by successive substitution method**
- **Underlying principle for making such division:**
  - **Equations which have strong influences to other equations are solved by the Newton-Raphson method**
  - **Equations which have less influence to other are solved by the successive substitution method**
- **This practice reduces code overhead while maintains superior convergence characteristics**

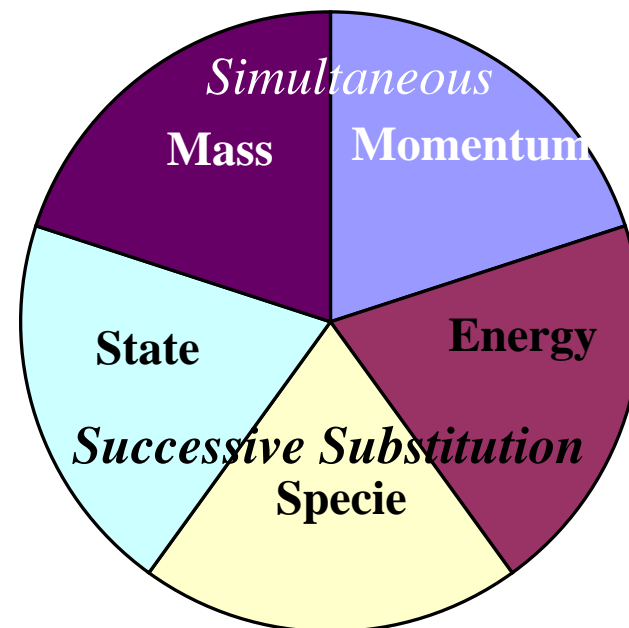


## GFSSP Solution Scheme

**SASS** : Simultaneous Adjustment  
with Successive Substitution

**Approach** : Solve simultaneously  
when equations are strongly  
coupled and non-linear

**Advantage** : Superior  
convergence characteristics with  
affordable computer memory





# CONVERGENCE

- Numerical solution can only be trusted when fully converged
- GFSSP's convergence criterion is based on difference in variable values between successive iterations. Normalized Residual Error is also monitored
- GFSSP's solution scheme has two options to control the iteration process
  - Simultaneous (SIMUL = TRUE)
  - Non-Simultaneous (SIMUL = FALSE)



# CONVERGENCE

## Simultaneous Option

- Single Iteration Loop
  - First solve mass, momentum and equation of state by the Newton-Raphson (NR) scheme
  - Next solve energy and specie conservation equation by Successive Substitution (SS) scheme
  - Solution is converged when the normalized maximum correction,  $\Delta_{\max}$  is less than the convergence criterion

$$\Delta_{\max} = \text{MAX} \left| \sum_{i=1}^{N_E} \frac{\Phi_i'}{\Phi_i} \right| \quad N_E \text{ is the total number of equations solved by the Newton-Raphson scheme}$$





# CONVERGENCE

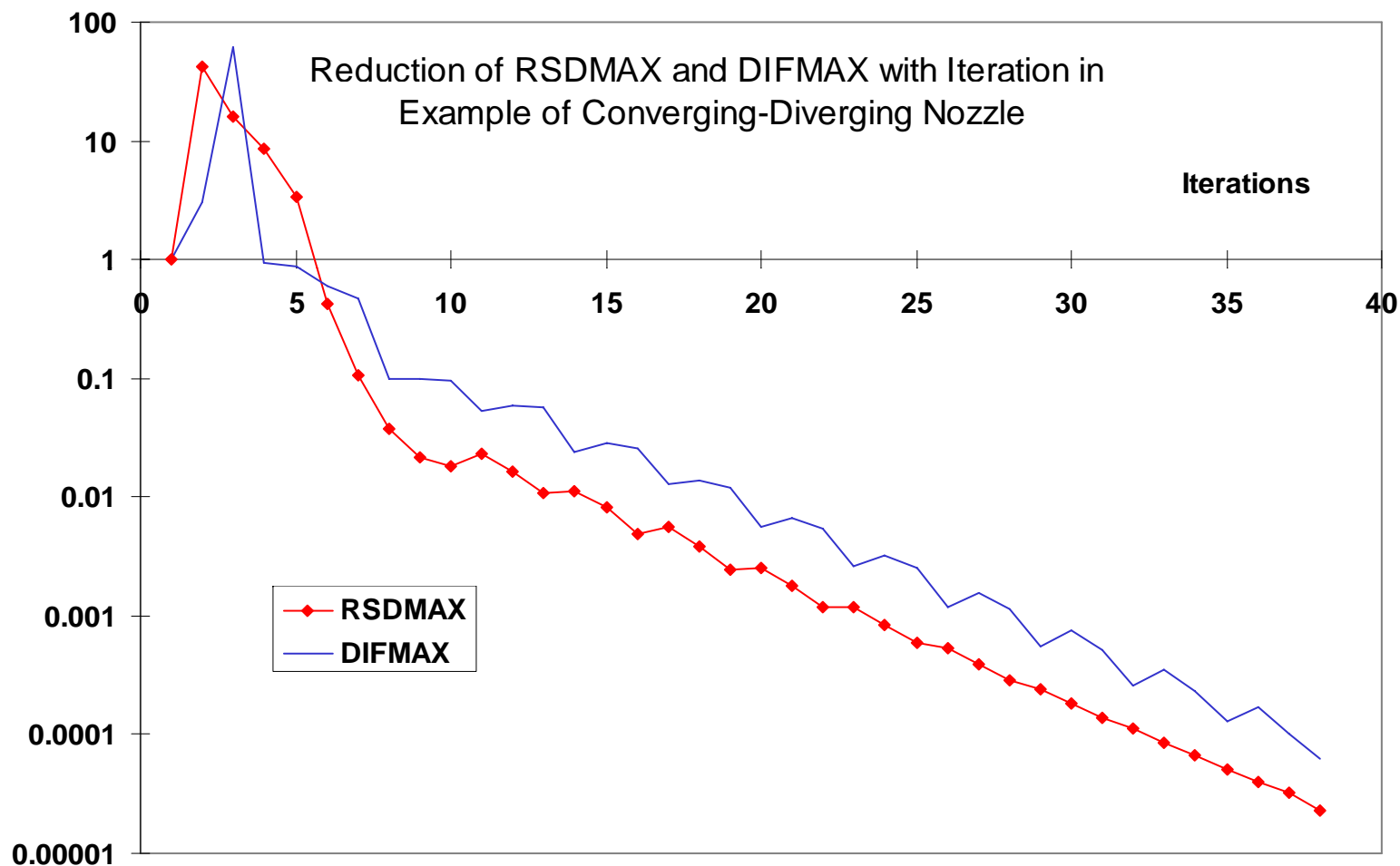
## Non-Simultaneous Option

- Inner & Outer Iteration Loop
  - Mass, Momentum and Equation of state is solved in inner iteration loop by NR scheme
  - Energy and Specie conservation equations are solved in outer iteration loop by SS scheme
  - Convergence of NR scheme is determined by  $\Delta_{\max}$
  - Convergence of SS scheme is determined by  $\Delta_{\max}$

$$\Delta_{\max}^o = MAX |\Delta_{K_f}, \Delta_{\rho}, \Delta_h \text{ or } \Delta_s| \quad \Delta_{K_f} = MAX \left| \sum_{i=1}^{N_B} \frac{K_f'}{K_f} \right| \quad \text{etc.}$$

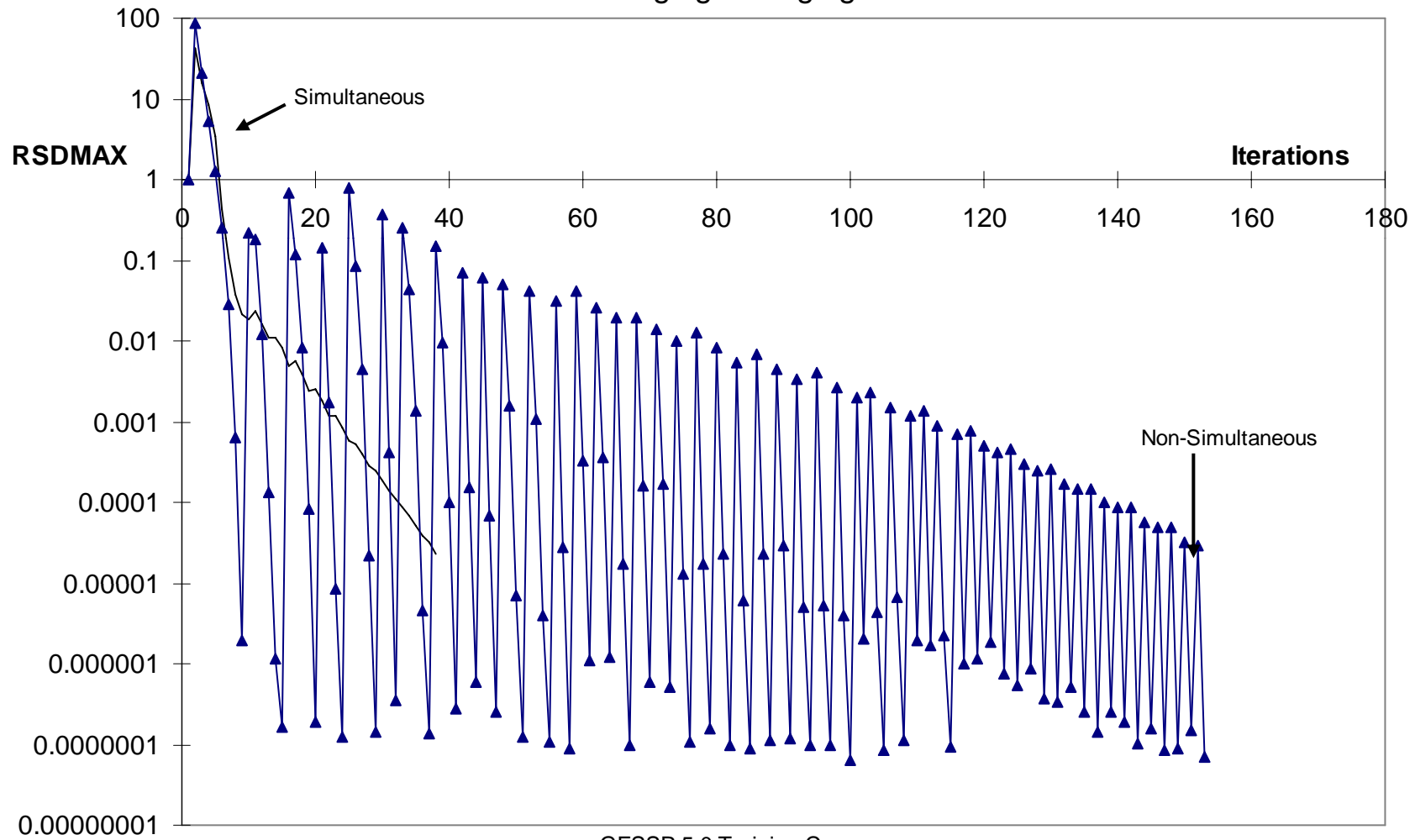


## Convergence Characteristics For Simultaneous Option





## Comparison of Convergence Characteristics between Simultaneous and Non-Simultaneous Option in Converging-Diverging Nozzle





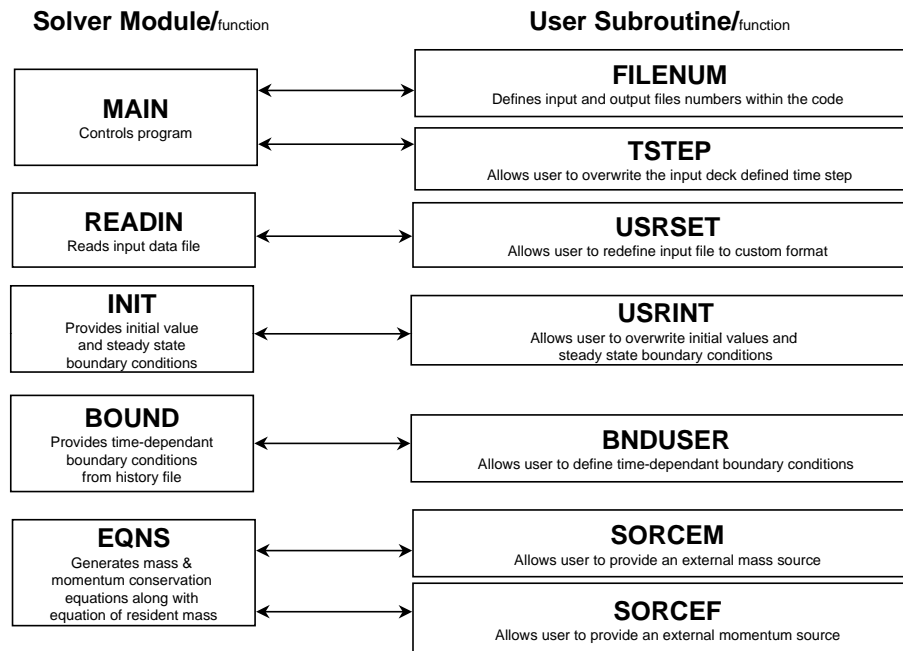
# SOLUTION PROCEDURE

## Summary

- Simultaneous option is more efficient than Non-Simultaneous option
- Non-Simultaneous option is recommended when Simultaneous option experiences numerical instability
- Under-relaxation and good initial guess also help to overcome convergence problem
- A lack of realism in problem specification can lead to convergence problem
- Lack of realism includes:
  - Unrealistic geometry and/or boundary conditions
  - Attempt to calculate properties beyond operating range



# USER SUBROUTINES



**Alok Majumdar**  
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*Marshall Space Flight Center  
GFSSP Training Course*



# CONTENT

- Motivation and Benefit
- How they work



# MOTIVATION AND BENEFIT

- Motivation: To allow users to access GFSSP solver module to develop additional modeling capability
- Benefit: GFSSP users can work independently without Developer's active involvement



## How do they work?

- A series of subroutines are called from various locations of solver module
- The subroutines do not have any code but includes the common block
- The users can write FORTRAN code to develop any new physical model in any particular node or branch

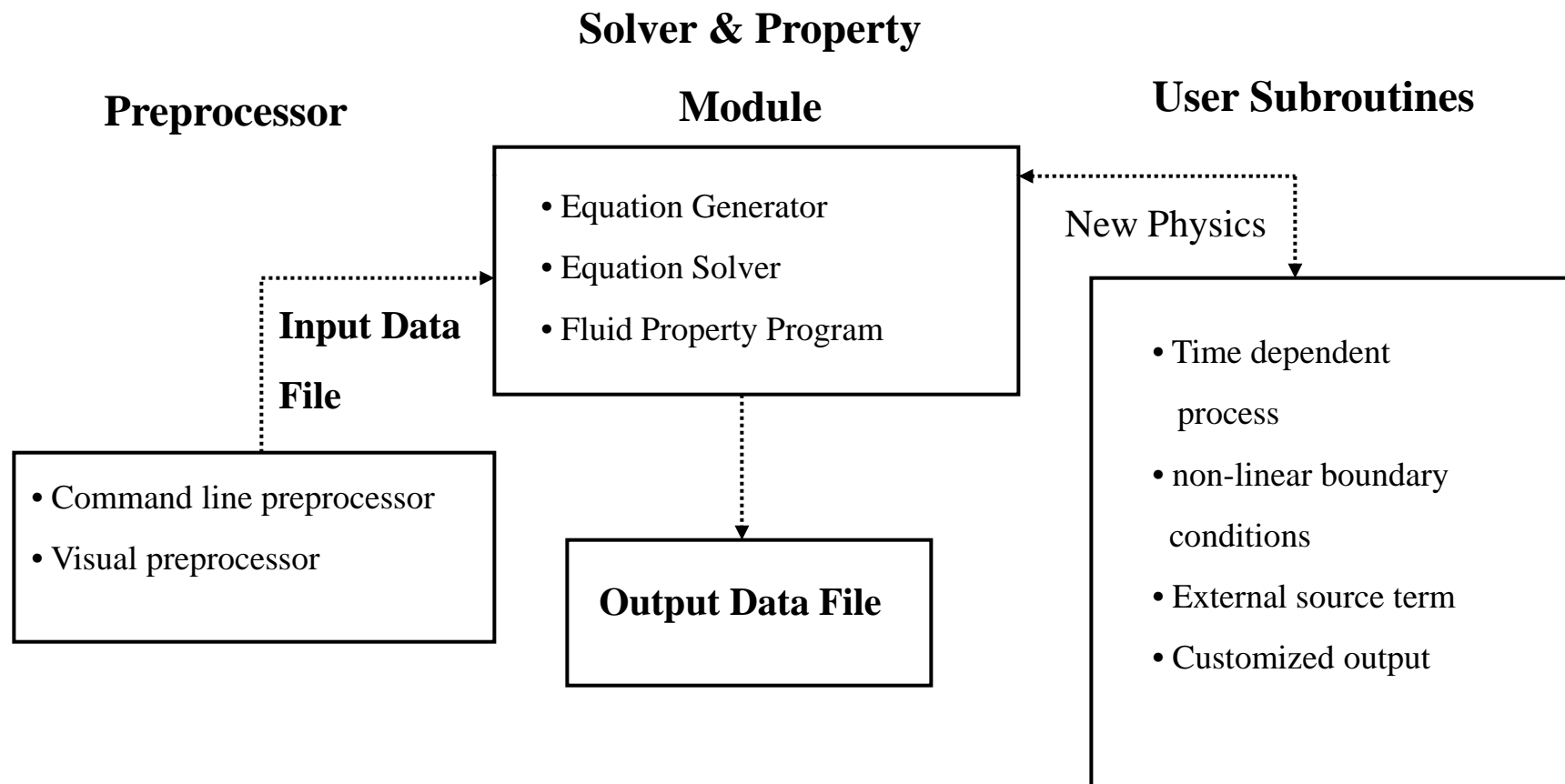
## What users need to do?

- Users need to compile a new file containing all user routines and link that with GFSSP to create a new executable





# GFSSP PROCESS FLOW DIAGRAM





# DESCRIPTION OF USER SUBROUTINES

Twelve User Subroutines were provided:

- SORCEM: External Mass Source
- SORCEF: External Force
- SORCEQ: External Heat source
- SORCEC: External Concentration source
- KFUSER: New resistance option
- PRPUSER: New fluid property
- TSTEP: Variable time step during a transient run



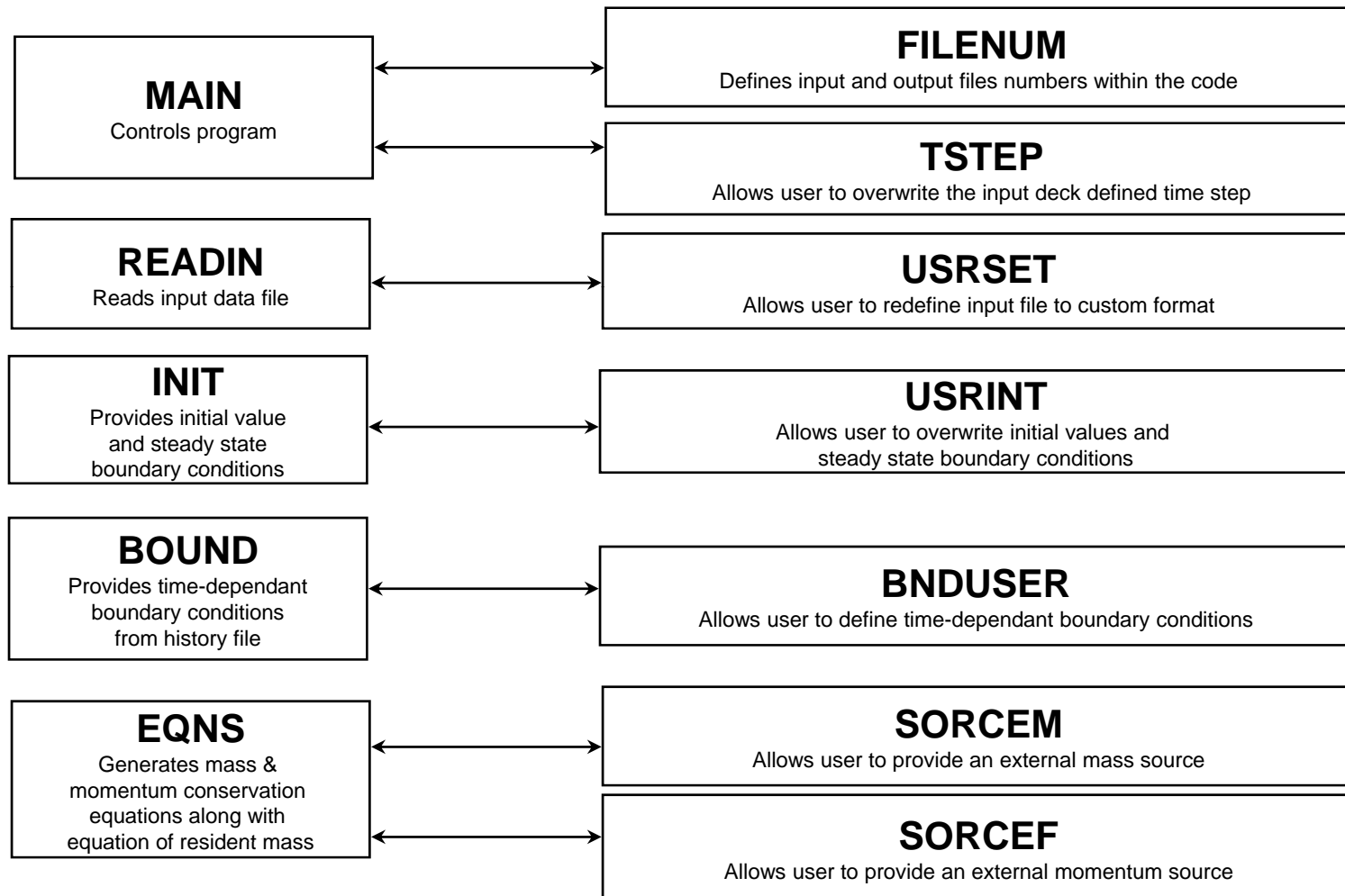
# DESCRIPTION OF USER SUBROUTINES

- BNDUSER: Variable boundary condition during transient run (Alternative to history file)
- USRINT: Provide initial values and steady state boundary conditions
- PRNUSER: Additional print out or creation of additional file for post processing
- FILNUM: Assign file numbers; users can define new file numbers
- USRSET: User can supply all the necessary information by writing their own code



## Solver Module/function

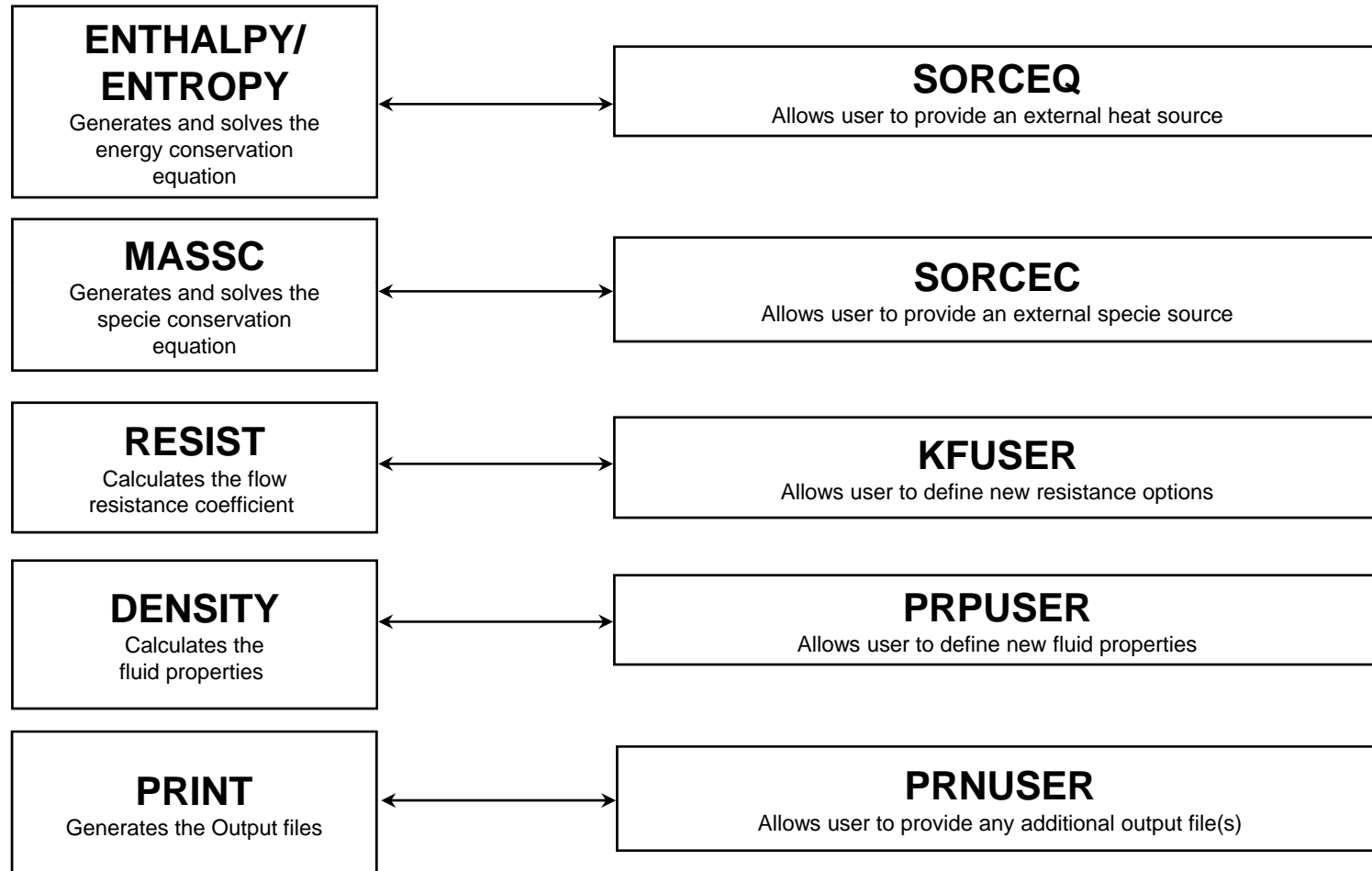
## User Subroutine/function





### Solver Module/function

### User Subroutine/function





# GFSSP INDEXING SYSTEM

- Node and Branch Variables are stored in one-dimensional array
- Node variables include:
  - Name
  - Pressure, Temperature, Concentration, Thermodynamic properties
- Branch variables include:
  - Name
  - Flowrate, Velocity, Resistance coefficients, Reynolds number
- Three subroutines are made available to Users for finding location and indices for a given node or branch



# NODE & BRANCH INDEX

- User defined node names are stored in NODE-array.
- NODE-array includes both internal and boundary nodes.
- Total number of elements in NODE-array is NNODES
- The internal nodes are stored in INODE-array.
- There are NINT elements in INODE-array.
- Branch names are stored in IBRANCH-array
- There are NBR elements in IBRANCH-array



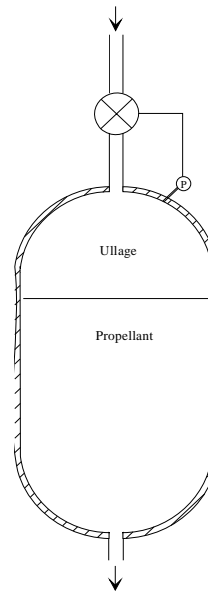
# SUMMARY

- User Subroutines can be used to add new capabilities that are not available to Users through Logical Options
- New capabilities may include:
  - Introducing new type of resistance
  - Incorporating heat or mass transfer in any given node
  - Variable time step for a transient problem
  - Customized output
- Checklist for User Subroutines
  - Identify subroutines that require modifications
  - Select GFSSP variables that require to be modified
  - Make use of GFSSP provided User variables in your coding





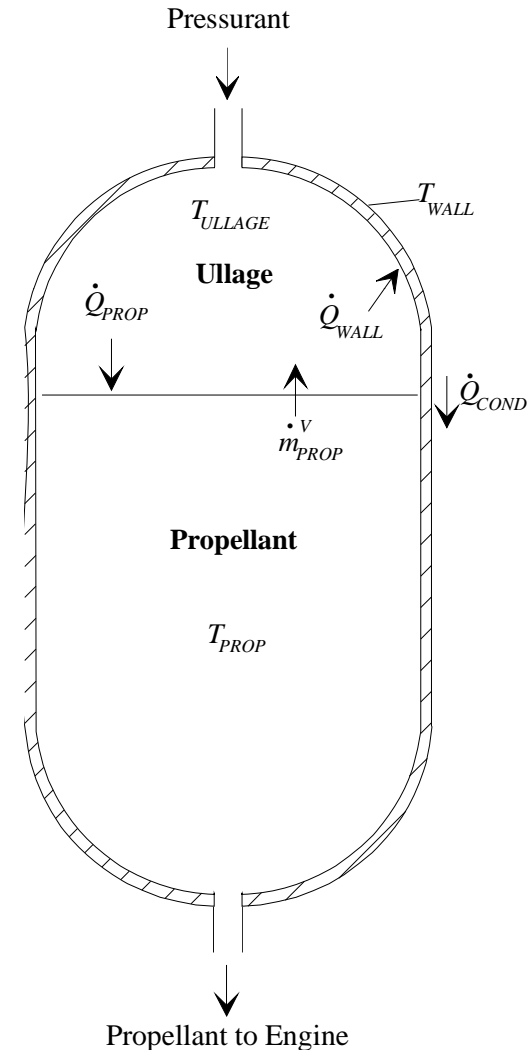
# TANK PRESSURIZATION





# TANK PRESSURIZATION

- Predict the ullage conditions considering heat and mass transfer between the propellant and the tank wall
- Predict the propellant conditions leaving the tank

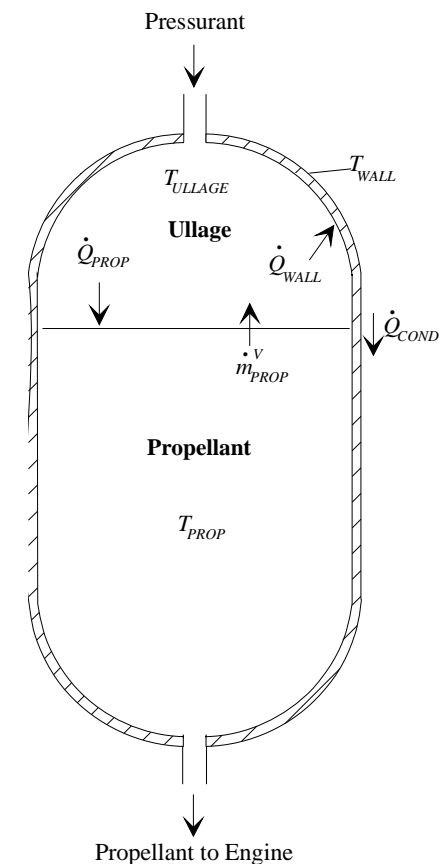




# TANK PRESSURIZATION

## ADDITIONAL PHYSICAL PROCESSES

- Change in ullage and propellant volume.
- Change in gravitational head in the tank.
- Heat transfer from pressurant to propellant.
- Heat transfer from pressurant to the tank wall.
- Heat conduction between the pressurant exposed tank surface and the propellant exposed tank surface.
- Mass transfer between the pressurant and propellant.

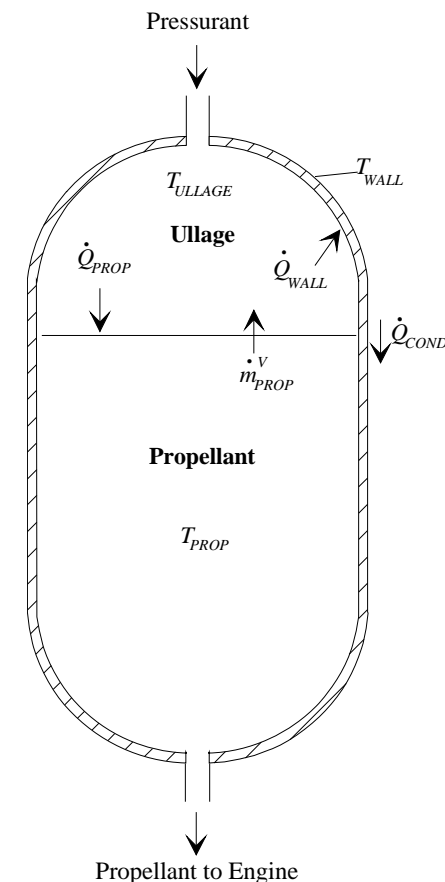




## TANK PRESSURIZATION CALCULATION STEPS

For each time step calculate

- Ullage and Propellant Volumes
- Tank Bottom Pressure
- Heat Transfer between pressurant and propellant and pressurant and wall
- Wall Temperature
- Mass Transfer from propellant to ullage

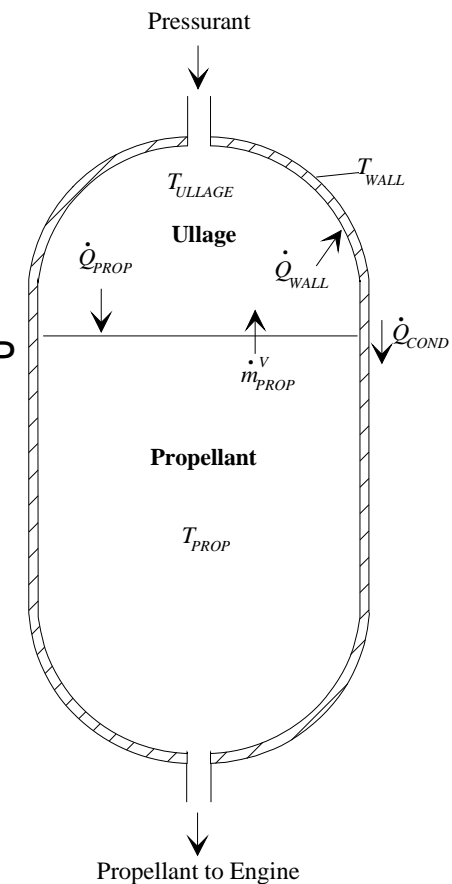




# TANK PRESSURIZATION

## ADDITIONAL INPUT DATA FOR PRESSURIZATION

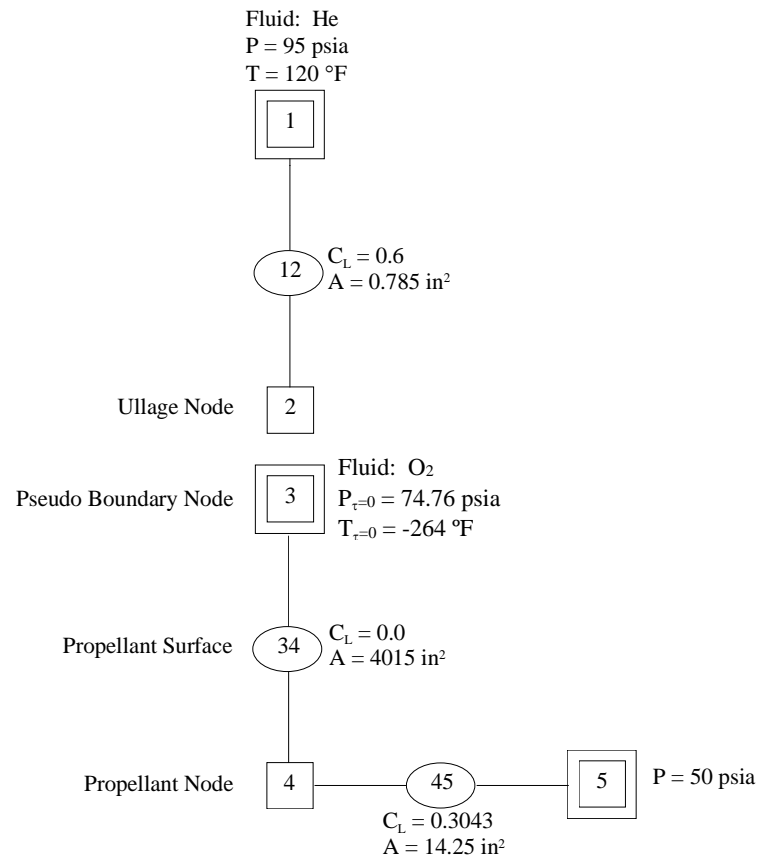
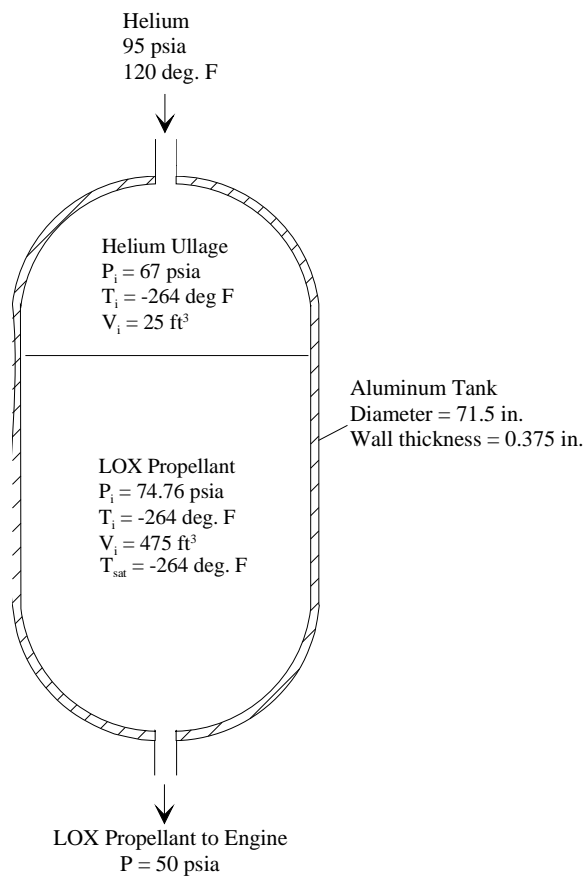
PRESS	Logical Variable to Activate the Option
NTANK	Number of Tanks in the Circuit
NODUL	Ullage Node
NODULB	Pseudo Boundary Node at interface
NODPRP	Propellant Node
IBRPRP	Branch number connecting NODULB & NODPRP
TNKAR	Tank Surface Area in Ullage at Start, in <sup>2</sup>
TNKTH	Tank Thickness, in
TNKRHO	Tank Density, lbm/ft <sup>3</sup>
TNKCP	Tank Specific Heat, Btu/lbm - R
TNKCON	Tank Thermal Conductivity, Btu/ft-sec-R
ARHC	Propellant Surface Area, in <sup>2</sup>
FCTHC	Multiplying Factor in Heat Transfer Coefficient
TNKTM	Initial Tank Temperature, °F





# TANK PRESSURIZATION

## EXAMPLE 10 TANK SCHEMATIC AND GFSSP MODEL





# TANK PRESSURIZATION

## EXAMPLE 10 PRESSURIZATION INPUT

NODE	PRES (PSI)	TEMP(DEGF)	MASS SOURC	HEAT SOURC	THRST AREA	VOLUME	CONCENTRATION
2	0.6700E+02	-0.2640E+03	0.0000E+00	0.0000E+00	0.0000E+00	0.4320E+05	1.0000 0.0000
4	0.7476E+02	-0.2640E+03	0.0000E+00	0.0000E+00	0.0000E+00	0.8208E+06	0.0000 1.0000

ex10h1.dat  
ex10h3.dat  
ex10h5.dat

•  
•  
•

NUMBER OF TANKS IN THE CIRCUIT

1

NODUL	NODULB	NODPRP	IBRPRP	TNKAR	TNKTH	TNKRHO	TNKCP	TNKCON	ARHC	FCTHC	TNKTM
2	3	4	34	6431.91	0.375	170.00	0.20	0.0362	4015.00	1.00	-264.00

NODE DATA FILE  
FNODE.DAT  
BRANCH DATA FILE  
FBRANCH.DAT

### Tank Input Units

VOLUME, in<sup>3</sup>  
TNKAR, in<sup>2</sup>  
TNKTH, in  
TNKRHO, lbm/ft<sup>3</sup>  
TNKCP, Btu/lbm-R  
TNKCON, Btu/ft-s-R  
ARHC, in<sup>2</sup>  
TNKTM, deg. F



# TANK PRESSURIZATION

## EXAMPLE 10 PRESSURIZATION OUTPUT

### SOLUTION

#### INTERNAL NODES

NODE	P (PSI)	TF (F)	Z	RHO (LBM/FT^3)	EM (LBM)	CONC HE	O2
2	0.9138E+02	-0.1347E+03	0.1006E+01	0.1047E+00	0.5144E+01	0.9690E+00	0.0310
4	0.9869E+02	-0.2640E+03	0.2310E-01	0.6514E+02	0.2937E+05	0.0000E+00	1.0000

### BRANCHES

BRANCH	KFACTOR (LBF-S^2/(LBM-FT)^2)	DELP (PSI)	FLOW RATE (LBM/SEC)	VELOCITY (FT/SEC)	REYN. NO.	MACH NO.	ENTROPY GEN. BTU/(R-SEC)	LOST WORK LBF-FT/SEC
12	0.238E+05	0.362E+01	0.148E+00	0.445E+03	0.156E+06	0.129E+00	0.281E-02	0.127E+04
34	0.000E+00	0.000E+00	0.163E+03	0.899E-01	0.412E+06	0.114E-03	0.000E+00	0.000E+00
45	0.263E+00	0.487E+02	0.163E+03	0.253E+02	0.690E+07	0.323E-01	0.115E+00	0.176E+05

NUMBER OF PRESSURIZATION SYSTEMS = 1

NODUL	NODPRP	QULPRP	QULWAL	QCOND	TNKTMT	VOLPROP	VOLULG
2	4	1.9642	8.5069	0.0022	196.4447	450.8641	49.1359

SOLUTION SATISFIED CONVERGENCE CRITERION OF 0.100E-02 IN 5 ITERATIONS  
TAU = 10.0000 ISTEP = 100

### Tank Output Units

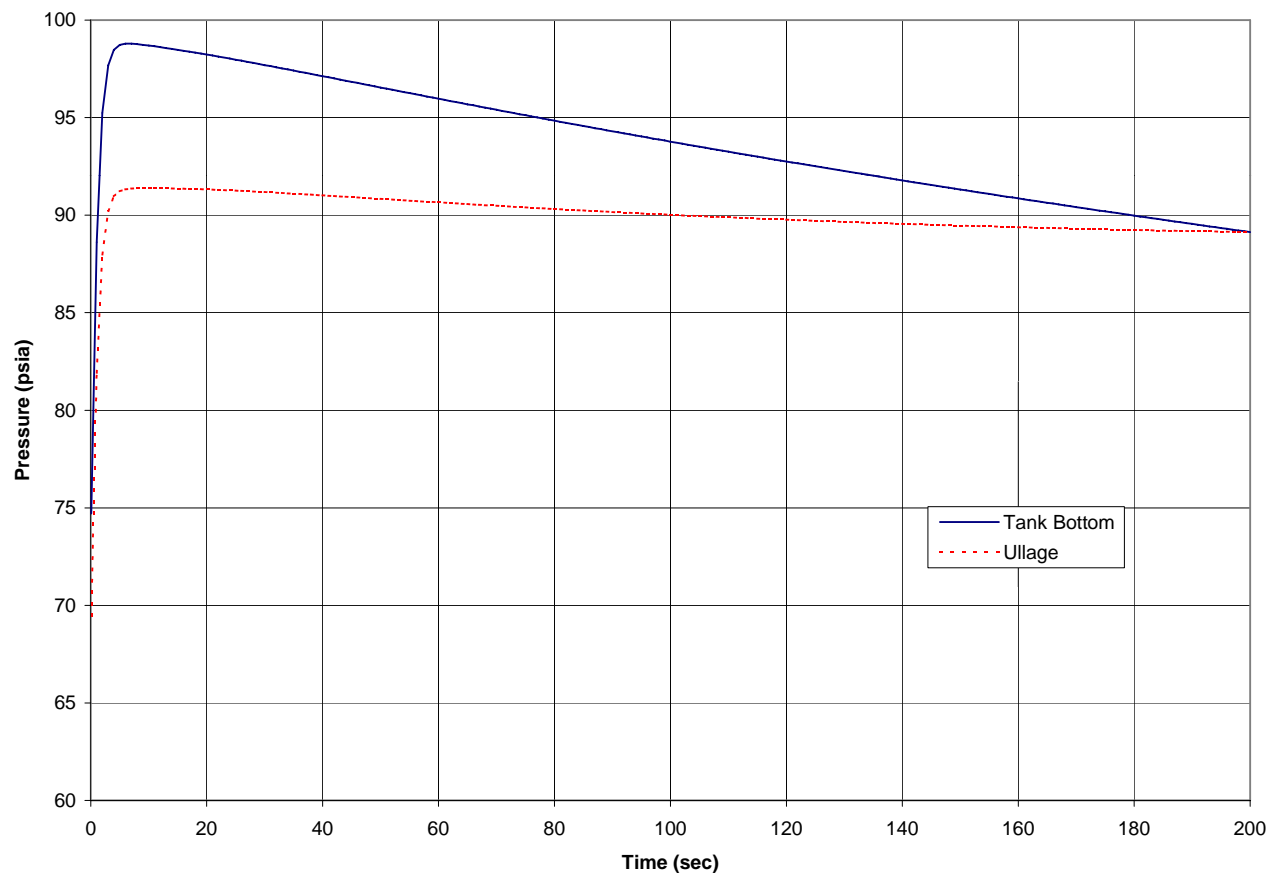
QULPROP, Btu/s  
QULWAL, Btu/s  
QCOND, Btu/s  
TNKTMT, deg. R  
VOLPROP, ft<sup>3</sup>  
VOLULG, ft<sup>3</sup>





# TANK PRESSURIZATION

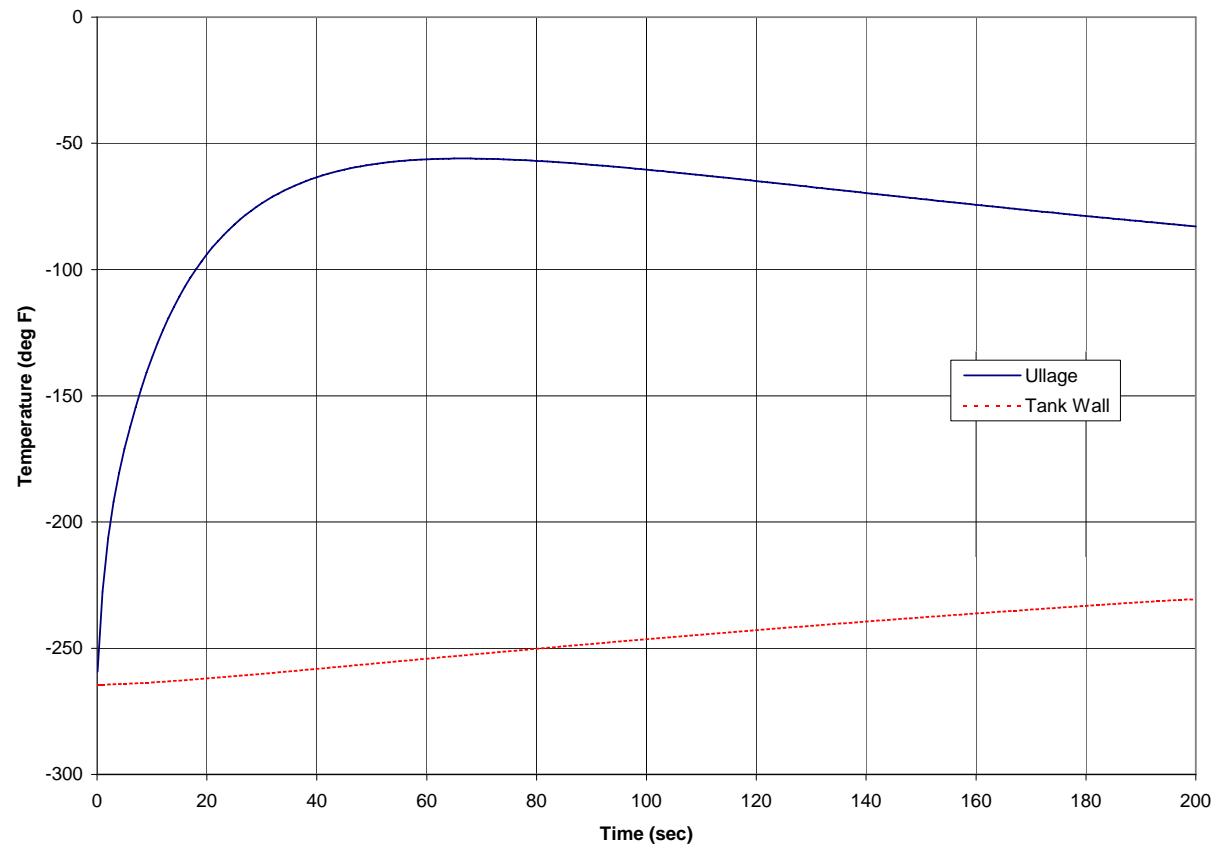
## EXAMPLE 10 ULLAGE AND TANK BOTTOM PRESSURE HISTORY





# TANK PRESSURIZATION

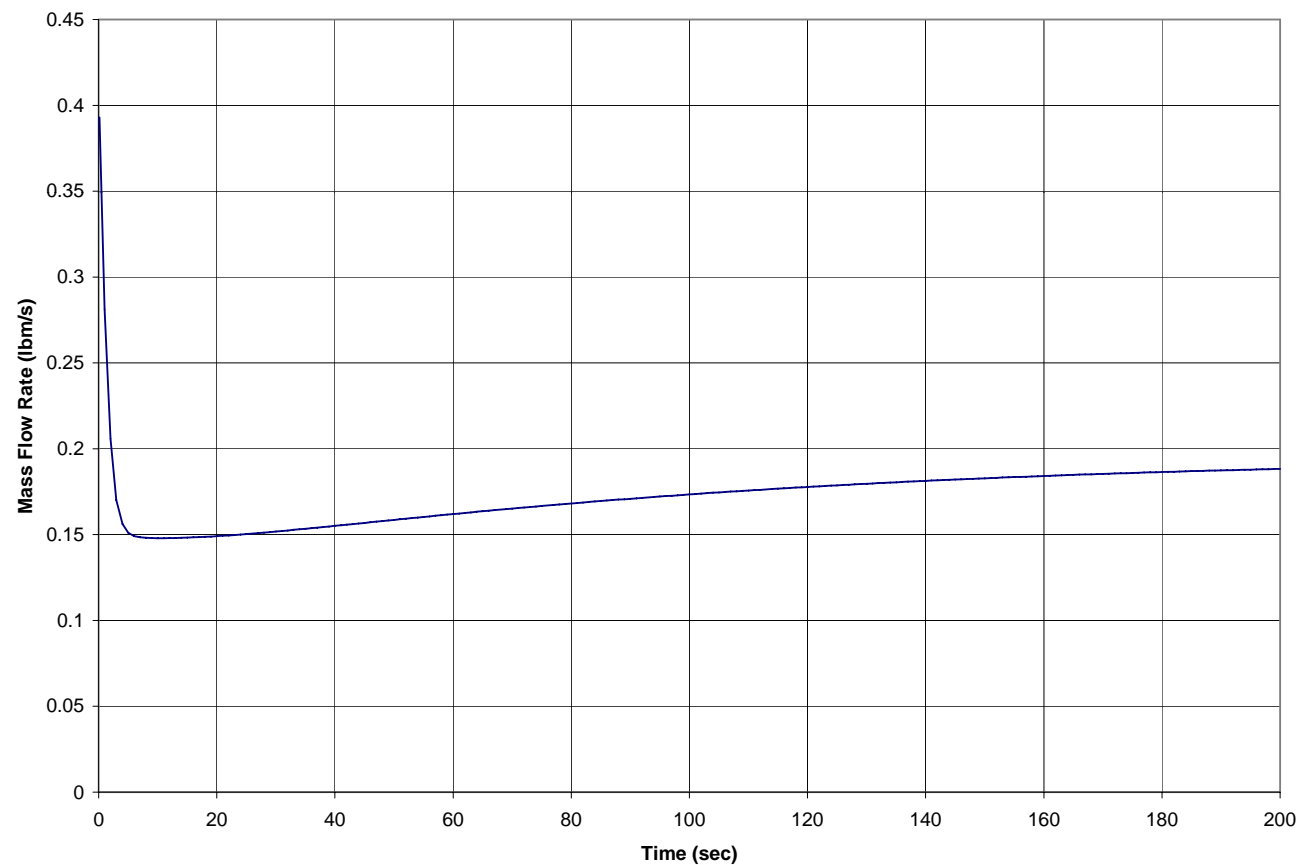
## EXAMPLE 10 ULLAGE AND TANK WALL TEMPERATURE HISTORY





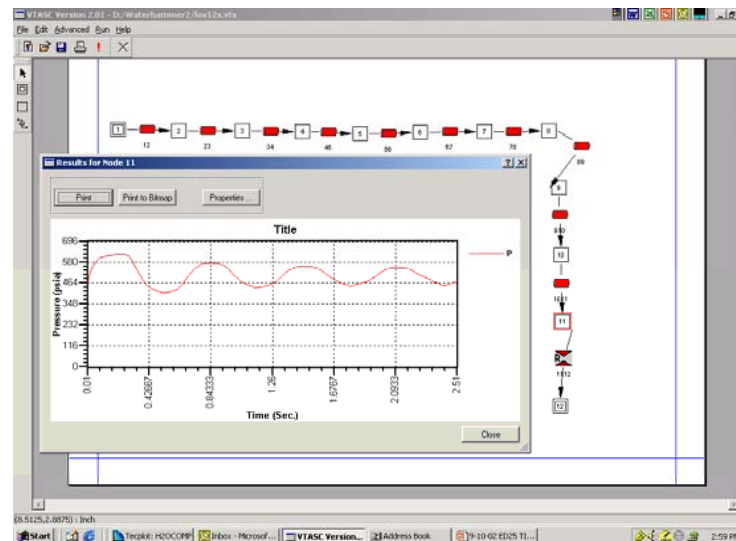
# TANK PRESSURIZATION

## EXAMPLE 10 HELIUM FLOW RATE HISTORY





# FLUID TRANSIENT





# CONTENT

- Classification of Unsteady Flow
- Causes of Transient
- Valve Closing
  - Comparison with Method of Characteristics
- Valve Opening
- Conclusions



## CLASSIFICATION OF UNSTEADY FLOW

- Quasi-steady flow is a type of unsteady flow when flow changes from one steady-state situation to another steady-state situation
  - Time dependant terms in conservation equation is not activated
  - Solution is time dependant because boundary condition is time dependant
- Unsteady flow formulation has time dependant terms in all conservation equations
  - Time dependant term is a function of density, volume and variables at previous time step



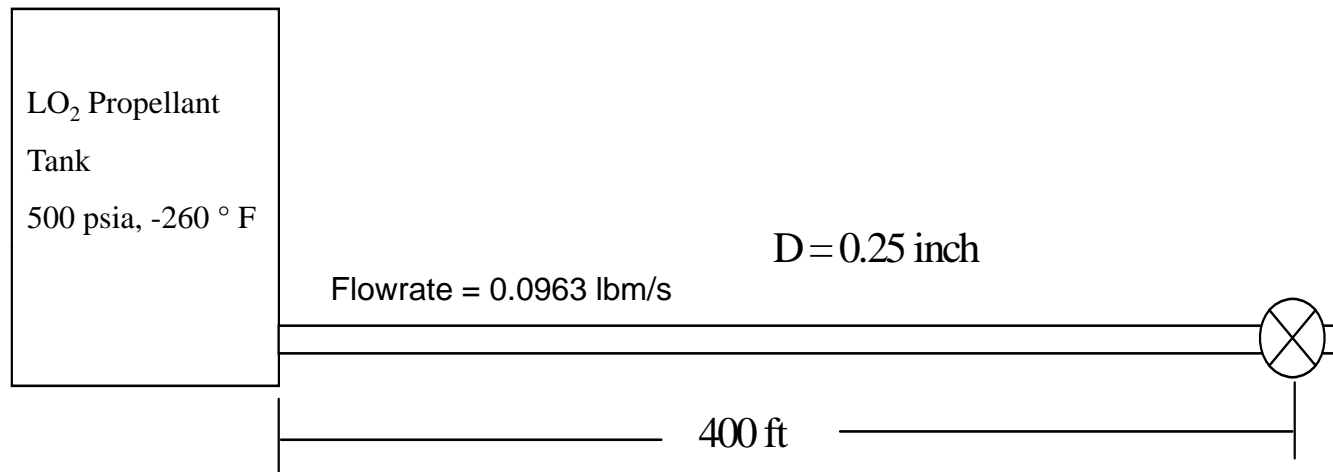
## CAUSES OF TRANSIENT

- Changes in valve settings, accidental or planned
- Starting or stopping of pumps
- Changes in power demand of turbines
- Action of reciprocating pumps
- Changing elevation of reservoir
- Waves in reservoir
- Vibration of impellers or guide vanes in pumps or turbines
- Unstable pump characteristics
- Condensation



## PROBLEM DESCRIPTION

### Rapid valve closing



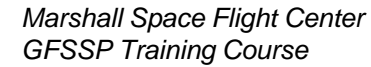
#### Valve Closure History

Time (sec)	Area (in <sup>2</sup> )
0.00	0.0491
0.02	0.0164
0.04	0.0055
0.06	0.0018
0.08	0.0006
0.10	0.00

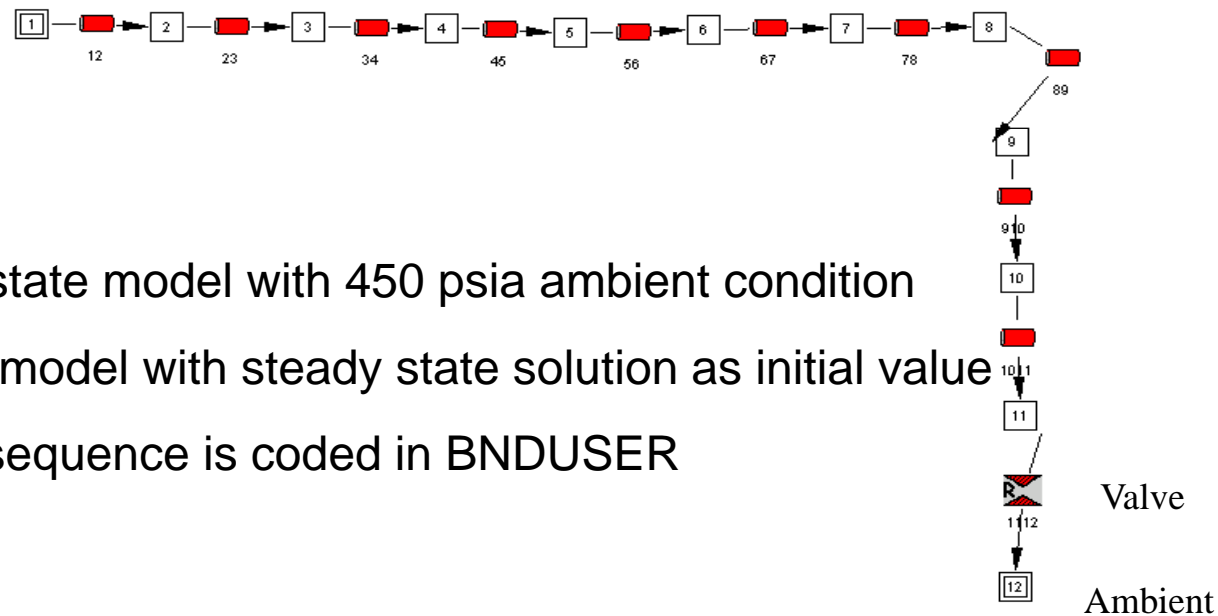
#### Objectives of Analysis:

- Maximum Pressure
- Frequency of Oscillation





500 psia, -260 ° F



- Run a steady state model with 450 psia ambient condition
- Run unsteady model with steady state solution as initial value
- Valve closing sequence is coded in BNDUSER



## Description of Test cases

$$P_{\text{tank}} = 500 \text{ psia}$$
$$T_{\text{tank}} = -260 \text{ }^{\circ}\text{F (Oxygen)} = 70 \text{ }^{\circ}\text{F (Water)} = -414 \text{ }^{\circ}\text{F (Hydrogen)}$$

Case No.	Fluid	Number of Branches	Time Step (sec)	Sound Speed (ft/sec)	Flowrate (lb/sec)	$p_{\text{max}}$ (psia)	Period of Oscillation (sec)
1	LO <sub>2</sub>	10	0.01	2462	0.0963	626	0.65
2	LO <sub>2</sub>	20	0.005	2462	0.0963	632	0.65
3	LO <sub>2</sub>	5	0.02	2462	0.0966	620	0.65
4	H <sub>2</sub> O	10	0.005	4874	0.071	704	0.33
5	LH <sub>2</sub>	10	0.02	3577	0.0278	545	0.43
6	LO <sub>2</sub> & GHe (0.1%)	10	0.01	1290**	0.0963	580	1.24
7	LO <sub>2</sub> & GHe (0.5%)	10	0.01	769**	0.0963	520	2.08
8*	LO <sub>2</sub> (2 Phase) $x_{\text{exit}} = 0.017$	10	0.01	-----	0.0963	550	1.17
9*	LO <sub>2</sub> (2 Phase) $x_{\text{exit}} = 0.032$	10	0.01	-----	0.0963	538	1.22
10	LO <sub>2</sub>		0.01	2462	0.0963	611	0.65

\* Pressure oscillations are due to condensation

\*\* Estimated from period of oscillation [ $a=4L/\lambda$ ]

- Time step for each test case is so chosen that Courant number is less than unity
- Courant Number = Length of Branch/(Sound Speed X Time Step)

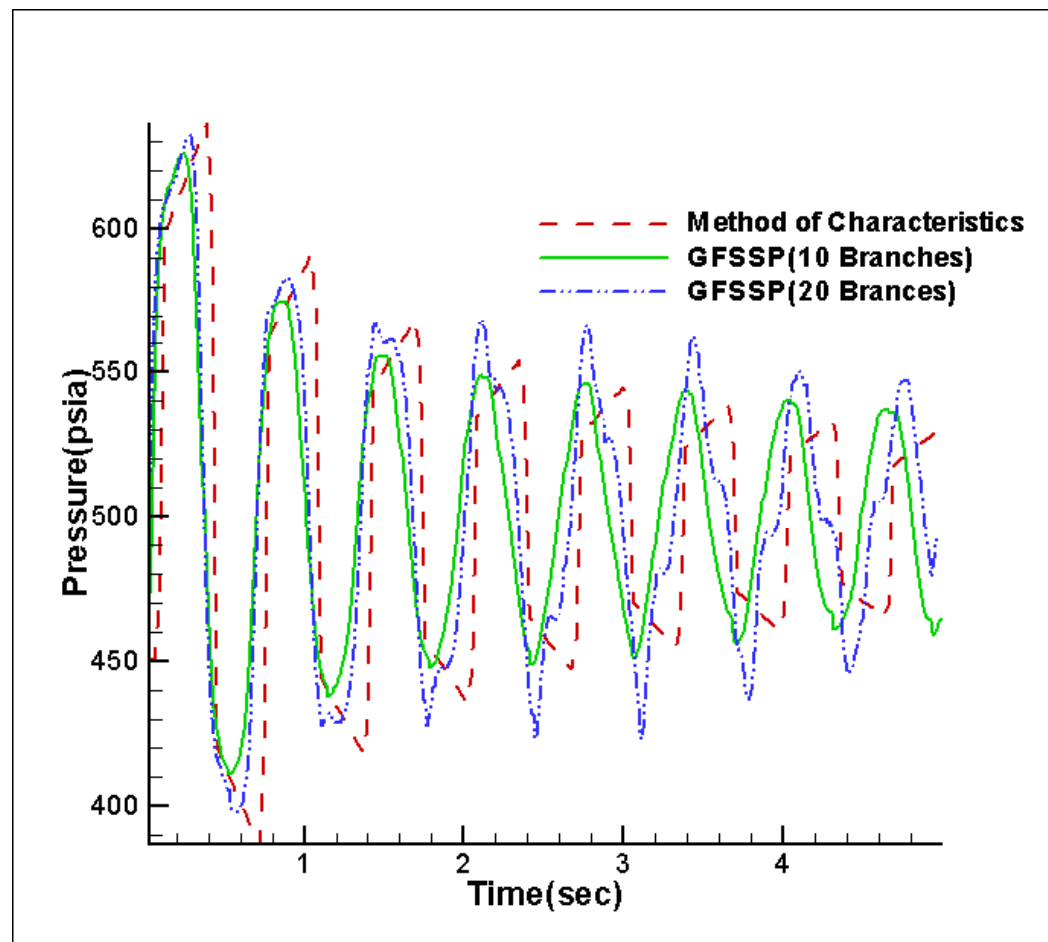


## COMPARISON BETWEEN GFSSP & MOC SOLUTION FOR THREE FLUIDS

Fluid	Flowrate (lb/s)	Velocity (ft/s)	Friction Factor (Used in MOC solution)	Sound Speed (ft/s)	Max. Pressure rise above supply pressure (psi)		Period of Oscillation (sec)	
					MOC	GFSSP	MOC	GFSSP
Water	0.071	3.34	0.0347	4892	214	204	0.33	0.33
Oxygen	0.0963	4.35	0.0196	2455	136	126	0.65	0.65
Hydrogen	0.0278	19.01	0.0157	3725	61	45	0.43	0.43



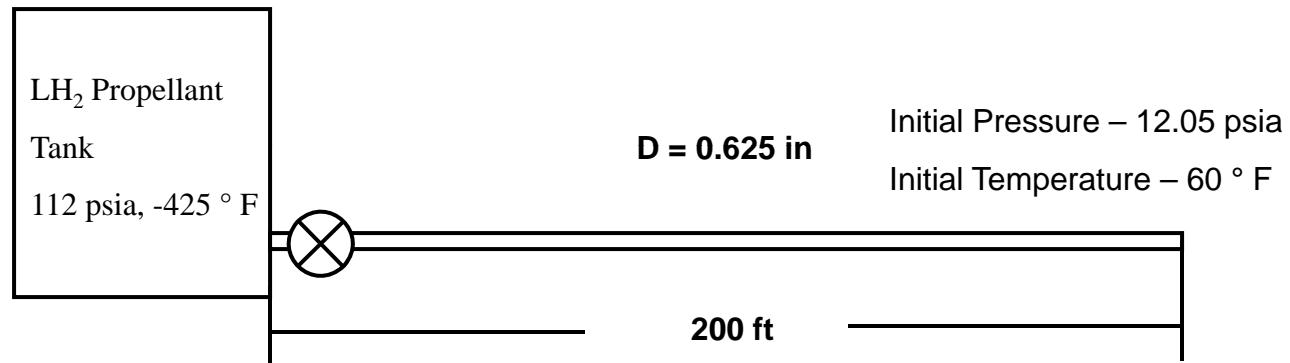
# COMPARISON BETWEEN GFSSP & MOC SOLUTION





# PROBLEM DESCRIPTION

## Rapid Valve Opening



### Objectives of Analysis:

- Maximum Pressure
- Time to reach steady-state

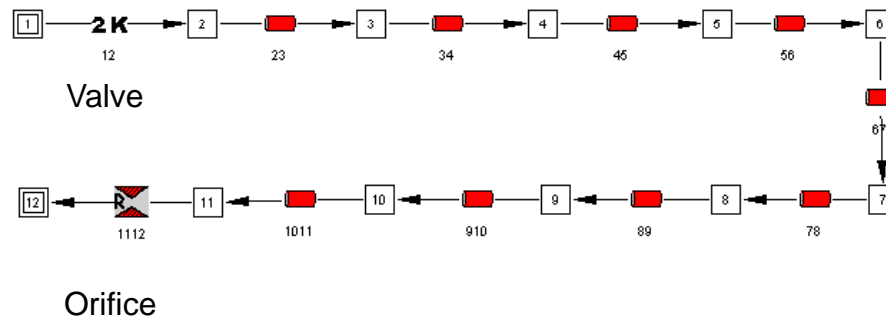
Time (sec)	Area (in <sup>2</sup> )
0.00	0.0
0.01	0.0088
0.02	0.1767
0.03	0.2651
0.04	0.3534
0.05	0.4418



## GFSSP Model

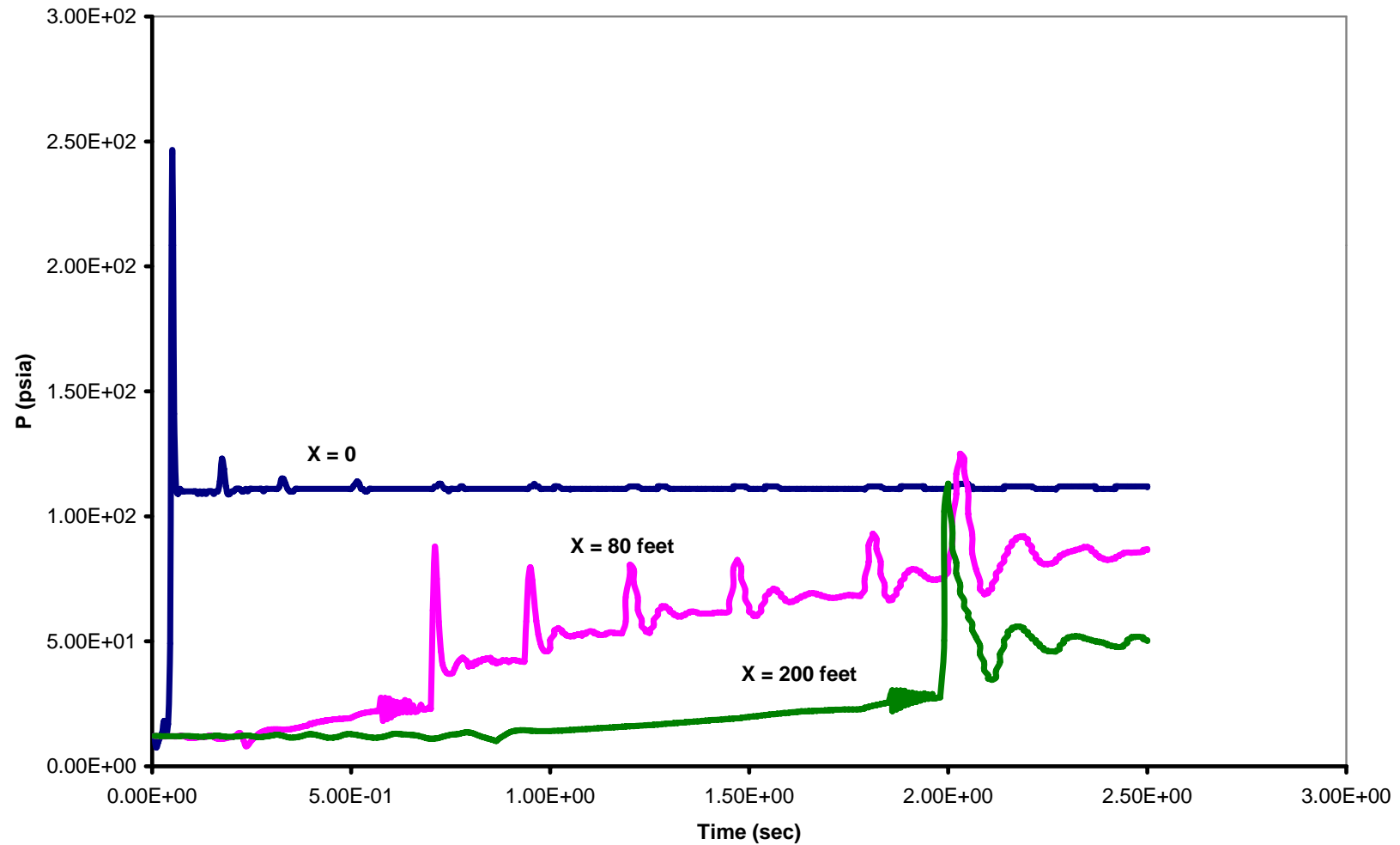
LH<sub>2</sub> Propellant Tank

112 psia, -425 ° F



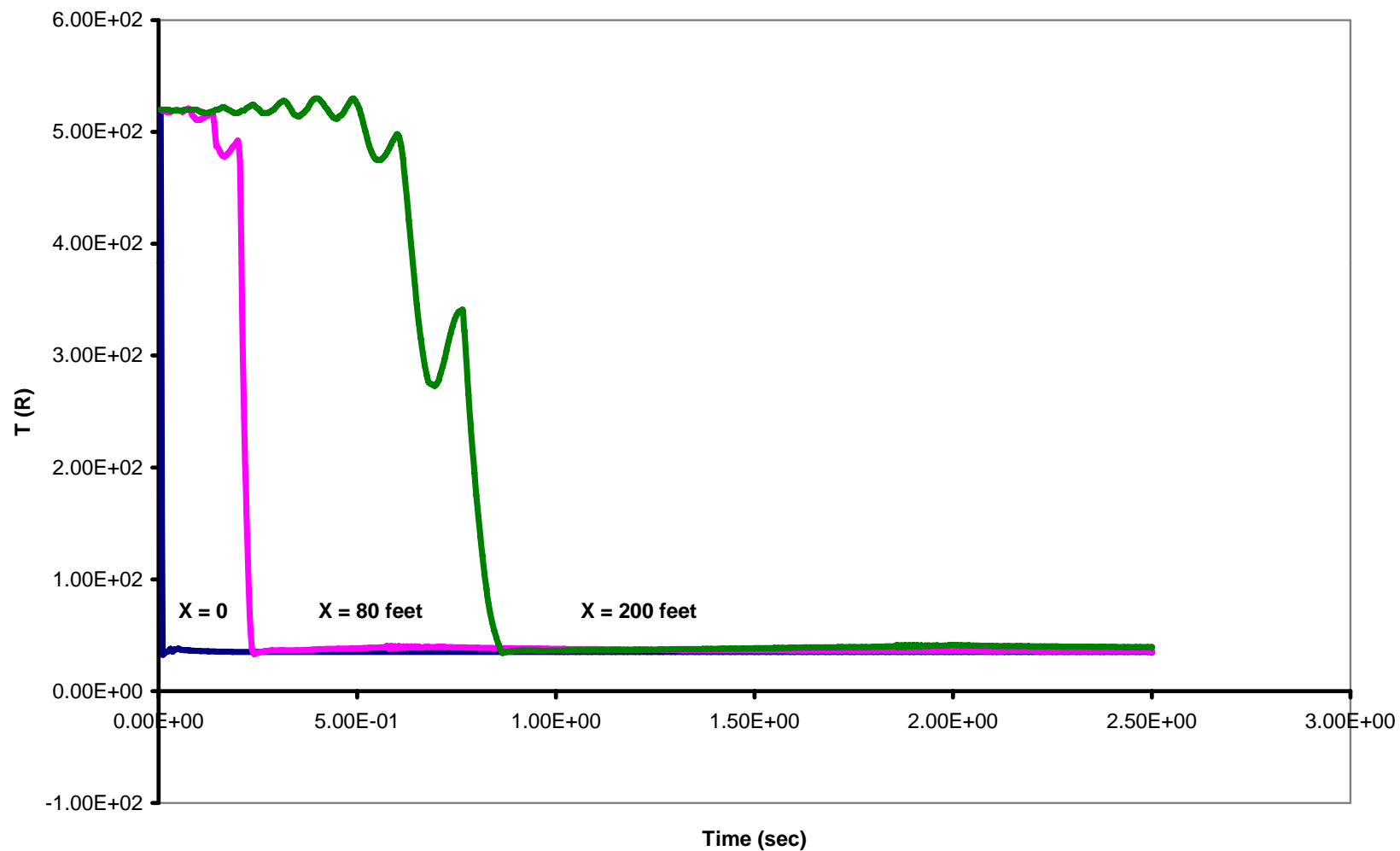


## Pressure





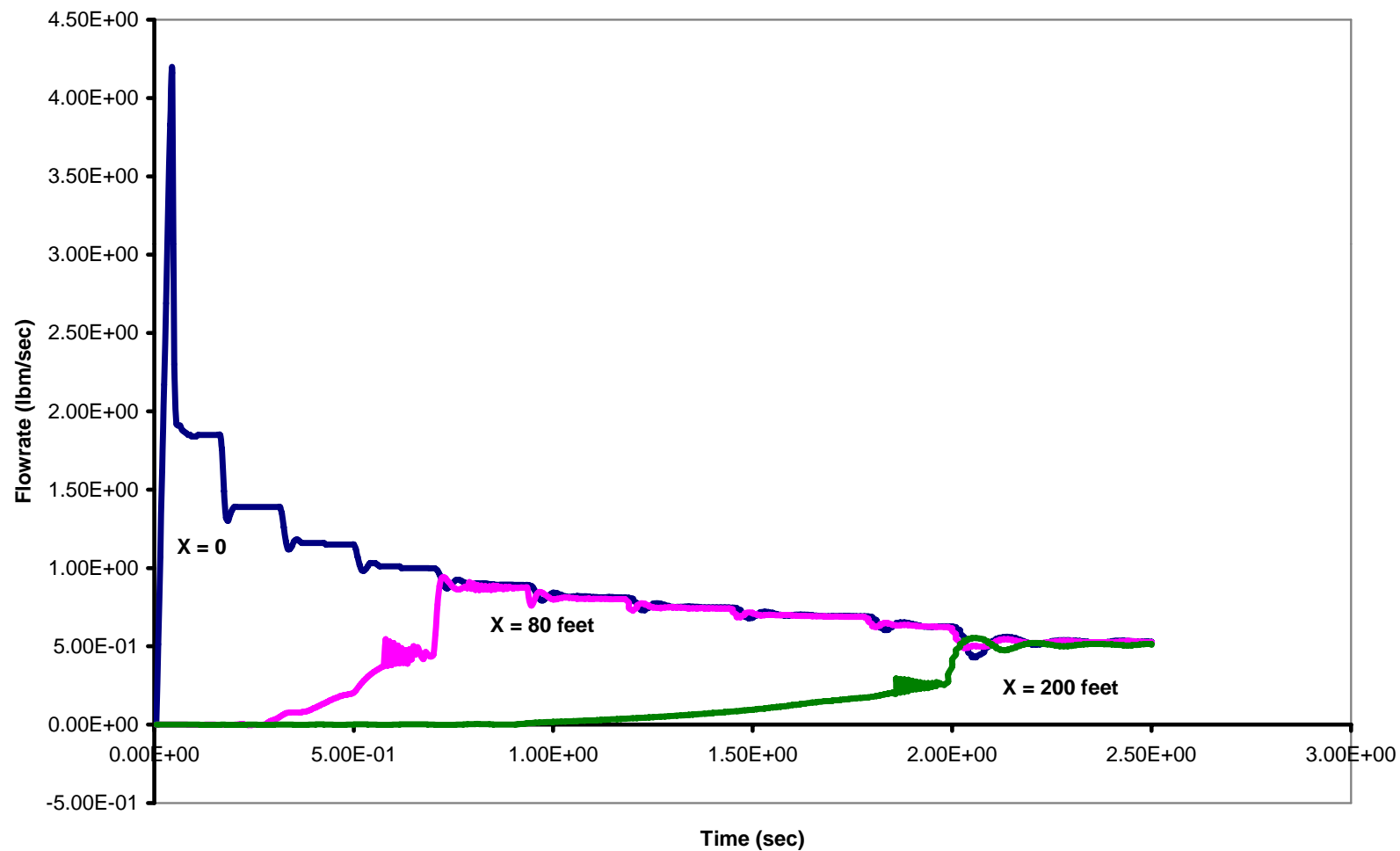
## Temperature





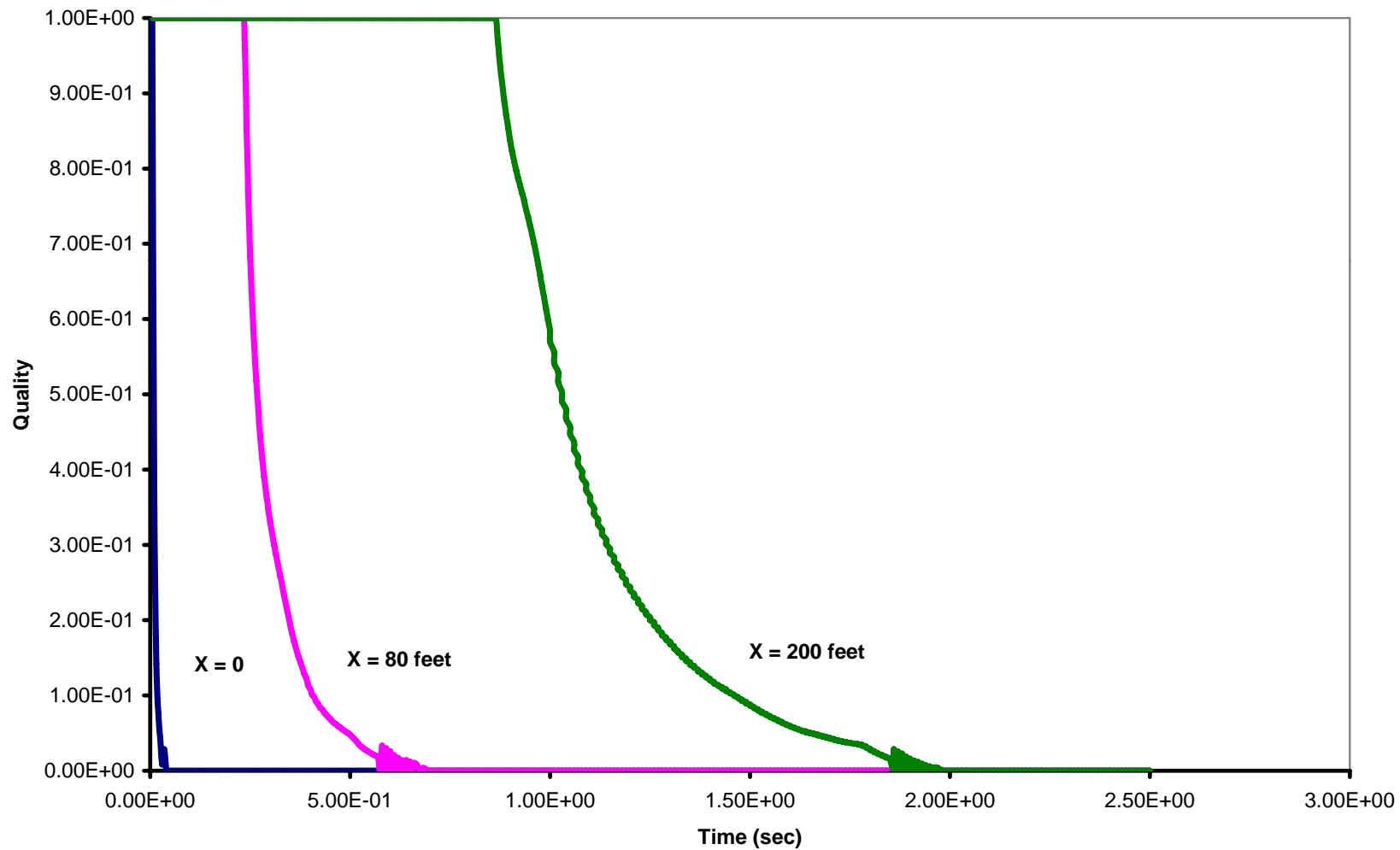


### Flowrate





### Quality





## CONCLUSIONS

- GFSSP has been used to compute fluid transient following rapid valve closure and opening
- GFSSP predictions have been compared with MOC solution:
  - Maximum pressure and frequency compares well
  - Discrepancies exist in damping rate and shape of the pressure curve
- Demonstrations have been made for
  - Sudden opening of cryogenic propellant in long pipeline
- Time step must satisfy Courant condition
- Predictions in all demonstration calculations show physical realism

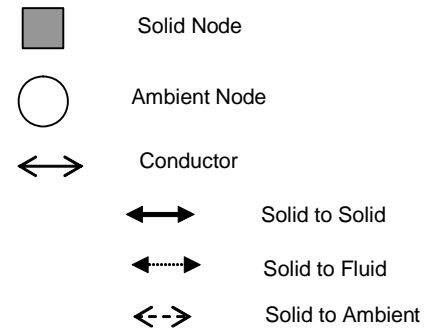
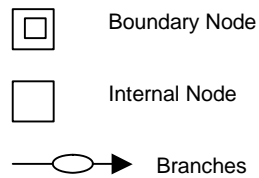
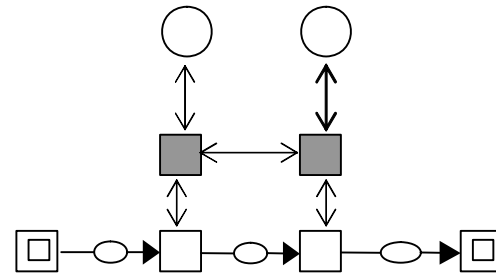


# Conjugate Heat Transfer



# Conjugate Heat Transfer

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# Mathematical Closure

## Unknown Variables

1. Pressure
2. Flowrate
3. Fluid Temperature
4. Solid Temperature
5. Specie Concentrations
6. Mass

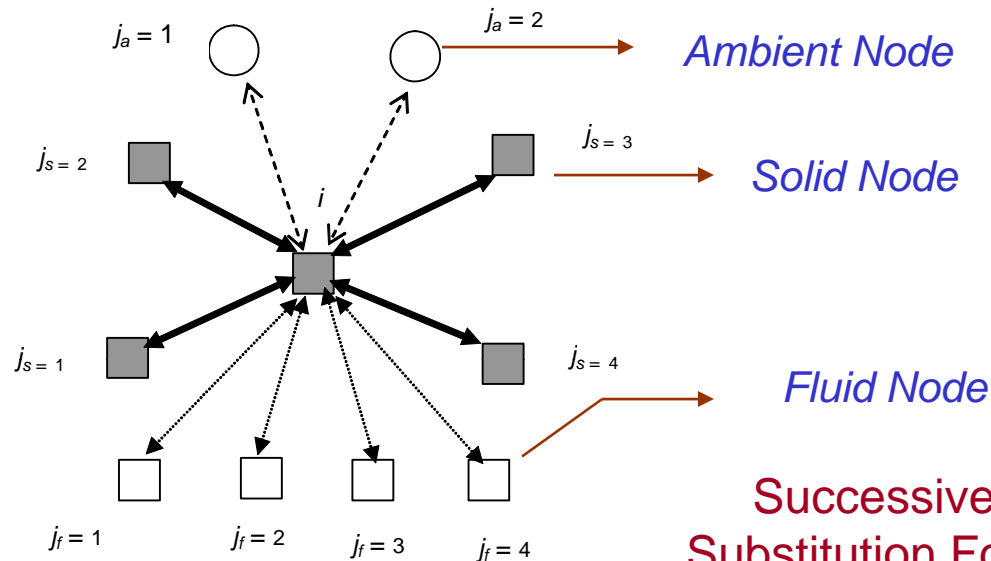
## Available Equations to Solve

1. Mass Conservation Equation
2. Momentum Conservation Equation
3. Energy Conservation Equation of Fluid
4. Energy Conservation Equation of Solid
5. Conservation Equations for Mass Fraction of Species
6. Thermodynamic Equation of State



# Heat Conduction Equation

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Conservation  
Equation

Successive  
Substitution Form

$$\frac{\partial}{\partial \tau} (m C_p T_s^i) = \sum_{j_s=1}^{n_{ss}} \dot{q}_{ss} + \sum_{j_f=1}^{n_{sf}} \dot{q}_{sf} + \sum_{j_a=1}^{n_{sa}} \dot{q}_{sa} + \dot{S}_i$$

$$\dot{q}_{ss} = k_{ij_s} A_{ij_s} / \delta_{ij_s} (T_s^{j_s} - T_s^i)$$

$$\dot{q}_{sf} = h_{ij_f} A_{ij_f} (T_f^{j_f} - T_s^i)$$

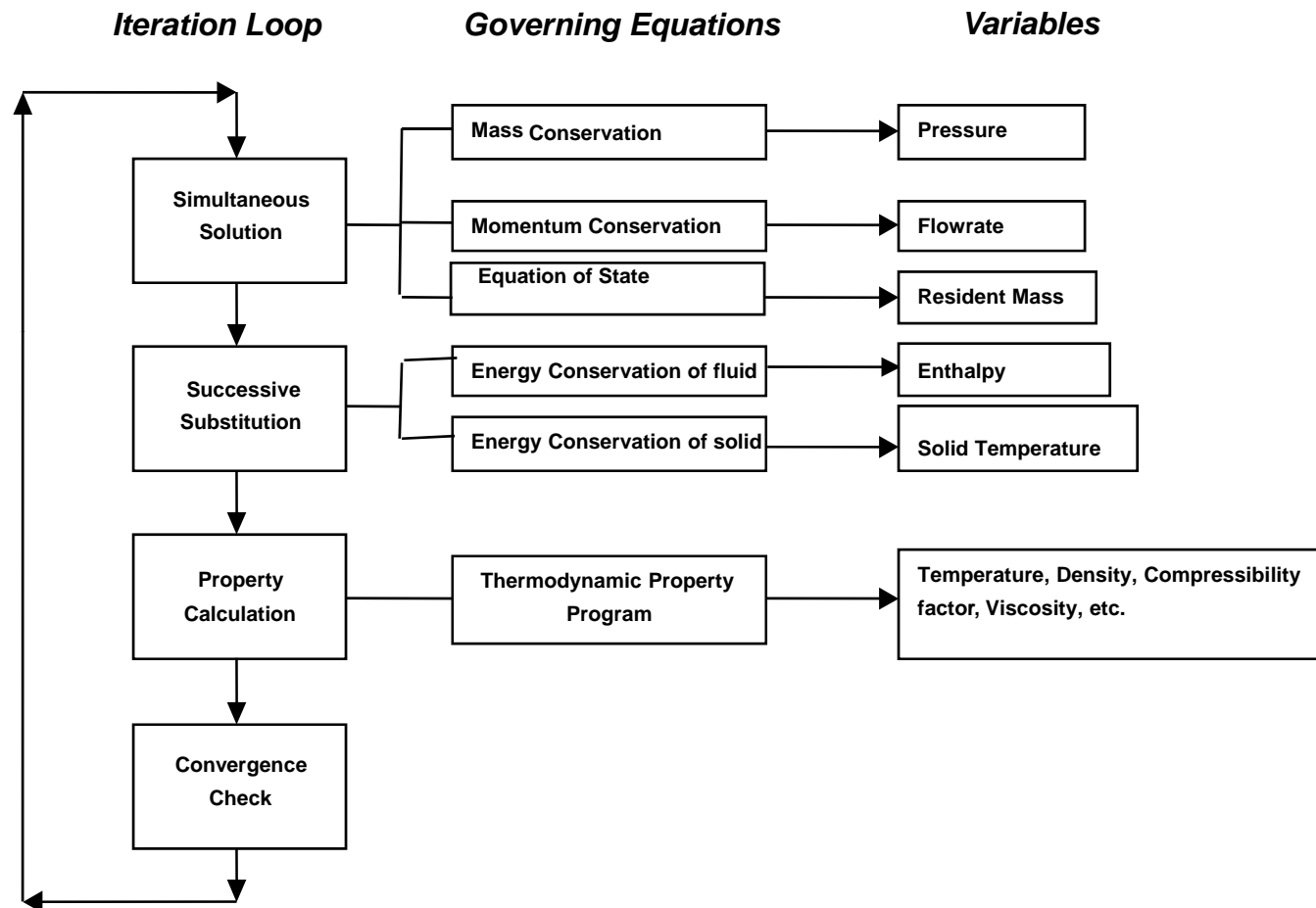
$$\dot{q}_{sa} = h_{ij_a} A_{ij_a} (T_a^{j_a} - T_s^i)$$

$$T_s^i = \frac{\sum_{j_s=1}^{n_{ss}} C_{ij_s} T_s^{j_s} + \sum_{j_f=1}^{n_{sf}} C_{ij_f} T_f^{j_f} + \sum_{j_a=1}^{n_{sa}} C_{ij_a} T_a^{j_a} + \frac{(m C_p)_m}{\Delta \tau} T_{s,m}^i + \dot{S}}{\frac{m C_p}{\Delta \tau} + \sum_{j_s=1}^{n_{ss}} C_{ij_s} + \sum_{j_f=1}^{n_{sf}} C_{ij_f} + \sum_{j_a=1}^{n_{sa}} C_{ij_a}}$$



# SASS Solution Scheme

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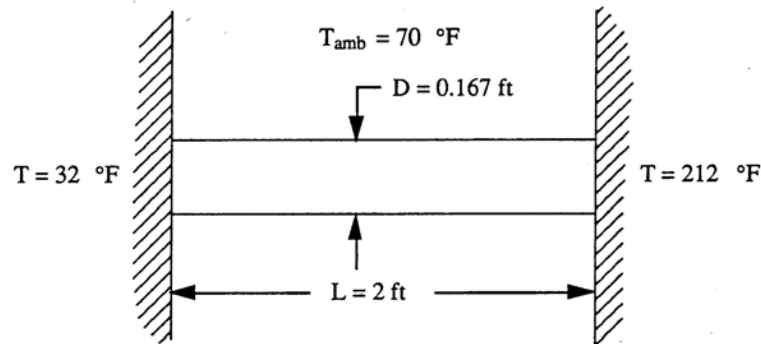




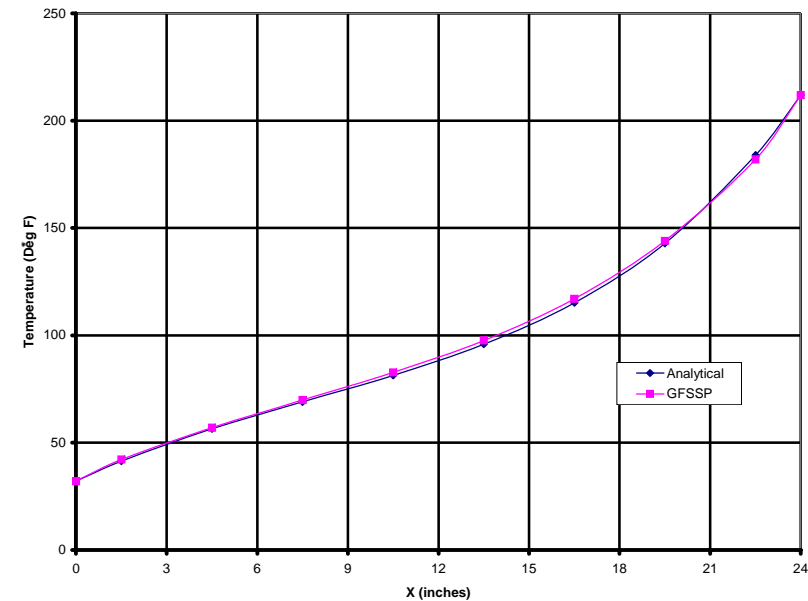


# Verification of Conjugate Heat Transfer Results

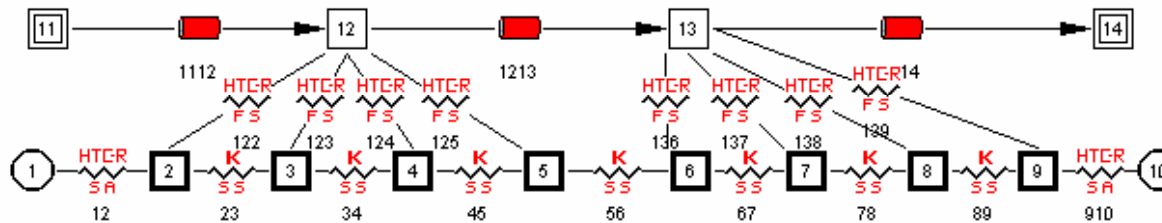
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Problem Considered



Comparison with Analytical Solution

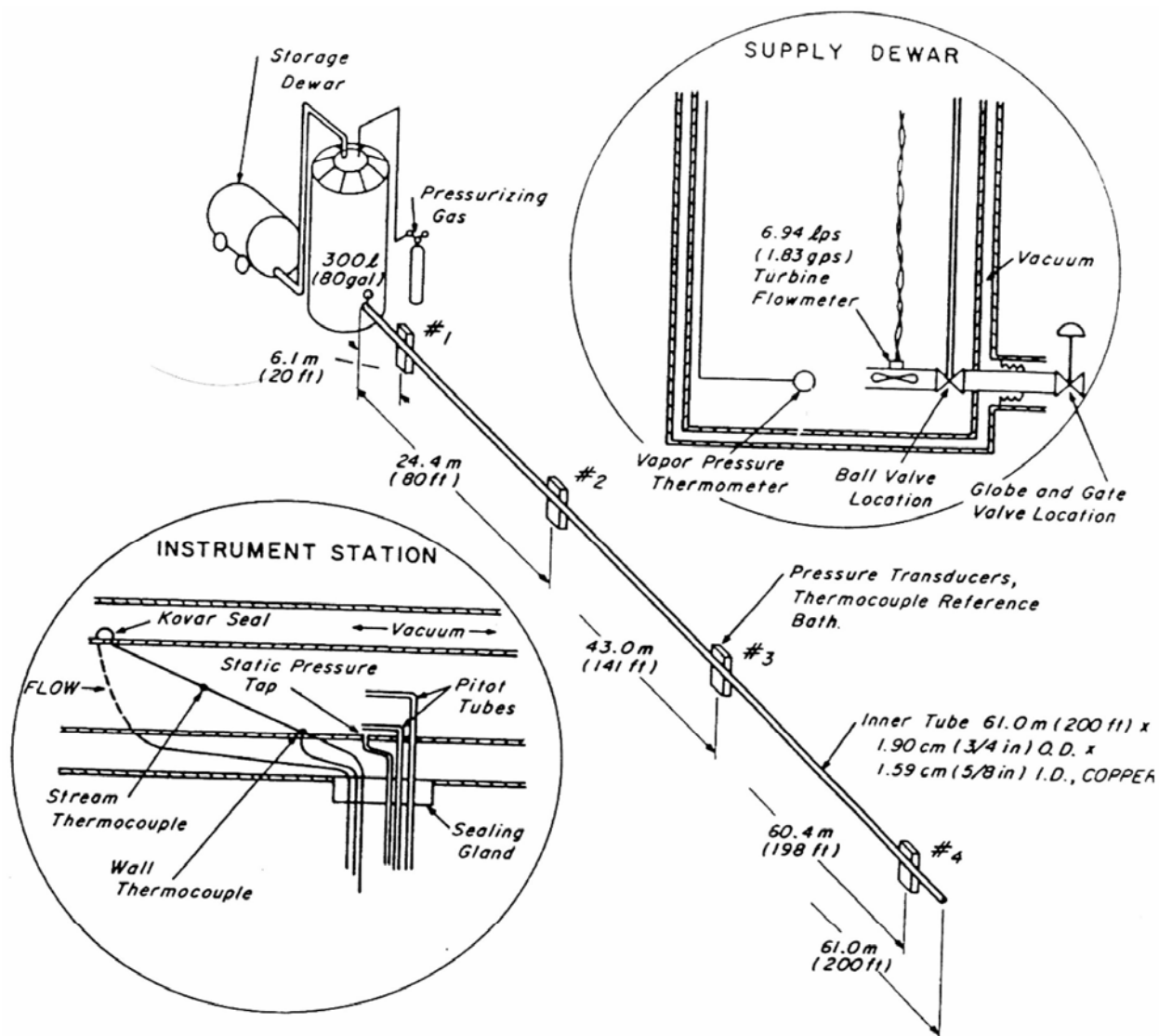


GFSSP Model



# NBS Test Set-up of Cryogenic Transfer Line

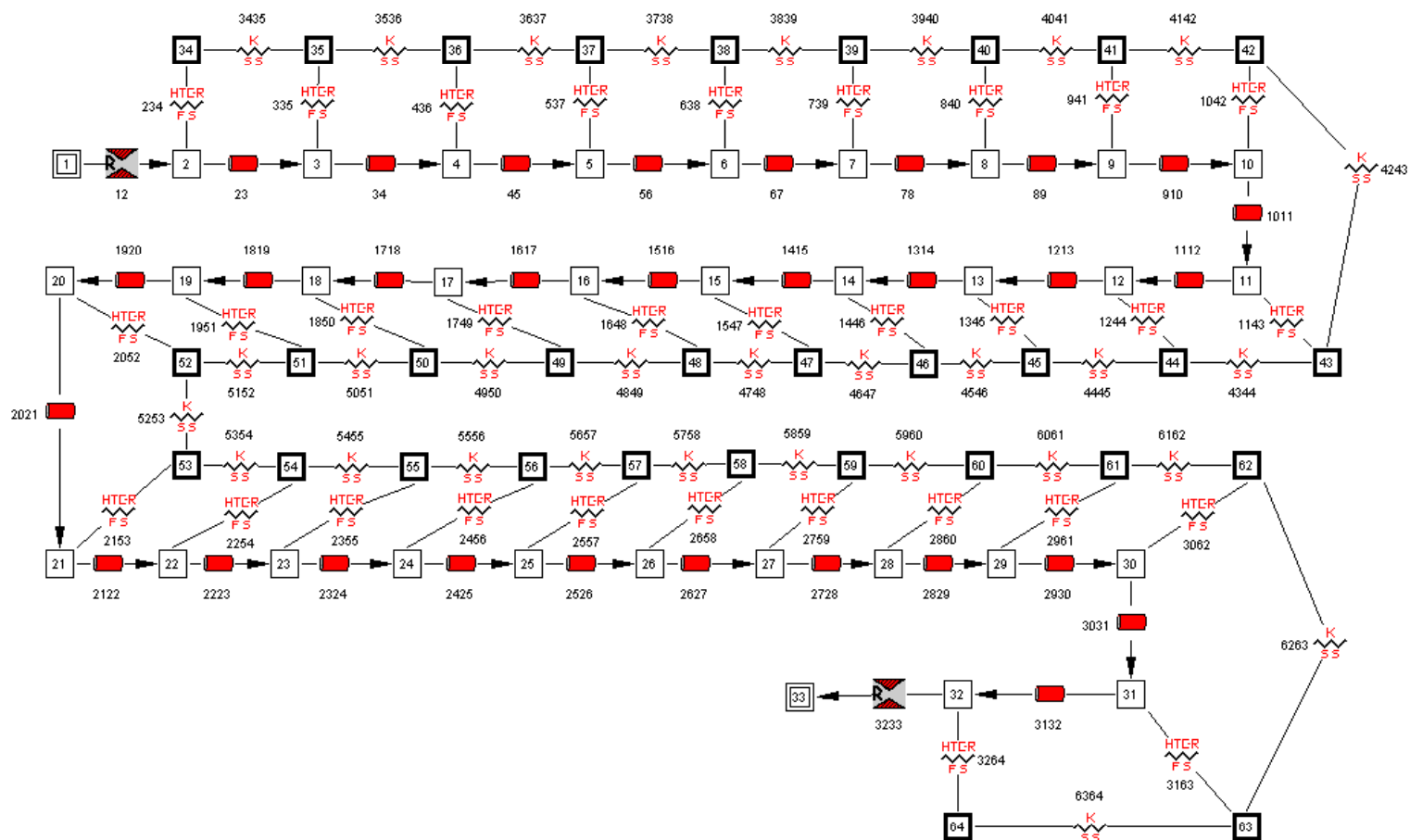
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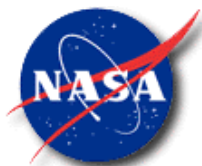




# GFSSP Model of Cryogenic Transfer Line

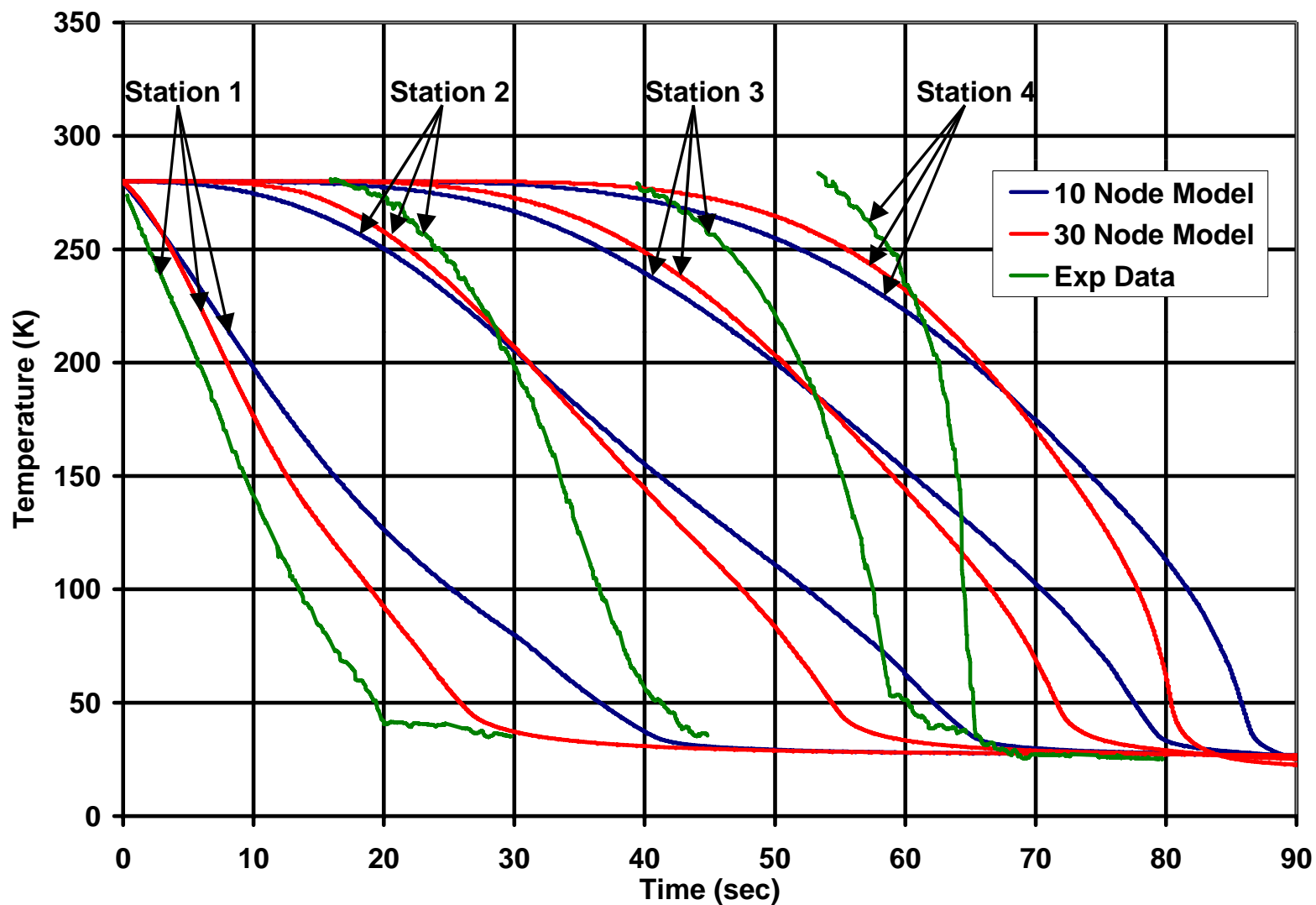
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# Comparison with Test Data

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# Summary

- **GFSSP has been extended to model conjugate heat transfer**
- **Fluid Solid Network Elements include:**
  - Fluid nodes and Flow Branches
  - Solid Nodes and Ambient Nodes
  - Conductors connecting Fluid-Solid, Solid-Solid and Solid-Ambient Nodes
- **Heat Conduction Equations are solved simultaneously with Fluid Conservation Equations for Mass, Momentum, Energy and Equation of State**
- **The extended code was verified by comparing with analytical solution for simple conduction-convection problem**
- **The code was applied to model**
  - Pressurization of Cryogenic Tank
  - Freezing and Thawing of Metal
  - Chillover of Cryogenic Transfer Line
  - Boil-off from Cryogenic Tank